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STABILITY CHARACTERIZATION OF REFRACTORY MATERIALS UNDER HIGH VELOCITY ATMOSPHERIC FLIGHT CONDITIONS

PART IV - VOLUME I: THEORETICAL CORRELATION OF MATERIAL PERFORMANCE WITH STREAM CONDITIONS

LARRY KAUFMAN, HARVEY NESOR, HAROLD BERNSTEIN
and JUDSON R. BARON
ManLabe, Inc.

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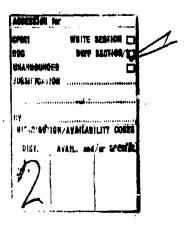
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FOREWORD

This report was prepared by ManLabs, Inc. under Project 7312, "Metal Surface Deterioration and Protection, "Task 731201, "Metal Surface Protection," and Project 7350, "Refractory Inorganic Nonmetallic Materials," Task Numbers 735001, "Refractory Inorganic Nonmetallic Materials: Nongraphitic," and 735002, "Refractory Inorganic Nonmetallic Materials: Graphitic," under AF33(615)-3859 and was administered by the Metals and Ceramics Divisions of the Air Force Materials Laboratory, Air Force Systems Command, with J.D. Latva, J. Krochmal, and N.M. Geyer acting as project engineers.

This report covers the period from April 1966 to July 1969.

ManLabs' personnel participating in this study included L. Kaufman, H. Nesor, H. Bernstein, J. R. Baron and G. Stepakoff.

The manuscript of this report was released by the authors in September 1969 for publication. This technical report has been reviewed and is approved.

W. G. Ramke

Chief, Ceramics and Graphite Branch Metals and Ceramics Division Air Force Materials Laboratory

The following reports will be issued under this contract.

Part/Volume

-		
	I-I	Summary of Results
	II-I	Facilities and Techniques Employed for Characterization of Candidate Materials
	11-11.	Facilities and Techniques Employed for Cold Gas/Hot Wall Tests
	II-III	Facilities and Techniques Employed for Hot Gas/Cold Wall Tests
	III-I	Experimental Results of Low Velocity Cold Gas/Hot Wall Tests
	ш-п	Experimental Results of High Velocity Cold Gas/Hot Wall Tests
	III-III	Experimental Results of High Velocity Hot Gas/Cold Wall Tests
	IV-I	Theoretical Correlation of Material Performance with Stream Conditions
	IV-II	Calculation of the General Surface Reaction Problem

ABSTRACT

The oxidation of refractory borides, graphites and JT composites, hypereutectic carbide-graphite composites, refractory metals, coated refractory metals, metal-oxide composites, and iridium coated graphites in air has been studied under high velocity atmospheric flight conditions. Elucidation of the relationship between hot gas/cold wall (HG/CW) and cold gas/hot wall (CG/HW) surface effects in terms of heat and mass transfer rates at high temperatures was a principal goal.

Published arc plasma test data for refractory materials taken in eight different facilities were collected and examined by comparing the observed surface temperature with calculated radiation equilibrium values. Wide variations in the ratio of calculated to observed temperature were encountered. Similar calculations performed for tests conducted in the present program yielded results close to unity especially when melting is encountered. Larger ratios (up to 1.5) were noted for specific materials which produce silicon oxides, implying enhanced resistance to energy absorbtion. Thus, an alternative method of presentation which compares recession rate as a function of heat flux and enthalpy for the candidate materials was developed. This description provides a means for comparing performance for various trajectories by applying a flux/enthalpy-altitude/velocity translation in considering candidate materials. Comparison of the trajectory of the FDL-7MC lifting reentry vehicle (Lift/Drag ratio between 2.5 and 3.0 and a 3" nose radius) eliminates all of the candidate materials except the boride composites. These composites have survived multicycle exposures totaling 20,000 seconds under conditions simulating the most severe portions of the FDL-7MC trajectory.

Calculations of the flux-enthalpy boundaries for recession rates of l mil/sec based on melting of the solid oxide forming materials are found to compare reasonably with observations. The model employed for these calculations is based on providing the latent heat required for melting at a rate of 1 mil/sec.

Calculation of the surface temperature based on stream and material properties is presented to predict internal temperature gradients for comparison with the "in-depth" measurements. Temperature gradients along the axis of a right circular cylinder which is heated from one end in an arc plasma test with front face and side radiation losses are considered. The effects of radius, length, thermophysical properties and an oxide film on the front face are included. Measurement of temperature gradients through oxide films formed during arc plasma exposures indicate substantial gradients (1000°R through 100 mils) can exist. Comparison of the measurements with computed results yield good agreement in view of the simple models employed. Explicit models are presented for computing the rate of graphite recession in air as a function of density, surface temperature, gas velocity, stagnation pressure and sample radius. The results are compared with observations covering a wide range of conditions.

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I. INTRODUCTION AND SUMMARY

A. Introduction

The response of refractory materials to high temperature oxidizing conditions imposed by furnace heating has been observed to differ markedly from the behavior in arc plasma "reentry simulators." The former evaluations are normally performed for long times at fixed temperatures and slow gas flows with well-defined solid/gas-reactant/ product chemistry. The latter on the other hand are usually carried out under high velocity gas flow conditions in which the energy flux rather than the temperature is defined and significant shear forces can be encountered. Consequently, the differences in philosophy, observables, and techniques used in the "material centered" regime and the "environment centered, reentry simulation" area differ so significantly as to render correlation of material responses at high and low speeds difficult if not impossible in many cases. Under these circumstances, expeditious utilization of the vast background of information available in either area for optimum matching of existing material systems with specific missions or prediction and synthesis of advanced material systems to meet requirements of projected missions is sharply curtailed.

In order to progress toward the elimination of this gap, an integrated study of the response of refractory materials to oxidation in air over a wide range of time, gas velocity, temperature and pressure has been designed and implemented. This interdisciplinary study spans the heat flux and boundary layer-shear spectrum of conditions encountered during high-velocity atmospheric flight as well as conditions normally employed in conventional materials centered investigations. In this context, significant efforts have been directed toward elucidating the relationship between hot gas/cold wall HG/CW and cold gas/hot wall CG/HW surface effects in terms of heat and mass transfer rates at high temperatures, so that full utilization of both types of experimental data can be made. The elucidation of various mass transfer reaction regimes have been studies in gaseous and solid oxide formation.

The principal goal of this study is the coupling of the material-centered and environment-centered philosophies in order to gain a better insight into systems behavior under high-speed atmospheric flight conditions. This coupling function has been provided by an interdisciplinary panel composed of scientists representing the component philosophies. The coupling framework consists of an intimate mixture of theoretical and experimental studies specifically designed to overlap temperature/energy and pressure/velocity conditions. This overlap has provided a means for the evaluation of test techniques and the performance of specific materials systems under a wide range of flight conditions. In addition, it provides a base for developing an integrated theory or modus operandi capable of translating reentry systems requirements such as velocity, altitude, configuration, and life time into requisite materials properties as vaporization rates, oxidation kinetics, density, etc., over a wide range of conditions.

The correlation of heat flux, stagnation enthalpy, Mach No., stagnation pressure, and specimen geometry with surface temperature through the utilization of thermodynamic, thermal and radiational properties of the material and environmental systems used in this study was of prime importance in defining the conditions for overlap between materials-centered and environment-centered tests.

Significant practical as well as fundamental progress along the above mentioned lines necessitated evaluation of refractory material systems which exhibit varying gradations of stability above 2700°F. Emphasis was placed on candidates for 3400°F to 6000°F exploitation. Thus, borides, carbides, boride-graphite composites (JTA), JT composites, carbidegraphite composites, pyrolytic and bulk graphite, PT graphite, coated refractory metals/alloys, oxide-metal composites, oxidation-resistant refractory metal alloys, and coated graphites were considered. Similarly, a range of test facilities and techniques including oxygen pickup measurements, cold sample hot gas, and hot sample cold gas devices at low velocities as well as different arc plasma facilities capable of covering the 50-2500 BTU/ft²sec flux range under conditions equivalent to speeds up to Mach 12 at altitudes up to 200,000 ft were employed. Stagnation pressures covered the range between 0.001 and 10 atmospheres. Splash and pipe tests were performed in order to evaluate the effects of aerodynamic shear. Based on the present results, this range of heat flux and stagnation enthalpy produced surface temperatures between 2000°F and 6500°F.

B. Summary

The present report, which is the seventh in a series (1-6)*, deals with theoretical methods for correlating the performance of the candidate refractory materials with stream conditions. In many respects, this correlation constitutes the essence of the entire study. Thus, given stream characteristics such as stagnation enthalpy, stagnation pressure and cold wall heat flux, is it possible to provide a means for predicting the response of a candidate material upon insertion into this stream? The development of such methods would be of twofold value to the current program. Operationally it would provide a means for checking the internal consistency of arc plasma tests. Thus, measurements of arc plasma stream conditions could be employed to compute the surface temperature achieved by candidate materials during exposure for comparison with observed surface temperature. In addition, since the behavior of these materials is strongly dependent on the surface temperature (or temperature regime) the foregoing correlation could provide a means of describing the materials performance in terms of flight characteristics. This translation could be affected by employing the relations between altitude, velocity and body radius on the one hand and enthalpy, pressure and heat flux on the other. In this way, a logical method for comparing the requirements of specific flight trajectories with the capabilities of refractory materials could be developed.

Accordingly, activities aimed at generating such methods were carried out during the course of the program. At the outset, a literature survey of published arc plasma test data for refractory materials was performed. Data taken in eight different facilities were collected and examined

[&]quot;Underscored numbers in parentheses indicate references given at the end of this report.

by comparing the observed surface temperature with values calculated on the basis of radiation equilibrium. Rather wide variations in the ratio of calculated to observed temperature, T(CALC)/T(OBS) were encountered. In most instances this ratio was greater than unity and in some instances (exposures performed in a given facility) the ratio T(CALC)/T(OBS) resulted in values near 2.0. Similar calculations were performed for all of the arc plasma tests conducted in the present program (6). In these cases (nearly 800 in total) the ratio T(CALC)/T(OBS) was reasonably close to unity. In particular the present results indicate that values near unity are observed when melting is encountered. Larger ratios (up to 1.5) were noted for specific materials which produce silicon oxides such as HfB₂+ SiC(A-4), KT-SiC(E-14), and WSi₂/W (G-18) and for Sn-Al/Ta-10W(G-19). low temperatures (i. e., 3000°R-3500°R) these materials exhibit T(CALC)/ T(OBS) ratios near 1.5 providing that melting does not occur. The occurrence of ratios which are larger than unity implies enhanced temperature capability due to resistance to energy absorbtion by the material. Although the origin of this resistance is not clear at present, it is probably due to blocking effects caused by evolution of gaseous oxides. These observations suggest a method of ranking the behavior of the refractory materials which differs from the customary recession vs. temperature curves. Thus, an alternative method of presentation which compares recession rate as a function of heat flux and enthalpy for the candidate materials was developed. This method does not require a knowledge of the spectral or the normal emittance and integrates the blocking effects characteristic of each material.

In the course of the present study, the oxidation of graphites in air has been investigated experimentally over a range of conditions (4-6) between 2500°R and 6500°R, at velocities between 1 ft/sec and Mach 3.2 The succeeding volume of this series presents a complete discussion of the surface reaction problem encountered in the oxidation of graphite. This discussion considers the coupling of mass transport through the boundary layer with reactions at the surface in detail. By contrast, the discussion presented here employs simplified models which provide an explicit means for computing the rate of graphite recession in air as a function of density, surface temperature, gas velocity, stagnation pressure and sample radius. The results are compared with observations covering a range of density between 80 and 115 lbs/ft³, temperatures between 2500° and 6500°R, velocities between 1 ft/sec and Mach 8.0, stagnation pressures between 0.007 and 1.0 atm and nose radii between 0.005 and 0.07 ft. The current description is based on the product of Arrhenius' term and an oxygen partial pressure term. The former consists of a pre-exponential of 0.74 lbs/ft²sec and an activation energy of 10,730 cal/mole. The oxygen partial pressure term has an exponent of 0.333 and is modified by an explicit correction factor which relates the oxygen concentration at the reacting surface to the oxygen concentration at the edge of the boundary layer. This correction factor is specified in terms of Mach No., body radius and pressure.

A method for describing the response of refractory materials to the enthalpy and heat flux characteristics of the stream has been developed. This description provides a means for comparing material performance at the stagnation point for various trajectories. The comparison can be made by translating the flux-enthalpy description into altitude-velocity characteristics based on established relations between stagnation pressure, altitude and velocity. The Fay-Riddell relation is employed to specify heat flux in terms of altitude, velocity and body radius. The material ranking afforded by this description shows that HfB2+SiC(A-4) possesses the widest range of applicability of all the candidate materials investigated in the present study. In addition, the different medes of behavior exhibited by ablators (such as graphite and tungsten) and solid oxide formers are clearly displayed.

The applicability of the flux/enthalpy-altitude/velocity description in considering candidate materials has been illustrated by comparing the trajectory of the FDL-7MC lifting reentry vehicle with the behavior of candidate refractory materials. This vehicle is designed for a Lift/Drag ratio between 2.5 and 3.0. The conditions imposed by this trajectory for the case of a 3" nose radius eliminate all of the candidate materials except the boride composites. These composites have survived multicycle exposures totaling 20,000 seconds under conditions simulating the most severe portions of the FDL-7MC trajectory (6).

Calculations of the flux-enthalpy boundaries for recession rates of 1 mil/sec based on melting of the solid oxide forming materials are found to compare reasonably with observations. The model employed for these calculations is based on providing the latent heat required for melting at a rate of 1 mil/sec.

Measurement of temperature gradients which exist through oxide films formed during arc plasma exposures indicate substantial gradients (1000 R through 100 mils) can exist (6).

A first order calculation of the surface temperature as a function of stream conditions and material properties is presented in order to provide a means for predicting internal temperature gradients for comparison with the "in depth" measurements. This calculation considers temperature gradients along the axis of a right circular cylinder which is heated from one end in an arc plasma test. Front face and side radiation losses are considered in describing the effects of radius, length and thermophysical properties on the surface temperature and internal gradients. In addition, the effect of an oxide film on the front face was included. The calculations indicate that small gradients occur when the surface temperature is low, or when the cylinder length and/or the length/radius ratio is small. Large values of the thermal conductivity of the cylinder material also leads to small gradients. Large values of surface temperature, length/radius ratio and small values of the thermal conductivity of the oxide and base material result in large gradients.

The model has been applied to calculation of temperature gradients for comparison with the experimental results obtained in sixty-five arc plasma tests on a variety of refractory materials. Ablutors and oxide forming materials covering a wide range of thermophysical and oxidation characteristics, such as $ZrB_2+SiC(A-8)$, $ZrB_2(A-3)$, $HfB_2+SiC(A-7)$, RVA (B-5), $ZrB_2+SiC+C(A-10)$, $WSi_2/W(G-18)$ and Hf-Ta-Mo(I-23) were included.

Observables consisted of the measured front face temperature T_f , the observed temperature at a distance, d mils from the front face, T(d), the cold wall heat flux, q, the stagnation enthalpy, i_e , and the stagnation pressure P_e . Additional input consisted of sample radius, R, length, L, and oxide coating thickness, I. The latter was equated to the conversion depth for the oxide formers (6). For WSi_2/W , I was equated to the WSi_2 coating thickness with I=0 for RVA(B-5) graphite which ablates without coating formation. Suitable values of the emittance, ϵ_S , and the thermal conductivities of the coating, k_F , and the substrate, k_S , were also employed.

The computed results are displayed in terms of the ratio of calculated front face temperature to observed front face temperature $T_f(\text{CALC})/T_f(\text{OBS})$ and the ratio of computed in-depth temperature $T_d(\text{CALC})$ to computed front face temperature $T_f(\text{CALC})$. The latter is compared with the ratio of observed in-depth temperature $T_d(\text{OBS})$ to observed front face temperature $T_f(\text{OBS})$. If agreement between calculated and observed temperatures results, $T_f(\text{CALC})/T_f(\text{OBS})$ would equal unity and the ratios $T_d(\text{CALC})/T_f(\text{CALC})$ and $T_d(\text{OBS})/T_f(\text{OBS})$ would coincide. Relatively good agreement was encountered as regards the latter comparison in view of the simple model employed to describe the complex tests. In addition, $T_f(\text{CALC})/T_f(\text{OBS})$ ratios were computed near unity for many of these tests. However, in line with the behavior noted above some systematic deviations were observed.

The largest of these occurred at low surface temperatures (i.e., $T_f \le 3300^{\circ}R$) for the materials which form SiO₂ as an exidation product. Thus, in cases where samples of HfB₂+SiC(A-7), ZrB₂+SiC(A-8), ZrB₂+SiC+C(A-10) or WSi₂/W(G-18) were exposed with shrouds or as large diameter hemispheres, T_f (CALC) is considerably larger than T_f (OBS). However, this difference is smaller than obtained when T_f is computed on the basis of front face radiation equilibrium alone. The cause of this behavior is presently unknown.

II. THE THERMAL RESPONSE OF REFRACTORY MATERIALS TO HOT GAS/COLD WALL EXPOSURES

A. Introduction

Prior to launching the extensive HG/CW testing program conducted under the present investigation, published data on the surface temperature of borides, graphites, graphite composites, silicon carbide, boron nitride, tungsten alloys and composites, refractory metal-oxide composites, coated refractory metals and iridium coated graphite exposed in arc plasma and Wave Superheater tests were collected. These data covered exposure conditions over a range of stagnation pressures between 0.002 and 70 atmospheres while Mach Numbers, stagnation enthalpy and heat flux levels ran from 0.2 to 8.5, 1400 to 18,000 BTU/lb and 20 to 4080 BTU/ft²sec, respectively. The specimen configurations tested included flat face cylinders and hemispherical caps with diameters between 0.25 and 3.00 inches. The facilities at which these exposures were performed included Avco/SSD, Cornell Aeronautical Laboratory, General Electric Space Science Center, Plasmadyne Corporation, Cinncinati Testing Laboratory, North American Aviation Center, Grumman Aircraft Engineering Corporation and Aerospace Corporation. The results were compared with computed surface temperatures based on radiation equilibrium in order to estimate the current level of reliability of surface temperature predictions from stream conditions. A similar comparison of the results obtained in the present program (6) has been provided and is compared with the earlier results.

B. Compilation of Experimental Surface Temperatures and Stream Conditions

Since the present study is concerned with a comparison of CG/HW and HG/CW exidation, correlation of the surface temperature of models (specimens) exposed under HG/CW conditions with specific stream characteristics is of paramount importance. In order to gain some insight into the relationship between the surface temperature and stagnation enthalpy, heat flux and pressure under subsonic and supersonic flow conditions, a review of available literature has been made. The results are contained in Tables 1-16 which identifies the material, Mach Number, stagnation pressure (Pe), enthalpy (ie), cold wall heat flux (qcw) and model configuration (D). As indicated, the diameter of cylindrical samples are designated by asterisks. In addition, the observed surface temperature, measured optically, is noted. Since the assignment of "observed temperature" is not performed uniformly in Tables 1-16, it is worthwhile to report the methods employed for each set of measurements. No attempt will be made at present to correct these "observed temperatures".

The ZrB₂, HfB₂ and HfB₂-SiC exposures reported under Reference (7) in Tables 1 and 2 were performed at Avco/SSD. Brightness temperatures were measured at $\lambda = 0.65$ microns and converted

to "observed temperature" by using an emittance value of 0.60. Exposure times of five to thirty minutes were employed. The high pressure exposures of ZrB_2 , Boride Z, ZrO_2 , etc., shown under Reference (8) in Tables 2-8 were performed in the Cornell Wave Superheater. The former runs, as well as those in Tables 4-6 and 8 designated by Reference (11) (which were also carried out in the Wave Superheater), were given T5 second exposure times. In these cases, the brightness temperature at $\lambda = 2.1 \pm 0.5$ microns is reported. These temperatures represent mean values observed during exposure.

The graphite and singsten exposures in Tables 7, 9, 12 and 13 denoted by Reference (10) were performed at Avco/SSD. These tests were of 60 to 120 seconds duration. Total emittance values were measured with an Eppley thermopile. In addition, brightness temperatures were measured at $\lambda = 0.65$ microns and converted to surface temperature by assuming that the emittance at $\lambda = 0.65$ microns is equal to the total emittance.

The results reported in Table 2 for tantalum (9) were obtained during 50-150 second exposures using an optical pyrometer at $\lambda=2.1\pm0.4$ microns. Brightness temperatures were converted to observed temperatures by using an emittance of 0.45. The latter value was obtained from the observation of the melting point of Ta₂O₅ at 3730 R. The HfC-C (20) exposure time was 6 seconds. The temperature level of this CAL Wave Superheater test was established by observation of incipient melting of HfO₂ (Table 4).

The ATJ Graphite exposures denoted by Reference (12) were performed in the General Electric tandem Gerdien and free jet facilities (Table 10). The RVA graphite and graphite composites designated by Reference (13), Tables 11 and 12 were similarly tested. Exposure time was 60 to 1000 seconds and surface temperatures were measured with a two color pyrometer.

Exposure times of 100-1200 seconds were employed testing the JTA and ZrO₂ materials shown in Tables 13 and 14 designated by Reference (16). The reported surface temperatures are optical brightness temperature measured at $\lambda = 0.65\mu$. The same situation holds for the subsonic, one atmosphere exposures, shown in Tables 15 and 16 denoted by Reference (18) except that 30-60 second exposures were performed for the latter cases. Finally, the iridium coated graphite and JTA tests in Table 14 designated by Reference (17) were exposed for 400-1200 seconds. Temperatures were measured with a two color pyrometer.

C. Correlation of Results

Tables 1-16 provide a valuable empirical guide to estimating surface temperatures from specific stream conditions. However, it is desirable to provide a means for comparing results contained in Tables 1-6 which are related (i.e., cases where a given material is exposed to two slightly different stream conditions). In other terms, it would be

useful to interpolate and extrapolate these in order to compare results obtained in different facilities and to predict the anticipated temperature for any given exposure.

The simplest means of performing such a correlation is to describe the conditions for radiation equilibrium at the stagnation point of the model on the assumption that the energy lost by radiation is equal to the heat transferred to the model. On this basis,

$$\sigma \in T^4 = h_e(i_e - i_w [T, P_e]) BTU/ft^2 sec$$
 (1)

where $0 = 0.47 \times 10^{-12}$ BTU/ft²sec^oR⁴, ϵ is the total hemispherical emittance, h_e is the stagnation point heat transfer coefficient, i_e [T,P_e] is the stagnation enthalpy, and i_w is the enthalpy of air at the wall (surface) of the model and T^oR is the wall temperature. Eq. 1 usually ignores reactions at the model surface which give rise to significant evolution or absorption of energy and is as a result but a crude approximation to estimating the surface temperature which depends on a knowledge of h_e . If the cold wall heat flux, q_{cw} , is defined as

$$q_{cw} = h_e i_e BTU/ft^2 sec$$
 (2)

then Eq. 1 becomes

$$\sigma \in T^{4} = q_{cw}(1 - i_{w}[T, P_{e}]/i_{e})$$
(3)

Eq. 3 describes the wall temperature, $T^{O}R$ as an explicit function of the stream parameters, i_{e} , q_{CW} , and the stagnation pressure P_{e} , as well as the enthalpy of air at the wall i_{W} [T, P_{e}] and the total hemispherical emittance of the surface ϵ .

Numerical values of the enthalpy of air at the wall are given in Table 17. Specification of ϵ , q_{CW} , i_e and P_e fixes the surface temperature. Eq. 3 can be solved numerically by using the values of $i_w[T, P_e]$ shown in Table 17. However, it is convenient to represent $i_w[T, P_e]$ by analytic functions in order to obtain algebraic solutions. The following equations have been employed for this purpose:

$$i_w[T, P_e] = (T/1000) (100+45 \log P_e + (T/1000) (46-14 \log P_e)) BTU/b$$
 (4)

and

$$i_w[T, P_e] \approx 33.9 (T/1000)^2 (2.0 - 0.13 log P) BTU/lb (5)$$

The numerical values of $i_w[T, P_e]$ described by Eq. 4 and Eq. 5 are compared with the established values of the enthalpy of air (21), (10) in Table 17. In the temperature and pressure range of interest (i.e., $2700^{\circ} < T < 7200^{\circ} R$ and $+2 > \log P_e > -2$), Eq. 4 represents a reasonably good representation of $i_w[T, P_e]$. Thus at pressures which are equal to or greater than 0.1 atm the difference between $i_w[T, P_e]$ and Eq. 4 are less than 200 BTU/lb. At 0.01 atm, larger differences are noted at 5400° and $6300^{\circ} R$. However, reference to Tables 1-16 shows that under low pressure testing conditions i_e is generally in the 5000-18,000 BTU/lb range. Under these circumstances, an error of 400 BTU/lb in $i_w[T, P_e]$ is not serious.

Eq. 5 is a poorer representation of $i_w[T, P_e]$. However, its simple quadratic form permits direct solution of Eq. 3 as follows:

$$0.47\epsilon \left(\frac{T}{1000}\right)^{4} = q_{cw} \left(1-33.9 \left(\frac{T}{1000}\right)^{2} \left(2.0-0.13 \log P_{e}\right)/i_{e}\right)$$
(6)

hence

$$(T/1000)^4 + 33.9q_{cw}(2.0-0.13 \log P_e) (0.47\epsilon i_e)^{-1} (T/1000)^2 - q_{cw}/0.47\epsilon = 0$$
 (7)

or

$$(T/1000)^2 = -0.5b + 0.5 (b^2 - 4c)^{1/2}$$
 (8)

where

$$b \approx 33.9q_{cw} (2.0-0.13 \log P_e) (0.47\epsilon i_e)^{-1}$$
 (9)

and

$$c = -q_{cw}/0.47\epsilon \tag{10}$$

Eqs 6-10 can be employed to obtain a crude estimate of the surface temperature at pressure equal to or less than 1 atm. At higher pressures, temperatures computed to be greater than 5400°R will be too low.

Eq. 3 has been employed to compute i_e for fixed values of q_{cw} and T at various stagnation pressures and emittance levels. These calculations were performed by using the established values of $i_w[T,P_e]$ given by Reference (10). The results are contained in Figures 1 to 15 and constitute exact solutions to Eq. 3.

Reference to Figures 1 to 15 illustrate the expected effect of the total emittance on surface temperature. Thus, under stream conditions corresponding to a cold wall heat flux of 200 BTU/ft-sec, a stagnation pressure of one atmosphere and a stagnation enthalpy of 10,000 BTU/lb; a material having a total emittance of 0.2 would reach a surface temperature of 6200°R (Figure 6). Under the same conditions, a material having an emittance of 1.0 would reach a surface temperature of only 4400 OR (Figure 10). These curves also show that with increasing stagnation enthalpy, the surface temperature is dependent only on heat flux and emittance as implied by Eq. 3 for (i_w/i_e) zero. However, when the heat transfer coefficient is large, the temperature depends upon the stagnation enthalpy. In other terms, even under high flux conditions, the surface temperature cannot exceed the gas temperature. Note that in all of the proceeding discussions, heats of chemical reaction at the surface are ignored. Naturally, such effects could result in generation of surface temperatures which are in excess of stream temperatures. The inflections present in the ie [T] curves at log Pe = -2.0 (Figures 11-15) result from the temperature dependence of the enthalpy as indicated in Table 17.

Eq. 3 has been employed to compute the surface temperature for each of the exposures shown in Tables 1-16 by employing the representation of $i_w[T, P]$ given by Eq. 4. These calculations are performed on a computer using a Newton-Raphson technique to obtain solutions. The resultant ratio of computed temperature to observed temperature is contained in Tables 1-16. The ratio, T(CALC)/T(OBS), is plotted as a function of cold wall heat flux, q_{cw} , stagnation enthalpy, i_c , stagnation pressure, P_c , Mach Number and observed temperature in Figures 16-20. These calculations have been performed using a total hemispherical emittance of 0.6 for all exposures in Tables 1-16 except those in which emittance values were measured. In the latter cases, the reported values were employed.

An alternate calculation of the "Radiation Equilibrium" surface temperature can be performed which employs the Fay-Riddell relation (22) to compute the heat transfer coefficient, h_C. This procedure is performed by setting

$$h_{e} = 0.94 \left(\frac{2R_{B}}{u_{\infty}} \frac{du_{e}}{dx} \right)^{1/2} \left(\frac{(\rho\mu)_{e}u_{\infty}}{2R_{B}} \right)^{1/2} \left(\frac{(\rho\mu)_{w}}{(\rho\mu)_{e}} \right)^{1/10}$$

$$lbs/ft^{2}sec \qquad (11)$$

Letting

$$\left(\begin{array}{cc} \frac{2R_{B}}{u_{m}} & \frac{du_{e}}{dx} \end{array}\right) = Q \tag{12}$$

for convenience and noting that

$$Q^{2} = \frac{8 ((\gamma - 1) M^{2} + 2)}{(\gamma + 1) M^{2}} \left[1 + \frac{(\gamma - 1)}{2} \frac{((\gamma - 1) M^{2} + 2))}{(2\gamma M^{2} - (\gamma - 1))}\right]$$
(13)

for Mach Numbers greater than one, and

$$Q^2 = 9 \tag{14}$$

for Mach Number = 0, where $\gamma = C_D/C_v = 1.4$ for an ideal gas, and M is the Mach Number. Approximating

$$((\rho\mu)_{w}/(\rho\mu)_{e})^{1/10} = 1$$
 (15)

yields

$$h_e = 0.94Q^{1/2} ((\rho \mu)_e u_{\infty} / 2RB)^{1/2}$$
 (16)

Eq. 16 can be evaluated by estimating the viscosity by (23, 24) the following:

$$\mu = 2.17 \times 10^{-8} T^{1/2}$$
 lbs (force) sec ft² (17)

Setting P = ρ RT and R = 1724 ft²/sec² o_R and noting that

$$(T_e'T_\infty) = 1 + 0.5 (\gamma - 1) M^2$$
 (18)

with $M^2 = u_{\infty}^2 / \gamma RT_{\infty}$ yields

$$h_{e} = 0.94Q^{1/2} [1.09 \times 10^{-8} \frac{(lbs, force) sec)}{ft^{2} o_{R}^{1/2}} (\gamma/R)^{1/2} M(1+0.5(\gamma-1)M^{2})^{-1/2}]^{1/2} (P_{e}/R_{B})^{1/2}$$
(19)

Also for high Mach Numbers (M >> 1), Q = 1.11 and

$$h_e = 0.99 (6.90 \times 10^{-10})^{1/2} (P_e/R_B)^{1/2} lbs (force) sec/ft^3$$
 (20)

where Pe is in lbs (force)/ft2 and RB is in feet. Thus

$$h_e = 2.60 \times 10^{-5} (P_e/R_B)^{1/2} \text{ lbs (force) sec/ft}^3$$
 (21)

(lbs (force) = 32.2 lbs ft/sec^2). Hence

$$h_e = 8.37 \times 10^{-4} (P_e/R_B)^{1/2} lbs/ft^2 sec$$
 (22)

When the stagnation pressure is in atmospheres. (1 atm = 2117 lbs (force)/ft²)

$$h_e = 0.0386 (P_e/R_B)^{1/2} lbs/ft^2 sec$$
 (23)

Eq. 23 is a reduced Fay-Riddell relation applicable at high Mach Numbers. The general relation given above in Eq. 19 is

$$h_e = 0.0245Q^{1/2} [M^2(1+0.5(\gamma-1)M^2)^{-1}]^{1/4} (P_e/R_B)^{1/2}$$

$$lbs/ft^2sec \qquad (24)$$

where P is in atmosphere and R_B in feet. If the quantity $0.0245Q^{1/2}[1+0.5(\gamma-1)M^2)^{-1}]^{1/4}$ is approximated by $0.0386/(1+0.17M^{-1})$, which is accurate to within 5% for M>0.1, then Eq. 24 becomes

$$h_e = 0.0386 (1 + 0.17 M^{-1})^{-1} (P_e/R_B)^{1/2} lbs/ft^2 sec$$
 (25)

Substitution into Eq. 1 yields

$$\sigma_{\epsilon} T^{4} = 0.0386 (1 + 0.17 M^{-1})^{-1} (24 P_{e}/D)^{1/2} (i_{e} - i_{w}[T, P_{e}])$$
 (26)

where D is the diameter of the hemispherical cap in inches.

Eq. 26 has also been applied to compute the radiation equilibrium temperature for all of the exposures in Tables 1-16 by employing the description of $i_w[T, P_e]$ afforded by Eq. 4. These calculations were performed for flat faced cylinders by setting

$$D = 2.5 Diameter of Cylinder$$
 (27)

in Eq. 26 in order to account for the difference in heat transfer between hemispherical caps and flat faced cylinders. The stagnation point heat flux to a flat faced cylinder having a diameter D_c is equivalent to the heat transfer to a hemisphere having an effective diameter, $D_{\rm eff}$, equal to fD_c . The values of f noted in the literature are 2.1 (25), 2.5 (12) and 2.9 (12). In addition, the following numerical values of f have been reported: 2.0 (26), 3.2 (26), 3.3 (27) and 3.08, 3.34 and 3.72 at Mach Number 2.0, 3.0 and 4.76, respectively (28). Thus it appears that values of 2.0 $\leq f \leq 3.72$ have been employed for relating flat faced cylinders to hemispheres. As indicated above, a value of 2.5 is currently being employed in the present calculations (Eq. 27). This variation may result from individual facility characteristics and measurement techniques. For example, the value f = 2.0 (26) has a heat transfer basis while f = 3.2 (26) has a pressure gradient basis. All of the remaining values except 2.1 (25) were obtained experimentally.

The results of applying Eq. 26 contained in Tables 1-16 under the heading "Fay-Riddell" are displayed in Figure 21 which shows the ratio T(CALC)/T(OBS) vs. the observed cold wall heat flux.

D. Discussion of Results

Examination of Tables 1-16 and Figures 16-21 shows the wide divergence between observed and calculated temperatures. Although it is presently impossible to define the causes of these discrepancies, some of the possibilities are worthwhile noting. To begin with, the exposure times for the tests under consideration are variable. Thus the Avco-MeB₂ and HfB₂ + SiC; the Plasmadyne-ZrO₂ and JTA; the General Electric ATJ, RVA and JTA; and the Cincinnati Testing Laboratory JTA and Ir coated graphite exposures were of 100-1800 seconds duration. The remaining tests were for 30-60 seconds with the exception of the CAL exposures which were limited to 15 seconds. Under these conditions, it might be expected that radiation equilibrium is more readily attained in the long time exposures (i.e., times greater than 100 seconds). Thus, the observed temperature in a short time exposure would be lower than the calculated radiation equilibrium temperature.

A second major source of error is the measurement of surface temperature. This depends upon the particular value of spectral emissivity employed in correcting the observed brightness ($\epsilon = 1$) to the true temperature. For the G.E. and CTL results, obtained with a two color pyrometer, the error in assigning a true surface temperature depends upon how well grey body conditions are approximated. Although these temperature measurement errors can be significant, it is not likely that they are the prime source of the present discrepancies in the T(CALC/OBS) ratios. A related source of error is the value of total emittance used in computing the temperature in Eq. 3. As indicated earlier, measured values have been employed where available. Where no values are available, a mean value of 0.6 has been employed. If the true value of total emittance is 1.0, then the calculated temperature will be too high by the one fourth root of (1/0.6) or fourteen percent. Such corrections could improve the present results. However, such changes would not eliminate the current level of disagreement between observed and calculated temperature.

The relative sizes of the model and the arc are an additional variable which has not been considered in the analysis. Table 13 contains the results obtained during the exposure of silicide coated refractory metal alley feil. In these experiments, the arc diameter was about one inch and the model was a two inch square foil. The foil radiated from both the front and back face. As a result, the ratio of T(CALC)/T(OBS) is much larger than unity. These ratios are not included in Figures 16-21.

Comparison of the T(CALC)/T(OBS) values shown in Tables 1-16 for the "Cold Wall" and "Fay-Riddell" Heat Transfer Coefficient computations indicate that in some cases substantial differences exist between these heat transfer coefficients. Thus, the "Cold Wall" and "Fay-Riddell" heat transfer coefficients differ markedly in the data generated at Avco and CTL. There is some evidence that this may be due to turbulent test streams (29). On the other hand, the CAL, General Electric, Aerospace Corp., Plasmadyne, North American and Grumman data show good correlation between the "Cold Wall" and "Fay-Riddell" values. Reference to Figure 21 shows that the ratio of T(CALC)/T(OBS) is poorer for the "Fay-Riddell" calculation than for the "Cold Wall" calculation.

As previously indicated, the present calculations ignore heat liberated at the surface due to oxide formation or heat absorbed by the surface in order to vaporize or melt the model material. Inclusion of the former effects could raise the present values of T(CALC)/T(OBS) while inclusion of the latter would lower the current T(CALC)/T(OBS) ratios. Although these effects can be substantial, they do not appear to be the source of the current differences. Inclusion of the heats of combustion would raise T(CALC) and further aggravate the T(CALC)/T(OBS) ratios. Moreover, reference to Figure 20 shows that the largest deviations from T(CALC)/T(OBS) = 1, occur below 5000°R where copious vaporization of the materials under consideration is unlikely.

The most important (and unfortunately the least tractable) sources of error are the reported values of the stagnation enthalpy and cold wall heat flux. Nonuniform flux and enthalpy conditions in addition to gas radiation losses and variation in the techniques employed for the measurement of these quantities is probably the most important single source of the deviation of T(CALC)/T(OBS) from unity in Figures 16-21. Thus, comparison of theAvco (half-filled squares) and General Electric (rimmed squares) results for ATJ graphite shows an extremely large discrepancy. The Avco data yield values of T(CALC)/T(OBS) which fall between 1.10 and 1.18 based on observed "Cold Wall" heat transfer coefficients while the General Electric results for the same material range from 1.13 to 2.05. It should be pointed out that the latter results were obtained for long time exposures.

Apart from the General Electric data, values of T(CALC)/
(OBS) which differ from 1.0 + 0.2 are frequent in the Aerospace and CAL
exposures. As indicated earlier, these exposures were for short times
and the high values may be due to the fact that radiation equilibrium was
not established.

E. Results Obtained during the Present Testing Program

In the course of the present program, more than 700 are plasma tests were performed (6). Measurements of stagnation enthalpy, stagnation pressure, cold wall heat flux and brightness temperature were performed in each case. The latter were converted to true temperature by using the measured values of the emittance for the candidate materials (5) under oxidizing conditions. In addition, radiated heat flux was also measured in these tests so that normal emittance, ϵ_N , could be deduced (6). Table 18 summarizes the average values of ϵ_N and the T(CALC)/T(OBS) ratios for each of the materials tested (6). The latter ratio is computed on the basis of Eqs. 3 and 4 corresponding to radiation equilibria.

Ideally, if radiation equilibria were the dominant factors and all measurements were accurate, these ratios should be unity. Although there are departures, it is satisfying to note that the differences are small compared to those obtained by considering the results of other studies (i.e., Figures 16-21 and Tables 1-16). Reference to Table 18 shows that ratios of T(CALC)/T(OBS) are lower for cases where melting is observed than for cases where a solid oxide (or coating) is present. Moreover, Table 18 shows that large values of T(CALC)/T(OBS) are characteristic for some of the materials. The occurrence of ratios which are larger than unity implies resistance to energy absorption by the material. Thus, exposure of HfB2 + SiC(A-4) and HfC + C(C-11) to identical stream conditions (i.e., stagnation pressure, enthalpy and cold wall heat flux) would result in an 11% lower surface temperature than that reached by HfC + C(C-11). This conclusion would apply if stream conditions were not sufficient to produce melting of HfB2 + SiC(A-4). At lower levels, KT-SiC(E-14), WSi2/W(G-18) and Sn-Al/Ta-10W(G-19), which exhibit T(CALC)/T(OBS) ratios of 1.43, 1.56 and 1.41, respectively demonstrate similar resistance to energy transfer. Although the origin of this resistance is not clear at present, it is probably due to blocking effects caused by evolution of gaseous oxides. These observations suggest a method of ranking the behavior of the refractory materials which differs from the customary recession vs. temperature curves. In Section IV, an alternative method of presentation which compares recession rate as a function of heat flux and enthalpy for the candidate materials is considered. This method does not require a knowledge of the spectral or the normal emittance and integrates the blocking effects characteristic of each material.

III. SIMPLIFIED MODEL FOR CALCULATING THE OXIDATION BEHAVIOR OF GRAPHITE UNDER HIGH VELOCITY AIR FLOW CONDITIONS

A. Introduction and Summary

In the course of the present study, the oxidation of graphites in air has been investigated experimentally over a range of conditions (4-6) between 2500°R and 6500°R, at velocities between 1 ft/sec and Mach 3.2. Additional studies (12), (13) have been performed. The succeeding volume of this series (30) presents a complete discussion of the surface reaction problem encountered in the oxidation of graphite. This discussion considers the coupling of mass transport through the boundary layer with reactions at the surface in detail (30). By contrast, the discussion presented below employs simplified models (7, 31-33) which provide an explicit means for computing the rate of graphite recession as a function of density, surface temperature, gas velocity, stagnation pressure and sample radius. The results are compared with observations covering a range of density between 80 and 115 lbs/ft³, temperatures between 2500° and 6500°R, velocities between 1 ft/sec and Mach 8.0, stagnation pressures between 0.007 and 1.0 atm and nose radii between 0.005 and 0.07 ft.

B. Derivation of the Simplified Model for Graphite Oxidation

Comparison of the recession rates observed for graphite in the present study at air flow rates between 1 ft/sec and 250 ft/sec (Figure 37 and pp 23-29 of reference 5) demonstrate that most (if not all) of the graphite oxidation data previously determined (34, 35) is controlled by a supply limit. Thus, for example, Eqs. 28 (Gulbransen et al. (34)) corresponds to a recession rate of

$$m = 1.86 \times 10^{-6} P_{O_2}^{0.32} e^{-3600/RT} gm/cm^2 sec$$
 (28)

where P_{O_2} is in torr and T is in ${}^{\circ}K$, and

$$\dot{S} = 2.44 \times 10^{-2} P_{O_2}^{0.32} e^{-3600/RT}$$
 mils/min

for graphite with a density ρ = 1.80 gms/cm³ or 112 lbs/ft³. For air, P_{O_2} = 160, thus Eq. 28 yields a rate of 0.0007 mils/sec at 3000°R. This compares with values near 0.1 mils/sec at a flow rate of 1 ft/sec and 1 mil/sec at 250 ft/sec observed in the current study (5, Figure 37). The oxidation rates reported by Okada and Ikeqawa (35) between 1800°R and 5400°R under slow flow conditions at a pressure of 0.21 atm O₂ yield similar comparisons. Values corresponding to 0.02, 0.07 and 0.10 mils/sec are

reported (35) at temperatures of 2000 R, 3000 R and 4000 R, respectively. As indicated (5), transport of sufficient quantities of oxygen to the reacting surface and a knowledge of the oxygen partial pressure at the reacting surface are required in order to make any measurement meaningful. Theoretical studies along these lines are presented in reference (30) in order to calculate concentration gradients present in the gaseous boundary layer adjacent to the reaction surface. However, since the details of this treatment are quite complex it is instructive to consider the approximate treatment described earlier (Eq. 15 p. 242, Reference (7)).

The correct picture for the oxidation of graphite above 800°C (1472°F) is one of continually increasing rate with both temperature and oxygen pressure according to the Arrhenius relationship:

$$\dot{\mathbf{m}} = \mathbf{k} \mathbf{P}^{\mathbf{z}} \mathbf{e}^{-\mathbf{E}/\mathbf{R}\mathbf{T}} \tag{29}$$

From 800° to 2065° C (1472° - 3750° F) z=0.32-0.38 and E=3600-4200 cal/mol based on results of Gulbransen (34) and this study on Speer 710 and RVA graphites (5). Although the activation energies observed in the present study (5) are comparable to those reported by Gulbransen et al., the rates are much higher in the present investigation. Thus, for $PO_2=150$ torr, at $T=1700^{\circ}$ F, Eq. 28 yields a rate of about 0.04 mils/minute. The present study indicates 60 mils/minute at 250 fps. Eq. 28 converts to

$$\dot{m} = 3.16 \times 10^{-5} P_{O_2}^{0.32} e^{-3600/RT} lbs/ft^2 sec$$
 (30)

where P_{O_2} is in atmospheres and T is in ${}^{O}K$, and

$$\dot{S} = 3.39 \times 10^{-3} P_{O_2}^{0.32} e^{-3600/RT}$$
 mils/sec (31)

for graphite with a density $\rho = 1.80 \text{ gms/cm}^3$ or 112 lbs/ft³. These reaction kinetic equations are altered (7) to reflect diffusion control (31) yielding

$$\dot{m} = k(C_{O_2, e}^{M/M}O_2)^2 P_e^2 (1-m/m_D)^2 e^{-E/RT} lbs/ft^2 sec$$
 (32)

where P_e is the stagnation pressure and C_{O2} , e is the mass fraction of oxygen at the edge of the boundary layer.

If one considers the reaction

$$C + 1/2O_2 + 2N_2 \rightarrow CO + 2N_2$$
 (33)

as dominant, then M = 28.5, $M_{O_2} = 32$ and $C_{O_2, e} = 0.21$. In addition (7, 32)

$$\dot{m}_{\rm D} = 0.006 \left(F_{\rm e}/R_{\rm B} \right)^{1/2} \, {\rm ibs/m}^2 {\rm sec}$$
 (34)

Thus, Eq. 32 describes the oxidation of graphite in terms of an Arrhenius term (ke^{-E}/RT) and a pressure correction term $(1-m/m_D)$ which is based on a diffusion limit m_D given by Eq. 34. In other terms, the $(1-m/m_D)$ coefficient can be considered as a pressure correction which relates the exygen concentration at the edge of the boundary layer to that at the graphite surface. Divergent values have been reported for the Arrhenius constants k and E(5). Part of the difficulty undoubtedly arises from the fact that CO₂ is the dominant product gas of graphite oxidation at low temperature while CO dominates at high temperature (36). In contrast to the values $k = 3.16 \times 10^{-5}$ and E = 3600 implied by Eq. 30, Scala and Gilbert (32) have proposed $k = 6.729 \times 10^8$ and E = 44,000 for "fast kinetics" and $k = 4.473 \times 10^4$ and E = 42,300 for "slow kinetics". The current results indicate that lower values of the activation energy may be more appropriate (5).

As indicated below the results of the present study (5, 30) have been examined in order to obtain the most appropriate values of k and E. To begin with, Eq. 34 was re-examined in order to allow for air flow rate effects at subsonic velocities since the diffusion limited rate for carbon removal, \dot{m}_{KC} , was originally approximated by (7)

$$\dot{\mathbf{m}}_{\mathrm{KD}} = \mathbf{A}(\mathbf{P}_{\mathrm{e}}/\mathbf{R}_{\mathrm{B}})^{1/2} \mathbf{C}_{\mathrm{K}} \tag{35}$$

where C_K is the mass fraction of carbon in the oxidation reaction. Since the average molecular weight in Eq. 33 is 25.8 (with $M_{O_2} = 32$) then $C_K = 1/7$. The coefficient "A" was estimated by analogy with the Fay-Riddell relation (22, 27)

$$q = 0.042 (P_e/R_B)^{1/2} (i_e-i_w)$$
 (36)

where the enthalpy difference (i_e - i_w) is the analog of C_K in the mass loss relation. Thus, with A = 0.042, m_D is defined by E_d . 34. However, at low velocities a Mach Number correction (of Eq. 25) is required and the result yields (30)

$$\dot{m}_{\rm D} = 0.0072 (1 + 0.17 {\rm M}^{-1})^{-1} ({\rm P_e/R_B})^{1/2} {\rm lbs/ft}^2 {\rm sec}$$
 (37)

with

$$\dot{m} = 0.74 P_{O_2}^{1/3} e^{-10.730/RT} lbs/ft^2 sec$$
 (38)

for the reaction rate, then

$$\dot{\mathbf{m}} = (0.170)^{1/3} \left[1 - \dot{\mathbf{m}} (\mathbf{R}_{\mathbf{B}})^{1/2} / 0.0072 \left(1 + 0.17 \mathbf{M}^{-1} \right)^{-1} \mathbf{P}_{\mathbf{e}}^{1/2} \right]^{1/3}$$

$$\dot{\mathbf{m}} = (0.170)^{1/3} \left[1 - \dot{\mathbf{m}} (\mathbf{R}_{\mathbf{B}})^{1/2} / 0.0072 \left(1 + 0.17 \mathbf{M}^{-1} \right)^{-1} \mathbf{P}_{\mathbf{e}}^{1/2} \right]^{1/3}$$

$$\dot{\mathbf{m}} = (0.170)^{1/3} \left[1 - \dot{\mathbf{m}} (\mathbf{R}_{\mathbf{B}})^{1/2} / 0.0072 \left(1 + 0.17 \mathbf{M}^{-1} \right)^{-1} \mathbf{P}_{\mathbf{e}}^{1/2} \right]^{1/3}$$

where R is the body radius in feet. Since

$$\dot{S}$$
 (mils/sec) = \dot{m} (lbs/ft²sec) 12000/ ρ (lbs/ft³) (40)

Hence

$$\dot{S} = 4920 \rho^{-1} P_e^{1/3} \exp(-9720/T) \left[1 - \frac{0.0116 \rho R_B^{1/2} P_e^{-1/2}}{(1 + 0.17M^{-1})^{-1}}\right]$$

mils/sec (41)

for the carbon recession rate in air, where ρ is the density of graphite in lbs/ft³ and T is in ${}^{O}R$. The activation energy and pre-exponential factor in Eq. 38 were evaluated by employing the form of Eq. 41 and the data in Table 19 in order to obtain the best fit.

The experimental results shown in Table 19 include results for ATJ, RVA(B-5), PT0178(B-9) and Poco Graphite (B-10). The calculations refer to flat faced cylinders (RB = 2.5 Rc) exposed during the present study by R. A. Tanzilli (13), and by Metzger et al. (12). Exposures performed in the current study are designated by run numbers (6). Tanzilli's exposures of PT0178(B-9) are designated by B9-Gel-18 in Table 19. The exposures of ATJ graphite by Metzger et al. are denoted by MED. In addition to these arc plasma exposures, calculations were performed for serveral high velocity CG/HW tests conducted on RVA(B-5) (5). Exposure B5-L2, B5-L3 and B5-L4 were taken to represent the results contained in Figure 45 of (5) at 200 ft/sec. Examination of the samples after exposure indicates a nose radius equal to one sixteenth of an inch as shown in Figure 44 of (5), hence Rp = 62 mils or 0.0052 ft. The data and calculations cover temperatures between 2300° and 6500°R. Thus, it is possible that some of the high temperature points (T > 6000°R) reflect recession via diffusion limited vaporization where the observed values would be expected to exceed Eq. 41 since vaporization becomes a significant factor in the recession rate above 6000 R at one atmosphere. The present data cover the pressure range between 0.007 and 1 atmosphere and represent RVA(B-5), PT0179(B-9) and Poco(B-10) graphite. Moreover, the results include data generated in the HG/CW Avco Model 500 and ROVERS facilities (3) and the General Electric Tandem Gerdien Arc as well as the Lockheed M/S Co., CG/HW facility (2).

In addition, it should be pointed out that an independent experimental arc plasma study by Sallis et al. (37) between 2500° and 5000°P at stagnation pressures between 0.1 and 9 atmospheres yields a pre-exponential of 3.0 and an activation energy of 15,500 cal/mole as compared with the current results of 0.74 and 10,730 cal/mole shown in Eq. 28. In addition, Sallis et al. suggest that the exponent of the oxygen pressure is 0.4 rather than the value of 0.333 indicated by Eq. 38.

Eq. 41 has been employed to generate Figures 22-26, which compare with the observed and computed results of graphite oxidation at low velocities. Figure 22 shows excellent agreement for RVA(B-5) at 3310 R (Figure 37 of reference 5), while Figure 23 shows qualitative agreement (see Figure 45 of reference 5). While it is apparent that Eq. 41 provides an excellent description for the oxidation kinetics of graphite over a very wide range of conditions shown in Table 19, it is of interest to consider the results of a different kind of experiment relative to the predictions of Eq. 41.

Blyholder and Eyring (38) have measured the oxidation of graphite in flowing oxygen at 800° K. The oxygen pressure employed was 26 microns of mercury while the flow rate was 1000 cm/sec. The samples employed in this experiment were hollow cylinders of carbon having a density of 1.3 gms/cm³ (81 lbs/ft³) which were 1/4 inch in diameter with a one millimeter wall thickness. The samples were cut in half parallel to their longitudinal axes prior to exposure. Under these conditions, the oxidation rate of carbon (as measured by formation of CO) corresponded to 16×10^{15} molecules of CO/cm²sec. This corresponds to 3.2×10^{-6} gms of carbon/cm²sec. In order to apply Eq. 41 to compute the rate of carbon oxidation for comparison with this result is is necessary to estimate the value of the term within the braces in Eq. 41 first. Since the flow rate was 1000 cm/sec or about 33 ft/sec, M is approximately 0.03. The corresponding value of P_e is equal to 3.4×10^{-5} atm. (26μ of Hg) multiplied by (1.00/0.21) or 160×10^{-6} atm. Estimating $R_B \approx 2.5$ RC leads to $R_B \approx 0.026$ ft. Thus, Eq. 41 becomes

$$\dot{S} = 3.85 \times 10^{-3} (1-80\dot{S})^{1/3} \text{ mils/sec}$$
 (42)

or $\dot{S}=3.4\times10^{-3}$ mils/sec = 8.6 x 10⁻⁶ cm/sec = 11.2 x 10⁻⁶ gms/cm²sec. This value is in good agreement with the observed value of 3.2 x 10⁻⁶ gm/cm²sec in view of the estimates required for R_B and M. Moreover, this experiment was performed far from the range of conditions employed to fix the pre-exponential and activation energy in Eq. 38.

C. Comparison of the Scala-Gilbert and John-Schick Models for Graphite Ablation

John and Schick (33) have developed a theory for describing the diffusion controlled ablation of graphite which defines the linear recession rate, S(mils/sec), of graphite as follows:

$$\dot{S} = 12,000q_{HW} \rho^{-1} (i_e - i_w)^{-1} \left[\frac{M_{O_2}}{2M_C C_{O_2}} + \eta \right]^{-1} mils/sec$$
 (43)

where ρ is the density of the graphite (lbs/ft³), $i_e(BTU/lb)$ is the stagnation enthalpy, $i_w(BTU/lb)$ is the enthalpy at the wall, $M_{O2}=32$ and $M_C=12$ are the molecular weights of oxygen and carbon, $C_{O2}=0.21$ is the mass fraction of oxygen in air, η is a blowing factor, and q_{HW} is the hot wall heat flux. If we set $\eta=0.67$, which is the usual value, and

$$\frac{\mathbf{q}_{CW}}{\mathbf{i}_{e}} = \frac{\mathbf{q}_{HW}}{(\mathbf{i}_{e} - \mathbf{i}_{w})} \tag{44}$$

Then Eq. 43 becomes

$$\dot{S} = 12,000 \rho^{-1} (q_{cw}/i_e) 7^{-1} \text{ mils/sec}$$
 (45)

or

$$\dot{\mathbf{m}} = \mathbf{h}_{\mathbf{c}}(7)^{-1} \quad \mathbf{lbs/ft}^2 \mathbf{sec} \tag{45}$$

where hais the heat transfer coefficient. Setting

$$h_e = 0.042 (P_e/R_B)^{1/2}$$
 (47)

based on the Fay-Riddell relation at high Mach Numbers yields

$$\dot{m} = 0.006(P_e/R_B)^{1/2} lbs/ft^2 sec$$
 (48)

Eq. 48 is identical to the result obtained by Scala and Gilbert (32) in the diffusion limited range. Thus, it is apparent that

IV. INFLUENCE OF FLUX-ENTHALPY AND ALTITUDE-VELOCITY VARIABLES ON THE RECESSION OF REFRACTORY MATERIALS

Figures 1-8 of reference(6) describe the 30 minute recession or oxidation depths observed for the candidate materials in MC/CW are plasma tests and in CG/HW furnace tests as a function of surface temperature. In the former case, the surface temperature is a tesult of the interaction of the material with the stream. Thus, while comparison of a given material in the CG/HW and HG/CW cases based upon results at a given temperature is quite legitimate, evaluation of various materials in HG/CW tests on solely a temperature basis is not complete. As indicated earlier (6) factors such as emittance, exidation products and surface characteristics can lead to situations where identical stream conditions produce a variety of surface temperatures on different materials even after long exposure times. In order to consider an alternative method for comparison of the exidation characteristics and to relate the HG/CW tests to flight parameters, an additional description can be employed.

Figure 27 shows the stagnation pressure as a function of altitude and velocity. In addition, stagnation heat flux to a one inch sphere described as a function of altitude and velocity is displayed. In essence, these curves provide a means for relating the HC/CW are plasma tests to flight trajectory conditions. The relations are presented for the case of a one inch radius sphere. Since the heat flux is proportional to R-1/2, the heat flux to a 4 inch sphere would be one half of the values shown on the ordinate of Figure 27, while the heat flux to a 1/4 inch sphere would be twice those shown in Figure 27. The curves relating stagnation pressure, Pe, altitude, A, and velocity, V, in Figure 27 are the results of complex equations (39-41). However, between 50 Kilo feet and 250 Kilo feet, these curves can be represented simply by Eq. 50 as:

$$P_0 = V^2 (1 + (A/216)^4) 10^{-A/54} atm$$
 (50)

where P_e is the stagnation pressure behind a normal shock in atmospheres, V is the velocity in kilo feet/see and A is the altitude in kilo feet. The stagnation enthalpy, i_e , is approximately

$$i_a = 20V^2 BTU/1b$$
 (51)

Eqs. 50 and 51 when coupled with the Fay-Riddell relation for a hemisphere

$$q = 0.042 i_e (P_e/R_B)^{1/2} BTU/ft^2 sec$$
 (52)

where R_R is the body radius in feet yields

$$q = 2.9V^3 (1 + (A/216)^4)^{1/2} R_B^{-1/2} 10^{-A/108}$$
 (53)

where RB is the body mdius in inches.

Thus, Figure 27, or the approximations afforded by Eqs. 50-53 permit direct conversion from altitude-velocity space to flux-enthalpy space for a hemisphere. As indicated above, the behavior of the refractory materials of interest under HG/GW conditions simulating high velocity atmosphere flight is presented in Figures 1-8 of reference 6. The latter show material recession as a function of stagnation pressure and surface temperature. In this case, the surface temperature results from a combination of q, ie and material reactions with the stream. The relations between q, ie, Pe and surface temperature (for radiation equilibrium) is illustrated in Figures 1 to 15. Calculations of this type have been performed for each HG/CW exposure (6). Although reasonable agreement of computed T[ie,q,Pe] with observed surface temperature has been encountered, some systematic differences have been noted as shown in Table 18. Specifically, SiC and SiC bearing composites reach lower surface temperatures for given ie, q and Pe conditions than do the other materials considered. Similar behavior is noted for WSi2/W and SnAl/Ta-10W under conditions where the coatings do not fail.

An alternative method for describing the performance of these refractory materials is illustrated in Figures 28-36. These figures show heat flux as the ordinate and stagnation enthalpy as the abscissa. In addition, velocity as related to stagnation enthalpy by Eq. 51 is shown as the abscissa. Moreover, the relationship for a one inch radius sphere at 150 kilo feet is the ordinate. Thus, for a velocity of 16 kft/sec at an altitude of 150 kft, the stagnation enthalpy would be approximately 5100 BTU/lb and the stagnation heat flux to a one inch radius sphere would be about 600 BTU/ft²sec. If the body radius were 4¹¹, the heat flux would be 300 BTU/ft²sec (located by dropping down to 300 on the inner ordinate scale). Conversely, if the body radius were 1/4", the heat flux would be 1200 BTU/ft sec (located by moving up to 1200 on the inner ordinate scale). Thus, the double set of ordinate and abscissa scales permit direct translation of velocity and body radius to flux altitudes. Eq. 53 has been employed to construct the inserted curve shown on each graph which shows the ratio of q[Altitude] / q[150 kft] as a function of altitude. This ratio is 0.27, 0.48, 1.00, 2.46 and 6.85 at 250, 200, 150, 100 and 50 kft, respectively. Thus, at a velocity of 16 kft/sec and 250 kft altitude, the heat flux to a one inch sphere would be 600 x 0.27 or 162 BTU/ft²sec. Under these conditions a 4" radius would experience a heat flux of 81 BTU/ft²sec while a 1/4" radius sphere would be exposed to a heat flux of 324 BTU/ft²sec. Similarly at an altitude of 50 kft, the heat flux to a one inch radius sphere would be 2.46 x 600 = 1476 BTU/ft²sec and the flux to a 4" and 1/4" sphere would be 738 and 2952 BTU/ft²sec, respectively.

Thus, Figures 28-36 show heat flux and enthalpy for any velocity, altitude and body radius. Figure 28 shows the recession rates observed for hafnium diboride at stagnation pressures of one atmosphere (circles) and 0.01-0.1 atmospheres (squares). Recession rates which are less than 0.1 mils/sec, between 0.1 and 1 mil/sec and more than one mil/sec are indicated by open, half-filled and filled points, respectively. Recession

rates are plotted at flux and enthalpy co-ordinates for exposures given in reference 6. At present, sufficient data are not available to construct boundaries representing constant recession levels over the entire flux-enthalpy space. To bridge the gap, 1.0 mil/sec boundaries are approximated at each pressure by temperature levels obtained from Figures 1-15. Figure 36 summarizes all of the results for a 1 mil/sec boundary at one atmosphere.

The hyperbolic curves for all of the materials except graphites and tungsten, define the flux-enthalpy (or velocity-altitude-body radius) conditions where the recession rate exceeds 1 mil/sec. Flux-enthalpy conditions below and to the left of these boundaries result in recession rates which are less than 1 mil/sec. Flux-enthalpy conditions above and to the right of these boundaries yield recession rates in excess of 1 mil/sec. In the case of graphites and tungsten, the linear boundaries (based on diffusion limits) are shown for 0.5, 1.0, 2.0 and 4.0 mils/sec. Flux-enthalpy conditions lying below and to the right of these boundaries will result in lower recession rates, while those lying above and to the left will result in higher rates. The computed rates for graphite and tungsten are calculated on the basis of Reference 31 and Eq. 45.

Although Figure 36 is based on a limited number of tests, it provides a clear indication of the superiority of SiC and SiC composites. Figure 1 of reference 6 suggests that $ZrB_2(A-3)$ exhibits recession rates below 1 mil/sec at surface temperatures up to 5000°F. Figure 5 of Reference 6 shows that KT-SiC(E-14) exhibits rates below 1 mil/sec or less below 4000°F. However, Table 18 as well as Figure 36 indicate that the fluxenthalpy conditions which produce 5000°F surface temperature (and a 1 mil/sec recession rate) for ZrB2(A-3) will yield a surface temperature of only 4000°F (and a comparable recession rate) for KT-SiC(E-14). Of course, relative mechanical properties, thermal shock resistance, density and other factors may impose additional criteria for comparison. Nevertheless, Figure 36 presents a direct ranking of oxidation behavior as a function of flight conditions for extended periods of time at the stagnation point. The position of these curves could vary with stagnation pressure. There are several other inversions of rank relative to the CG/HW and HG/ CW tests using temperature as a base. Figures 42 and 45 of reference 6 show that HfB₂ + SiC(A-4) and (A-7) exhibit 1 mil/sec recession rates at lower temperatures than HfC + C(C-11). Nevertheless, for given stream conditions, the latter reaches surface temperatures which are 20% higher than the former (see Table 18) and ranks lower in Figure 36. Similarly, $SiO_2 + W(H-22)$ and Si/RVC(B-8) which degrade rapidly at $4000^{O}F$ and 3100°F respectively in CG/HW furnace tests rank high on the basis of Figure 36. The relative behavior of Si/RVC(B-8) and Ir/C(I-24) is also illustrated graphically in Figure 36. Reference to Figures 5 and 8 of reference 6 shows that Si/RVC(B-8) provides protection for graph.:e at surface temperatures up to 3700°F in the HG/CW are plasma tests while Ir/C(I-24) is protective up to surface temperatures of 4200°F. However, since the latter has an emittance of 0.3 and T(CALC)/T(OBS) = 1.16 (Table 18) while the former has an emittance of 0.70 and T(CALC)/T(OBS)= 1.43, Si/RVA(B-8) exhibits a much greater resistance to heat flux and enthalpy than does Ir/C(I-24). Addition of HfO2 to the Ir/C(I-24) coating system improves the performance of this material by increasing its emittance (6).

V. UTILIZATION OF THE FLUX-ENTHALPY VS. ALTITUDE-VELOCITY CORRELATION TO SCREEN MATERIALS FOR SPECIFIED TRAJECTORIES

In order to illustrate the wasans by which Figure 36 can be employed to predict the behavior of a candidate refractory material for a specific trajectory it is worthwhile to consider the Air Force Flight Dynamics Laboratory FDI -7MC lifting reentry vehicle's maximum cross range characteristics. This vehicle is designed for a Lift/Drag ratio between 2.5 and 3.0.

Table 20 provides the altitude and velocity as a function of time. These data were employed to calculate stagnation enthalpy, pressure and heat flux based on Eqs. 50-53 for a 3" nose radius. The results are shown in Table 20 and in Figures 37-40. These figures indicate that HfB2+SiC (A-4) and (A-7), HfB2(A-2), ZrB2+SiC(A-8) and HfC+C(C-11) could survive the entire trajectory in this configuration. In addition, ZrB2(A-3) and ZrB2+SiC+C(A-10) might also survive. Comparison of the flux-enthalpy values for this trajectory with the results of arc plasma tests shows that $HfB_2(A-2)$, $ZrB_2(A-3)$, $HfB_2+SiC(A-4)$ and (A-7), $ZrB_2+SiC(A-8)$, and ZrB2+SiC+C(A-10) survived tests which are equivalent to the FDL-7MC with very small recessions. Due to the low temperature oxidation of HfC+C(C-11), this material might be limited for reuse (6). The borides and boride composites would not suffer from this limitation. A number of long-time cyclic exposures of diboride composites have been performed (6) in the Model 500 and ROVERS facilities to evaluate reuse capabilities for trajectories typified by FDL-7MC. The results provide a striking illustration of the reuse capability of these materials for lifting reentry applications.

Sample HfB2. 1+20%SiC(A-7)-28R was exposed for thirteen cycles at 0.07 atm stagnation pressure, a stagnation enthalpy of 10,200 BTU/lb and a coldwall heat flux of 495 BTU/ft²sec. Each cycle was about 1800 seconds long with a total exposure time of 22,500 seconds. The surface temperature increased from one cycle to the next starting at 3500°R and holding near 5300°R for cycles 5 through 13. Total material recession was 15 mils after this extremely long exposure. Sample ZrB2.1+20%SiC(A-8)-15M was exposed for four cycles at 1.0 atm stagnation pressure, a stagnation enthalpy of 5000 BTU/lb and a cold wall heat flux of 380 BTU/ft²sec. Each cycle was 1800 seconds long, total exposure time was 7200 seconds. The surface temperatures were near 5000°R. Total material recession was 26 mils. Finally, sample ZrB2+SiC+C(A-10)-26R was exposed at 0.236 atmospheres stagnation pressure, a stagnation enthalpy of 7700 BTU/lb and a cold wall heat flux of 455 BTU/ft²sec. This test covered eleven cycles of approximately 1800 seconds duration for a total exposure time of 18,900 seconds. Surface temperature held near 5100°R after the first cycle. Total material recession was 83 mils.

These results illustrate the reuse capability of boride composites for lifting reentry application, since they exceed the range of conditions and FDL-7MC. This capability is unrivaled by any other materials system.

VI. CALCULATION OF THE FLUX-ENTHALPY BOUNDARIES FOR RECESSION RATES OF 1 MIL/SEC VIA MELTING

Figures 28-40 display the recession rates observed in arc plasma tests as a function of heat flux and stagnation enthalpy. This representation indicates the locus of heat flux, q, and stagnation enthalpies, i.e., which define the region where recession rates exceed 1 mil/sec for refractory materials. This value was arbitrarily chosen in order to illustrate a means by which the flux-enthalpy representation can be employed. The 1 mil/sec boundaries can be identified by collecting sufficient data to cover the q-ie space completely. Since sufficient data points are not available to do this experimentally, the procedure employed in Figures 28-35 is to associate the 1 mil/sec boundary for condensed oxide forming refractory materials with a specific temperature. This procedure is quite arbitrary.

An alternative method is to consider the 1 mil/sec rate as being characteristic of melting. Under these conditions, the convective heat flux can be considered as a source of the radiative losses and the heat of melting. This heat balance is defined by Eq. 54 as follows:

$$h_e(i_e-i_w[T,P]) = \sigma \epsilon (T/1000)^4 + (12,000)^{-1}\rho \Delta H_f \dot{S} BTU/ft^2 sec$$
 (54)

where the heat transfer coefficient, he, is

$$\mathbf{h_e} = \mathbf{q_{cw}}/\mathbf{i_e} \tag{55}$$

The enthalpy of air at the wall, $i_w[T,P]$, is given by the following expression from Eq. (4),

$$i_{\rm w}[T,P] = (T/1000) (100 + 45 \log P + (T/1000) (46-14 \log P)) BTU/1b$$
 (56)

and \mathbf{q}_{cw} and $\mathbf{i}_{\mathbf{e}}$ are the cold wall heat flux and stagnation enthalpy, respectively.

In Eq. 54, 0=0.47 BTU/ft²sec^OR⁴ is Boltzmann's constant, T^OR is the melting point of the refractory material, ρ is the density in lbs/ft³, ϵ is the total normal emittance, ΔH_f is the latent heat of melting in BTU/lb and S is the recession rate in mils/sec. Since there is not available experimental data for ΔH_f , estimates have been made as indicated in Table 21. Thus, Eq. 54 is similar to the radiation equilibrium surface temperature calculation except that an additional heat loss $(12,000^{-1}\rho\Delta H_f S)$ has been included. Rearrangement yields:

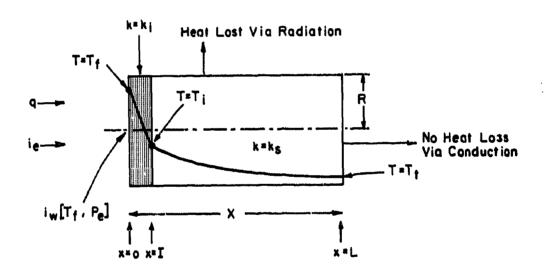
$$\dot{S} = \frac{(q/i_e) (i_e - i_w [T, P]) - \sigma_\epsilon (T/1000)^4}{(12,000)^{-1} \rho \Delta H_f} \quad mils/sec$$
 (57)

Eq. 57 defines the recession rate for melting under conditions where the convective heat flux is balanced by the radiated heat flux and the heat flux required to melt the material at a fixed recession rate. Table 21 contains values of T, ϵ , ΔH_f and ρ for the candidate materials. The latent heat of fusion has been estimated for most of these materials since virtually no measurements are available. Figures 41-44 show sample results obtained for HfB_{2.1}(A-2), ZrB₂(A-3), HfC+C(C-11) and ZrC+C(C-12) by setting $\dot{S}=1$, 3 and 10 mils/sec in Eq. 56. The location and form of these curves is in qualitative agreement with the results displayed in Figures 28-35.

VII. CALCULATION OF TEMPERATURE DISTRIBUTION THROUGH A COMPOSITE CYLINDER UNDER STEADY STATE CONDITIONS ALLOWING FOR SIDE LOSSES VIA RADIATION

The calculation of surface temperature from stream conditions based on radiation equilibrium described by Equations (1-4) in Section VII has been performed for all tests. The results which are given in Table 18 show that reasonable agreement can be obtained in most cases to within 10 or 15%. In most cases, the calculated temperature is too high. At present, the results indicate that the over estimated values are due to a reduction in heat transfer coefficient due to vaporizing oxidation products. This conclusion is based on the fact that materials containing SiC, SiO₂ and other vapor products (i. e. Sn-Al/Ta-10W(G-19) yield higher values of T(CALC)/T(OBS) as seen in Table 18. The following calculation deals with the problem of side losses via radiation and axial temperature gradients in the arc plasma test cylinders.

Representation of the steady state temperature distribution through a right circular cylinder of radius R, length L, and a coating which has a thickness I as shown below can be defined on the basis of a convective heat input and radiation losses from the front and sides according to the model shown above. In this description, the thermal conductivity of the coating and substrate are \mathbf{k}_i and \mathbf{k}_s , respectively.



The total heat balance requires that

$$(q/i_e) (i_e-i_w(T_f, P_e)) = 0 \in T_f^4 (1 + (2L/R) (F_F + F_S)) BTU/ft^2 sec$$
 (58)

where q and i are the cold wall heat flux and stagnation enthalpy respectively, and the enthalpy at the wall, i,, at a temperature T_f and a stagnation pressure P_a is approximately (from Eq. (4)),

$$i_w[T_f, P_e] = (T_f/1000) (100 + 45 \log P_e + (T_f/1000)(46-14 \log P_e)) BTU/1b$$
(59)

Evaluation of the temperature distributions along the length of the cylinder are performed on the basis of the following assumptions,

- (a) the gradient through the coating is linear for $0 \le x \le I$
- (b) the temperature distribution through the substrate is quadratic for $I < y < \underline{L}$
- (c) heat losses at x = L are negligible
- (d) there are no radial temperature gradients.

With these assumptions, the quantities F_F and F_S contained on the right side of Eq. (58) are defined as follows

$$F_{\mathbf{F}} = \int_{0}^{\overline{x}_{i}} (T/T_{f})^{4} d\overline{x}$$
 (60)

where $\bar{x} = x/L$, $\bar{x}_i = I/L$ and T is the temperature at any value of x. Similarly,

$$\mathbf{F}_{S} = \int_{\mathbf{X}_{i}}^{1.0} (\epsilon_{S}/\epsilon) (\mathbf{T}/\mathbf{T}_{i})^{4} d\vec{\mathbf{x}}$$
 (61)

where ϵ_S is the emittance of the substrate and ϵ is the emittance of the coating. If the ratio T/T_f is defined as T_i then

$$\overline{T} = 1 - (\overline{T}_i - 1) (\overline{x}/\overline{x}_i) \text{ for } 0 \le \overline{x} \le \overline{x}_i$$
 (62)

where \overline{T}_i is equal to T/T_f at x = I, and

$$\overline{T} = (1 - \overline{x}_i)^{-2} \left[\overline{T}_i + (\overline{T}_t - \overline{T}_i) \overline{x} (2 - \overline{x}_i) - \overline{T}_t \overline{x}_i (2 - \overline{x}_i) \right] \text{ for } \overline{x}_i \leq \overline{x} \leq 1$$
(63)

where $\overline{T}_t = T_t/T_f$ is the ratio of the back face temperature T_t to the front face temperature T_f . Matching of the conductive fluxes at the interface requires that

$$T_i/T_f = \widetilde{T}_i = \left[2\widetilde{x}_i + \overline{k}(1 - \widetilde{x}_i)\right]^{-1} \left[2\widetilde{x}_i \widetilde{T}_t + \overline{k}(1 - \widetilde{x}_i) - C_S \widetilde{x}_i (1 - \widetilde{x}_i)\right] \widetilde{T}^4 dx$$

where $\bar{k} = k_F/k_S$ is the ratio of thermal conductivities of the coating and substrate and C_S is defined by Eq. (65) as follows

$$C_{S} = 2 \epsilon \sigma T_{f}^{3} L^{2} / Rk_{S}$$
 (65)

Eqs (62)and (64)permit evaluation of FF since

$$F_F = \int_0^{\bar{x}_i} \bar{T}^4 dx = \int_0^{\bar{x}_i} (1 + \bar{x}(\bar{T}_i - 1)\bar{x}_i^{-1})^4 d\bar{x}$$

$$\mathbf{F}_{\mathbf{F}} = \bar{\mathbf{x}}_{i} \left[1 + 2(\bar{\mathbf{T}}_{i} - 1) + 2(\bar{\mathbf{T}}_{i} - 1)^{2} + (\bar{\mathbf{T}}_{i} - 1)^{3} + (1/5)(\bar{\mathbf{T}}_{i} - 1)^{4} \right].$$
 (66)

Similarly, defining

$$a_3 = [\overline{T}_i - \overline{T}_t \overline{x}_i (2 - \overline{x}_i)] (1 - \overline{x}_i)^{-2} \text{ and } a_4 = (\overline{T}_t - \overline{T}_i) a_3^{-1} (1 - x_i)^{-2}$$
 (67)

yields

$$(\mathbf{z}/\epsilon_{S})\mathbf{F}_{S} = \int_{\bar{x}_{i}}^{1.0} \mathbf{a}_{3}^{4} \left[1 + \mathbf{a}_{4}\bar{\mathbf{x}}(2 - \bar{\mathbf{x}})\right]^{4} d\mathbf{x}$$

$$= \mathbf{a}_{3}^{4} \text{ times}$$

$$\begin{bmatrix} \mathbf{a}_{4}^{4} \left(\frac{128}{315} + \mathbf{x}_{i}^{5} \left[-\frac{1}{9} \ \bar{\mathbf{x}}_{i}^{4} + \bar{\mathbf{x}}_{i}^{3} - \frac{24}{7} \ \bar{\mathbf{x}}_{i}^{2} + \frac{16}{3} \ \bar{\mathbf{x}}_{i} - \frac{16}{5} \right] \right)$$

$$+ \mathbf{a}_{4}^{3} \left(\frac{64}{35} + 4\bar{\mathbf{x}}_{i}^{4} \left[\frac{1}{7} \ \bar{\mathbf{x}}_{i}^{3} - \bar{\mathbf{x}}_{i}^{2} + \frac{12}{5} \ \bar{\mathbf{x}}_{i} - 2 \right] \right)$$

$$+ \mathbf{a}_{4}^{2} \left(\frac{16}{5} + 2\mathbf{x}_{i}^{3} \left[-\frac{3}{5} \ \bar{\mathbf{x}}_{i}^{2} + 3\bar{\mathbf{x}}_{i} - 4 \right] \right)$$

$$+ \mathbf{a}_{4} \left(\frac{8}{3} + 4\bar{\mathbf{x}}_{i}^{2} \left[\frac{1}{3} \ \bar{\mathbf{x}}_{i} - 1 \right] \right)$$

$$+ 1 - \bar{\mathbf{x}}_{i}$$

$$(69)$$

(69)

Eq. (63) can be employed to define the temperature gradient in the cylinder at the front face

$$-\left(\frac{d\overline{T}}{dx}\right)_{x=0} = (1-\overline{T}_{i})\overline{z}_{i}^{-1} \tag{70}$$

Since all of the energy entering the cylinder at its front face must be radiated away, then

$$-\left(\frac{d\vec{T}}{d\vec{x}}\right)_{\vec{x}=0} = \frac{1-\vec{T}_{i}}{\vec{x}_{i}} = C_{S}k^{-1}\int_{0}^{1} \vec{T}^{4} d\vec{x} = C_{S}k^{-1}(F_{F}+F_{S})$$
 (71)

or

$$(\mathbf{F}_{\mathbf{F}} + \mathbf{F}_{\mathbf{S}}) = (1 - \overline{\mathbf{T}}_{\mathbf{i}}) \overline{\mathbf{k}} / \overline{\mathbf{x}}_{\mathbf{i}} C_{\mathbf{S}}$$
 (72)

When $\bar{x}_i = I = 0$, i.e., no coating is present, Eq. (62) yields

$$-\left(\frac{d\vec{T}}{d\vec{x}}\right)_{\vec{x}=0} = 2(1-\vec{T}_t) = C_S F_S \tag{73}$$

or

$$\mathbf{F}_{\mathbf{S}} = 2(1 - \overline{\mathbf{T}}_{\mathbf{r}})/C_{\mathbf{S}} \tag{74}$$

The limiting case where I = L = 0, indicating no radiation losses from the sides, reduces Eq. (58) to

$$(q/i_e)(i_e - i_w [T_f, P_e]) = \sigma_e T_f^4$$
 (75)

for radiation equilibrium from the front face. This result is identical to Equations 1-4 of Section II.

Solution of Eq. (58) for the general case where I and L are not equal to zero requires location by iteration of the proper value of T_c which satisfies Eqs. (58), (64), (73), (68) and (72). This procedure is started by employing the solution of Eq. (58) (based on I=L=0) for T_c as a first guess. Once this first guess is available, C_c is defined by Eq. (65). These initial values of T_c and C_c are employed in conjunction with Eqs. (64). (66) and (68) to search for a value of T_c satisfying (72). The iteration procedure is begun by substituting the resultant value of $(F_c + F_c)$ into Eq. (58) and solving for the resultant error. The procedure is repeated by perturbing the initial T_c , repeating the solution and obtaining the resultant error for the second choice. Examination of these values of T_c and the resultant errors, using a Newton-Raphson method, permits the iteration process to proceed to convergence.

When I=0 the procedure is the same except that Eq. (72) is replaced by Eq. (64). In this case F_F is zero. A computer program has been developed to perform these calculations.

Sample calculations are shown in Table 22 for arc plasma tests ZrB₂ (A-3)-2MC which exhibited a front face temperature of 4930°R and an internal temperature of 3400°R at a distance of 100 mils from the front face (6). The effects of the internal pyrometer hole was considered theoretically and found to be negligible. Consequently, the present calculations are based on the total length of 429 mils. The results illustrate the large temperature gradients through the sample indicated in reference 6.

In order to gain some insight into the effects of length, L, radius, R, coating thickness. I, and the thermophysical properties, ϵ , ϵ , $k_{\rm F}$ and $k_{\rm S}$ on the temperature gradients, it is instructive to reconsider Equation (58). In Equation 58, the calculation of T, reduces to the simple radiation equilibrium case given by Equation 75 and Equations 1-4 of Section II when L is equal to zero or when R is infinitely large compared to L. When this is not the case, the temperature gradients are controlled by the values of $F_{\rm F}$ and $F_{\rm S}$ in Equation 58. These quantities are in turn dependent on the related values of T and T defined in Equations 62-64. The relation between T and T (Equation 64) specifically involves the thermophysical properties through $C_{\rm S}$ given by Equation 65. The explicit correspondence between the radiation parameters $F_{\rm F}$ and $F_{\rm S}$ and the thermophysical parameters $T_{\rm i}$, $T_{\rm c}$ and $T_{\rm c}$ specified by Equations 72 or 74. Figure 45 shows the variation of $T_{\rm c}$ and $T_{\rm c}$ specified by Equation 64 for a representative case where $k_{\rm F}/k_{\rm S} = 0.10$. Figure 46 illustrates a portion of the coating contribution $(F_{\rm F})$ to the radiation term in Equation 58 and its dependence on $T_{\rm c}$. The total radiation parameter $(F_{\rm F} + F_{\rm S})$ varies with $T_{\rm c}$ as shown in Figure 47. This dependence is illustrated for the case where I/L = 0.10 and $k_{\rm F}/k_{\rm S} = 0.10$ on the basis of Equations 66 and 69. Solutions for $T_{\rm c}$ and $(F_{\rm F} + F_{\rm S})$ are given by the indicated intersections using Equation 72. These calculations are carried out by the computer program which then solves Equation 58. In the special case where I = 0 (no coating present), $F_{\rm F}$ vanishes and $F_{\rm S}$ as given by Equation 69 is solely dependent on $T_{\rm c}$ as shown in the figure based on Equation 74.

The effect of the thermophysical properties and the radiation losses on the surface temperature gradient within the material is evident from Equation 71. For small C_S the temperature gradient is small, as may arise for low surface temperature, (T_f) , physically thin cylinders $(L \to 0)$, relatively thin cylinders $(L/R \to 0)$, or large thermal conductivity (k_f) aft of any coating. For large values of those parameters, C_S , and thus the temperature gradient, tend to be large. Relatively small thermal conductivity of the coating (i.e. $k = k_F/k_S$ small) also leads to larger thermal gradients. Lastly, the normalized radiation parameters, F_F and F_S , influence the gradient by partially compensating for the C_S effect. Figure 47 shows that small $(F_F + F_S)$ is associated with large C_S and vice versa. Physically, this implies larger gradients at the face lead to lesser radiation losses from the aft parts of the cylinder due to the lesser temperatures then present.

Tables 23-28 summarize the results obtained by comparing the observed internal temperatures (6) with calculated values based on Eqs. 58-75 for ZrB2+SiC(A-8), ZrB2(A-3), HfB2+SiC(A-7), RVA(B-5), ZrB2+SiC+C(A-10), WSi2/W (G-18) and Hf-Ta-Mo(I-23). These tables contain the measured front face temperature TF, the observed temperature at a distance, d mils from the front face, T(d), and the cold wall heat flux, q, the stagnation enthalpy, ie, and the stagnation pressure, Pe. In addition, these tables show the radius, R, length, L, and oxide coating thickness, I, The latter was equated to the conversion depth for the oxide formers (6). For WSi2/W, I was equated to the WSi2 coating thickness and I=0 for RVA(B-5) graphite which ablates without coating formation. Values of the emittance, es, taken from Table 18 as well as suitable values of the thermal conductivities of the thermal conductivity of the coating kF and the substrate ks are also shown in Tables 23-28.

The computed results are displayed in terms of the ratio of calculated front face temperature to observed front face temperature T_f (CALC)/ T_f (OBS) and the ratio of computed in depth temperature T_d (CALC) to computed front face temperature T_f (CALC). If the agreement is exact (e.g., Hf-Ta-Mo(I-23)-43R in Table 28), the ratio of T_f (CALC)/ T_f (OBS) would be 1.00. In the example T_f (CALC) is 4440°R vs. 4530°R = T_f (OBS). Similarly the measured temperature at 120 mils is 3560°R vs. T_d (CALC)=3380°R. In this case, the observed gradient is 960°R while the calculated gradient is 1060°R in 120 mils.

All of the runs shown in Tables 23-28 were performed on flat faced cylinders except those designated by a suffix H (hemisphere) or S (cylindrical shroud with a 200 mil wall). Photographs of these models have been presented (6). The shrouds and hemispherical caps did not alter the gradients observed for flat faced cylinders. Thus all of the calculations were based on flat faced cylinders ignoring the hemispherical caps and the shrouds. Reference to Tables 23-28 indicate relatively good agreement between calculation and observation, in view of the simple model employed and the complexities of the experiments.

The largest deviations occur at low surface temperatures (i.e., $T_f < 3300^{\circ}R$) for the materials which form SiO2 as an oxidation product. Thus, in cases where samples of HfB2+SiC(A-7), ZrB2+SiC(A-8), ZrB2+SiC+C(A-10) or WSi2/W(G-18) were exposed with shrouds or as large diameter hemispheres T_f (CALC) is considerably larger than T_f (OBS). However, this difference is smaller than obtained when T_f is computed on the basis of radiation equilibrium (6) (i.e., Eq. 75). The cause of this behavior is presently unknown (6). Reference to Tables 23-28 shows that the calculated and observed ratios of T_d/T_f are in general agreement.

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TABLE 1

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

)/T(OBS) I'ay-Riddell Coefficient		1.42	1.28	1,25	1.08	1.03	1,04	1.07	1.02	1.01	1.14	0.93	1.29	1.15	1.25	1.34	1.44	1.15	1.07	1.13	1.21
Ratio T(CALC)/T(OBS) Cold Wall I'ay-Riddell Heat Transfer Coefficient		1.00	1.05	1.05	0.92	0.94	0.94	0.93	0.91	1.03	1.08	0.94	1,18	1.06	1.11	1.16	1.25	1.02	0.94	1.15	1.23
(OR)		3033	3443	3560	4253	4406	4523	4640	5135	3983	4127	4379	4631	3910	3613	3204	3040	4291	4685	4860	4560
GETU)		27	56	65	92	100	801	112	160	264	260	264	423	100	8	65	89	120	120	2330	2420
D (in.)	•	0.488	0.483	0.484	0.470	0.483	0.480	0.480	0.478	0.500	0.452	0.449	0.451	0.426	0.384	0.385	0.425	0.425	0.425	0-500	0.500
i (BTU) 1b		00	8	F20	004	8	200	001	00,	80	280	720	000	000	00	00	00	8	00	35	1935
		64	71	7	ò	8	6	1	107	9	2	Ξ	4	8	83	72	8	114	12000	1935	<u></u>
Pe (atm)									0.022 107											36.0 193	36.0 19
Mach Pe No. (atm)		0.014	0.013	0.013	0.012	0.012	0.012	0.013		1.19	1.22	1.18	1.12	0.008	0.009	0.008	0.008	0.010	0.010		

* D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 2

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Ratio T(CALC)/::(OBS) Cold Wall Fay-Riddell Heat Transfer Coefficient	1.28	1.20	1,50 0,80 1,25		1.59	1.32	1.35	1,35	1.26	1.20	1.05
Ratio T(Cold Wall Heat Tra	1.03	1.19	1.33 0.69 1.13		1,32	1.23	1.18	1.14	1.28	1.22	1.05 0.98 1.02
(OR)	4091 4901	3443 4415	4307 4865 4973		4163	4523	4667	5171	4460	4560	4630 5350 5180
qcw (BTU) ft²sec			440 90 464				422			2430	180 270 246
D (in.)	0.279* 0.269*	0.301 0.301	0.301 0.302 0.291	4	0.303*	0.300	0.301	0.302	0.250	0.500	0.500 0.500 0.500
i (BTU) 1b	3580 4840	1680 1400	6640 1420 5160		6480	4640	5180 5180	7580	1925	1910	5000 6000 4900
Pe (atm)	1,03	1.18	1.02 1.45 1.06		1.05	1.07	1.10 1.10	1.05	36.0	37.0	0.050 0.085 0.106
Mach No.	0.18	0.47	0.20 0.52 0.36		0.30	0.33	0.43	0.32	2.80	2.80	3.0
Material	HfB ₂ ($\underline{1}$) A-94 A-83	A-211A A-212	A-211B A-213 A-201	HfB_2 -SiC ($\underline{1}$)	A-224	A-196A	A-196B A-238	A-223	$Z_{x}O_{2}$ (8)	Hf-Ta Alloy (8) on Ta-10W	Ta (9) c = 0.45

*D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinde: having a geometrical diameter of D inches.

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TABLE 3

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Materia!	Mach No.	Pe (atm)	i (BTU)	D (in.)	qcw (BTU) ff sec	(^O R) (obs)	Ratio T(CALC)/T(OBS) Cold Wall Fay-Ridde Heat Transfer Coefficient	/T(OBS) Fay-Riddell
Pyrolytic (10) Graphite	0.51 0.66 0.50 0.64	1.00	3860 1300 3740 1280	0.750* 0.750* 0.750* 0.750*	600 270 620 250	5895 4095 5760 4230	1.60 0.94 1.02 0.90	0.94 0.89 0.95
Boron- Pyrolytic Graphite (<u>8</u>)	2.10	72.0	2310 2060	1.250	3500 3100	4560 4760	1.40	1,39
Hafnium- Pyrolytic Graphite (<u>8</u>)	2.20	0.89	2180	0.624	4150	4460	1.41	1.40
Boron- Nitride (<u>8</u>)	2.40 3.10 3.10	56.0 26.0 27.0	1800 1720 1830	0.600	3400 2200 2200	3900 4210 3260	1.42 1.26 1.65	1.42 1.26 1.65
Silicon- Carbide (<u>8</u>)	2.20 3.00 2.60 2.20 2.90	68.7 31.6 47.7 69.1 36.0	1970 1810 2190 2180 - 1970	0.600 0.600 0.600 0.600	3830 2370 3570 4280 3360	6100 5750 5100 5100 4060	0.97 0.94 1.21 1.23 1.42	0.97 0.94 1.21 1.23 1.40
Macrolaminate of Mo, HfO_2 and ThO_2^2 (§)	2.10	74.0	2270	0.400	::	4260	11	1, 54 4

^{*} D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

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TABLE 4

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Ratio T(CALC)/T(OBS) Cold Wall Fay-Riddell Heat Transfer Coefficient	1.49 1.44 1.48 1.70 1.31	1.42	1.23 1.17 1.01 1.08 1.00
Ratio T(CAL Cold Wall Heat Transfe	1.19	1.06	1.23 1.17 1.01 1.08 1.00
(OBS)	4260 4360 4160 4160 3460 4760	4160	4850 4880 5480 4650 5550 5060
qcw (BTU) ft ² sec	300C 4300 4300	4030	3830 3490 4160 3500 3940 3610
(in.)	0.400 0.400 0.400 0.400 0.400	0.600	0.800
i (BTU) lb	2170 2120 2070 2040 1930 2060 1850	1930	2001 1860 1765 1531 1794 1796
P e (atm)	72.0 72.0 71.0 70.0 68.0 86.0	68.0 50	68.1 66.5 52.7 52.4 45.1 38.7
Mach No.	2.10 2.10 2.20 2.20 1.90 2.60	2.20	2.00 2.30 2.40 2.50 2.50
Material	Macrolaminate of Mo, HfO, Th O_2^2 (8)	Macrolaminate of Tungsten and HfO_2 (§) HfC_2 (Column 13 w/o Column 7 tungsten (11)	EH-1 EH-2 EH-3 EH-4 EH-5

 $^{^*}$ D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 5

STEEL STEEL

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Ratio T(CALC)/T(OBS) cld Wall Fay-Riddell Heat Transfer Coefficient	1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.	1.07 1.44 1.45
Ratio T(CALC)/T(OBS) Cold Wall Fay-Ri Heat Transfer Coefficie	2 2 2 2 2 2 2 2	1.07 1.44 1.47
(SE)	3660 4365 4366 3860 3460 3460 4460 4460 4560 4560 4560 4500 4500 4	5590 4200 4000
(BTU)	2370 6950 1160 1160 11730 2380 22850 2200 2200 3520 3520 4600 4600 4230	3560 3570 2740
D (in.)	0.531 0.530 0.530 0.530 0.530 1.000 0.530 0.530 0.530 0.530 0.530 0.530 0.330 0.330	0.600 0.600 0.600
i (BTU) lb	2035 2180 2180 2015 2015 2030 1875 2230 2210 2110 2060 1810 1770 1770 1770	1982 2118 2114
Pe (atm)	36.0 11.8 11.0 11.0 11.0 11.0 11.0 11.0 11	62.1 47.2 29.5
Mach No.	2.50 2.20 2.20 2.20 2.20 2.20 2.20 2.20	2.10 2.50 2.90
Material	Tungsten (8)	Tungsten (11) ZrB ₂ Coated EH-9 EH-10 EH-11

* D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 6

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Material	Mach No.	atm)	i (BTU) Ib	D (in.)	9cw (BTU) ft sec	(OR)	Ratio T(CA.LC)/T(OBS) Cold Wall I'ay-Ridd Heat Transfer Ccefficient	C)/T(OBS) Fay-Riddell Cc efficient
ZrB ₂ (<u>8)</u> on T ^u ngsten	2.10 2.60 3.00	70.4 47.7 29.8	1980 2190 2110	0.600	3900 3570 2740	4710 5000 3736	1.26 1.23 1.55	1.26 1.23 1.55
Tungsten (11) 11.4 w/o Ag								
EH-12	2.4	52,1	1540	0.660	2485	4530	1.10	1.10
Tungsten (<u>8</u>) Silver	2.50	53.1	1540	0,600	2500	3800	1.31	1.32
Tungsten (11) Durac Coated								
EH-7 EH-8	2.10	61.2 43.5	1979 2131	0.600	3500 3340	5500 5000	i.07 I.21	1.07
Durac (8) on Tungsten	2.20	68.7 44.4	1980 2130	0.600	3860 3360	4500 5200	1,32	1.32
Tungsten (i1) Sn-Al infilt.								
EH-24	2.40	50.8	1450	0.300	3240	2600	0.87	0.87

^{*} D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 7

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Material	Mach No.	9	i (BTU)	D (in.)	qcw (BTU) ft sec	$\frac{1}{(obs)}$	Ratio T(CALC:)/T(OBS) Cold Wall Fay-Riddell Heat Transfer Coefficient)/T(OBS) Fay-Riddell Coefficient
Tungsten $(\underline{8})$ Sn-A I	2.40		1450	0.300	3360	4500	1.08	1.08
Tungsten (10) 30% Zn z = 0.40	0.5	I. 0	3900	0,750	860	5700	1.12	1.07
Tungsten (8)		61.9	1900	0.600	3420	4800	1.18	1.18
Alloys	2.20	67.4	1990	0.600	3750	4040	1.47	1,47
•		64.0	1920	0.600	3520	4200	1,38	1.38
	•	62.5	1940	0.600	3720	4370	1.35	1,33
		32.0	2185	0.600	2940	4200	1,43	1,43
		45.1	2060	0.600	3240	3870	1.53	1.53
	•	65.0	2150	0.600	4100	3780	1.65	1.64
		30.0	1920	0.600	2430	4040	1.38	1.38
		65.0	1900	0.600	3500	4800	1.20	1,20
		0.69	2040	0.600	3890	4380	1.38	1.38
		65.0	2000	0.600	3700	5650	1.05	1,05
		0.99	2070	0.600	3880	4290	1.45	1.47
		63.0	1950	0.600	3520	6220	0.94	0,94

* D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 8

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Ratio T(CALC)/T(OBS) Cold Wall Fay-Riddell Heat Transier Coefficient													1.28					
Ratio Cold W Heat		1.5	0	1, (1.0	Ξ.	0) • <u>1</u>	T.	1.	1.0	1. (1.29	.=)	-	1.	-	
$\frac{\mathbf{T}}{(\mathbf{obs})}$		5050	6250	4750	4809	2000	2600	5650	4700	5350	4750	4880	4630	4250	4000	4000	4400	4600
$\frac{q_{cw}}{(BTU)}$		4070	3510	2410	3210	2930	3520	4040	3560	2760	3100	2910	3720	3550	2440	3580	3630	4090
D (in.)		0.600	0.600	0.600	009.0	0.600	0.300	0.300	0.300	0.600	0.300	0.300	0.600	0.300	0.600	0.300	6.300	0.300
i (BTU) lb		2157	396I	1481	202	2184	1519	1886	1963	1783	1563	1712	2065	1520	1480	1960	1850	1890
Pe (atm)		64.4	55.8	53.6	44.3	31.9	53.8	43.0	30.6	45.1	41.2	28.3	65.0	54.6	54.7	31.1	39.2	44.0
Mach No.				2.30								2.90		-	-	3.00	-	
Material	Tungsten (11) 10 w/o Cu	EH-13	EH-14	EH-15	EH-16	EH-17	EH-18	EH-19	EH-20	EH-21	EH-22	EH-23	Tungsten (8)	Copper	l I			

[‡] D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 9

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Ratio T(CALC)/T(OBS) Cold Wall Fay-Riddell Heat Transfer Coefficien			1.15	1.13	1.12	1.12	1.12	1.14	1.13	1.10	1.13	1.13		21.1	1.12	1.11	1.14	1, 12	I. 12	1.11	1.13	1.15	1.14	1.14	1.14	1.18	1.13	1.14	1.11	1.14	•
T (S)	(op8		623(6490	647(587(542(632(653(655(265	5310	7647	-	643(589(530(921(6530	652(597(515(635(638(582	512(9259	644(649	290(000
qcw (BTU)	ft sec		1360	1450	0191	1230	816	1250	1410	1480	1230	986	0371	1420	1610	1230	918	1250	1410	1480	1230	098	1450	1610	1230	816	1250	1410	1480	1230	
Œ.			6.750	0.759	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	*	***	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	0.750	H
i (BTU)	13		8240	7870	0 5 99	4900	3980	6800	8090	6470	4940	3890	6644	6211	6230	4940	3960	0998	7670	6340	4900	3900	7560	6420	491C	3910	8580	7560	6510	4970	
P e (atm)			1.07	1.18	1.36	-: 46	1.40	1.07	1,17	1.37	1.48	1.40		7.10	1.35	1.46	1.38	1.06	I. 17	1.37	1.48	1,38	1.16	1.37	1.48	1.37	1.07	1.17	1.37	1.49	
Mach No.			0.33	0.58	0.72	0.82	0.76	0.33	0.51	0.73	0.82	0.76	c c	2:.5	0.71	0.81	0.74	0.31	0.51	0.73	0.82	0.74	0.50	0.73	0.83	0.73	0.33	0.51	0.73	0.83	
Materia]		Graphite (10)	€ = 0.70		0.65	0.66	0.63	0.65	0.62	0.61	0.59	0.61	č	9.60	0.64	0.65	0.64	0.68	0.61	09.0	0.57	0.65	99.0	0.62	0.64	0.64	0.65	0.63	0.61	0.60	

 * D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 10

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Pe (atm)
0.008 11,880
056 056
056 11,880
010 13,560
010 13,560
010 13,560
008 11,880
800
008 11,880
800
800
008 11,880
008 11,880
007 5,420
5,420
5,420
007 5,420 1.000 _{**}
007 5,420
16,610
7

* D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 11

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

<u>Material</u>	Mach No.	Pe (atm)	i (BTU) 1b	D (in.)	q _{cw} (BTU)	(OR)	Ratio T(CALC),'T(OBS) Cold Wall Fay-Ridd Heat Transfer Coefficient	fay-Riddell Fay-Riddell Coefficient
RVA (<u>13</u>)	8.8.7	0.007	5440 13000	0.500* 0.500*	67 104	2690 2715	1.38 1.57	1. 42 1. 81
(Estimated Mach No.) =	8.9 6.2 7	1.000	16600 5400	0.500*	2240 320	5800 3190 4530	1.51	1.58 1.49
RVA (13) Purified		0.007 0.008 0.051	5400 13000 13060	0.500* 0.500* 0.250*	67 104 495	2420 2535 4750	1.54 1.68 1.29	1.58 1.94 1.38
(Estimated Mach No.) = PT-0178 (13)	8.8	1.000	16600 5400	0.500* 1.000	2240 49	5915 2610	1.48	1.54 1.35
Mach No.) = (Estimated	0 0	1.000	16600	0.500*	2240	6340	1.38	1.44
	8 8 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	0.028 0.051 0.007 0.008	5400 13000 5400 13000	0.250 0.250 0.500 0.500	320 495 67 104	3060 4235 2750 2785	1.69 1.41 1.35 1.53	1.56 1.52 1.39 1.77
PT-0181 (13)	8.5	0.007	5400 5400 13000	1.000* 0.500* 0.500*	49 67 104	2645 2640 2875	1.31 1.41 1.48	1.33 1.44 1.71
(Estimated Mach No.) =	8.0.8	0.051	13000 16600	0.250	495 2240	4200 6620	1.45	1.56

*D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 12

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Ratio T(CALC)/::(OBS) Cold Wall Fisy-Riddell Heat Transfer Coefficient	1.31 1.39 1.74 1.49	1.36 1.52 1.65 1.65	1.69	1.14 1.27 1.23 1.21
Ratio T(CA) Cold Wall Heat Transfe	1, 29 1, 35 1, 51 1, 39	1.30 1.35 1.35 1.75 1.75	1.60	1.18 1.22 1.29 1.20
(OR)	2690 2740 2825 4395	6735 4320 4320 2880 2880 4320	5400 6390 6230	2560 3010 3760
q _{cw} (BTU)	49 67 104 495	2240 400 400 280 280 400	2240 1460 905	1280 20 51 51 99
D (in.)	1.000* 0.500* 0.500* 0.200*	0.500 0.250* 0.250* 0.250* 0.250*	0.500 0.750* 0.750*	3.000*
i (BTU) lb	5400 5400 13000 13000	16600 13000 13000 5400 5400	16600 6550 4061	10363 10363 8208 8203 8180
Pe (atm)	0.007 0.007 0.008 0.008	1.000 0.051 0.051 0.028 0.028	1.000	1.00 1.00 0.006 0.006
Mach No.	ໝູ ໝູ ໝູ ນຸ ນ ນ ທ ທ	8 R. R. 2 2 R.	8.0 0.76 0.80	0.59
Material	PT-0182 (13)	(Estimated Mach No.) = JTA (13)	(Estimated Mach No.) = 70% Dense Tungsten (10)	 e = 0,4i Hf-20Ta Coating on Ta-10W (14)

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*D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 13

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

Material	Mach No.	Pe (atm)	ie (BTU) lb	(in.)	qcw (BTU) ft ² sec	(OR) (obs)	Ratio T(CALC)/T(OBis) Cold Wall Fay-Rid Heat Transfer Coefficie	C)/T(OBis) Fay-Riddell r Coefficient
W-3 Coated 3.00 TZM Alloy (15, 10) 3.00	3.00	0.005	13900	****	80 140	3000	1.44	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
e = 0.70	9.6	0.008	15600	* *	45	3000	1.49	
(one inch diameter arc	900	0.007	17100	***	130	3080	1.42	1.49
two inch	9.00	0.009	15800	* * * 00 .	130	3070	1.42	1.51
square foil;	3.00	0.008	16800	* 00.1	126 148	3070	1.46	1.43
	3.00	0.007	17100	1.00	136	3070	1.44	1.50
Cr-Ti-Si	3.00	900.0	13800	1.00*	138	3000	1.46	1.42
Coated Cb752	3.00	0.005	12700	1.00	112	3020	1.38	1.35
Alloy (15, 10)	3.00	0.008	13100	1.00	140	3060	1.44	1.42
= 0° 10	3.00	0.008	12400	1.00*	124	3090	1.38	1.33
(exposure same	3.00	0.008	12900	1.00	140	3060	1.44	1.42
as above)	3.00	900.0	13000	1.00	124	3090	1.38	1,35
JTA (16)	3.00	0.016	12000	2.00	175	4280	1.05	1.07
€ = 0.80	3.00	0.012	7180	2.00	91	4830	0.78	0.80
	3.00	0.020	18000	2.00	305	5140	1.01	1.03
	3.00	0.015	11080	2.00	170	4460	1.00	1.00
	3.00	0.015	9770	2.00	153	4430 ₁	0.98	0.9"
	3.00	0.020	18060	2.00	262	5180	0.99	1.0
	3.00	0.035	2900	2.00	139	3110^{\intercal}	1,33	1.33
	3.00	0.042	13450	2.00	336	3810	1.38	1.30
	3.00	0.046	11100	2.00	290	3740^{T}	1.35	1.36
*D is the hemispherical	- T	ap diameter	r of the model.	odel. An	asterisk indi		cates a flat face cylinder	r having
a geometricai diameter	diameter o	of D inches	ı.					

TABLE 14

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

1	Ratic T(CALC)/T(DBS) Cold Wall Heat Transfer Coefficient	1.10	1.14 1.05 1.01 1.81	1.48	35	1.41
	Ratic T(CALC)/T(DBS) Cold Wall Fay-Ridde Heat Transfer Coefficient	1, 07 0, 94 0, 94 1, 12	1.03 0.99 1.08	0.91	1.06	1.09
	(OR)	4200 4760 4750 4020	4370 4550 4180	4180	4180	3515
	(BTU)	173 170 170 174	176 175 80	80	80	080
	(in.)	22.2.2.60	2.00 2.00 0.50 r side	0.500 r side	0.503 r side	000.0
	(Bru)	12230 12000 11930 11890	11785 12120 5380 cylinde	4170 0 cylinder si	cylinder	
	Pe (atm)	0.016 0.016 0.016 0.016		1.00		l I
	Mach No.	3.00 3.00 3.00 3.00	3.00 Sub- sonic	Sub- sonic 3.5	3,5	
	Materia1	z = 0.3 foam (16) z = 0.3 fibrous (16) z = 0.3	Ir coated C (17) $\epsilon = 0.3$	JTA (17) $\epsilon = 0.50$ Ir coated (17)	$\epsilon = 0.3$ JTA (17)	

TABLE 15

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SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

//T(OBS) Fay-Riddell Coefficient	1.07	1.25 1.25 1.25 1.33 1.35 1.35	1.35 1.45 1.52 1.52 1.52
Ratio T(CAI.C)/T(OBS) Cold Wall Fay-Ri Heat Transfer Coefficie	1.02 1.05 1.08 1.12 1.16	2.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1	1.28 1.33 1.33 1.44 1.45 56
$\binom{\Gamma}{\binom{OR)}{(obs)}}$	6120 5940 5760 5580 5400	5400 5400 5220 5220 5220 5220 4860 4860	4860 4860 4680 4500 4320 4320 3960
q _{cw} (BTU) ft ² sec	700 700 700 700 700	700 700 700 700 700 700 700 700	700 700 700 700 700 700 700
D (in.)	0.500* 0.500* 0.500* 0.500*	0.000000000000000000000000000000000000	0.500* 0.500* 0.500* 0.500* 0.500*
i (BTU) 1b	6620 6260 6260 6260 6260 6260	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0929 0929 0929 0929 0929 0929
Pe (atm)	1.00	000000000000000000000000000000000000000	1.00 1.00 1.00 1.00 1.00
Mach No.		666666666666666666666666666666666666666	0.90 0.90 0.90 0.90 0.90 0.90
Material	Tho ₂ (18) BPG PG Tho ₃ + CuSo ₄ TiC (HP)	PG HfC-C (AC) TiC-C (AC) HfC-C (AC) TiC-C (AC) ZrC-C (AC) AIJ AIJ BN PT0179	W (forged)(18) JTA SiC (HP) PT0178 W + (Cu-Zn) W + (Cu-Zn) W + (Cu-Zn)

*D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

TABLE 16

SUMMARY OF PUBLISHED FLUX-TEMPERATURE DATA FOR REFRACTORY MATERIALS

LC),'T(OBS) Fay-Riddell T Coefficient	1.23 1.25 1.25 1.25 1.33 1.33 1.41 1.62 1.62
Ratio T(CALC)/T(OBS) Cold Wall Fay-Ridd Heat Transfer Coefficient	1.65 1.31 1.34 1.38 1.43 1.47 1.52 1.52 1.55
(OR)	3780 6480 6300 6300 6120 5940 5760 5760 5580 5460
$\frac{q_{cw}}{(BTU)}$	700 2400 2400 2400 2400 2400 2400 2400 2
D (in.)	0.500* 0.400* 0.400* 0.400* 0.400* 0.400* 0.400* 0.400* 0.400* 0.400* 0.400* 0.400*
i (BTU) 1b	6260 10500 10500 10500 10500 10500 10500 10500 10500 10500 10500
P e (atm)	1.00 1.00 1.00 1.00 1.00 1.00 1.00 1.00
Mach No.	0.90 0.90 0.90 0.90 0.90 0.90 0.90 0.90
Material	W + (Sn-AI) BPG HfC (HP) ATJ TaC-C (HP) JTA HfC-C (AC) W + (Sn-AI) W + (ThO.) HfC-C (AC) W (forged) BN W (forged) W (forged) W (forged)

*D is the hemispherical cap diameter of the model. An asterisk indicates a flat face cylinder having a geometrical diameter of D inches.

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TABLE 17
TEMPERATURE AND PRESSURE DEPENDENCE OF THE ENTHALPY OF AIR (BTU/LB)*

т			Log P		
o _R	+2	+1	<u> </u>	-1	2_
2700	704	704	704	704	704
	707	707	709	709	709
	(644)	(625)	(605)	(585)	(566)
	(430)	(462)	(494)	(526)	(558)
3600	968	968	968	971	980
	980	980	981	983	991
	(917)	(937)	(956)	(975)	(995)
	(76 4)	(821)	(8 79)	(936)	(993)
4500	1238	1244	1263	1318	1485
	1274	1279	1295	1346	1497
	(1220)	(1300)	(1382)	(1462)	(1543)
	(1194)	(1284)	(1373)	(1462)	(1551)
5400	1530	1574	1707	2055	2641
	1596	1634	1750	2063	2630
١.	(1551)	(1716)	(1881)	(2046)	(2211)
	(1720)	(1848)	(1977)	(2106)	(223 4)
6300	1867	2053	2485	3060	3293
	1967	2115	2489	3047	3341
	(1911)	(2183)	(2456)	(2727)	(3000)
	(2341)	(2516)	(2691)	(2866)	(3041)
7200	2321	2746	3329	3596	3719
	2416	2759	3301	3631	3789
	(2301)	(2703)	(3104)	(3506)	(3908)
	(3058)	(3286)	(3515)	(3743)	(3972)
8100	2890	3495	3879	4073	4523
	2943	3447	3877	4131	4585
	(2720)	(3274)	(3828)	(4382)	(4936)
	(3870)	(4159)	(4448)	(4738)	(5027)
9000	3536	4086	4382	4913	6468
	3515	4037	4412	4964	6469
	(3168)	(3897)	(4626)	(5354)	(6084)
	(4778)	(5135)	(5492)	(5847)	(6206)

^{*}The sequence of values is, Reference (21) Reference (10) (Equation (4)) and (Equation (5)).

TABLE 18

OBSERVED TEMPERATURES DERIVED FROM HOT GAS/COLD WALL ARC PLASMA TESTS AVERAGED VALUES OF TOTAL NORMAL EMITTANCE AND RATIOS OF CALCULATED AND

		<u>.</u>		Calculated Temperature Ratio	re Ratio
	Comm	Commited Norms Emittence	6 5 5 1	T(CALC)/T (OBS) - Co	old Wall Heat
Material/Code		(Solid)	(Melting)	(Solid)	::ent (Melting)
HfB _{2,1}	A-2	0.45	0.39	1.16	0.98
ZrB2	A-3	0.47 (0.500")	0.39 (0.500")	1.09 (0.500")	1,08 (0,500")
ZrB2	A-3	0.57 (0.750")	(0.750")	1.12 (0.750")	(0.750")
$HfB_2 + 20 \text{ e/}\circ \text{SiC}$	A-4	0.62	0.48	1.22	1.07
Boride Z	A-5	0.75		1.20	
$HfB_{2.1} + 20 \text{ v/o SiC}$	A-7	0.55	0.47	1.25	0.99
$Z_{rB_{2,1}} + 20 \text{ v/o SiC}$	A-8	0.59	0.50	1.34	1.02
HfB _{2.1} + 35 v/o SiC	A-9	0.55	0.52	1,17	0.99
ZrB2 + 14 v/o SiC + 30 v/o C	A-10	0.62	0.55	1.20	1.11
RVA	B-5	0.52	0.75	1.17	1,20
PG	В-6	(0.500 inch) 0.41 [to C	(0.740 inch) 0.41 (1 to C	(0.500 inch) 1.19 (to C	(0.740 inch)
BPG	B-7	0,37 to C	0.4311 to C	1,18 to C	1.03 H to C
Si/RVC	B-8	0.69 Coated	0.56 Bare	1.36 Coated	1.07 Bare

TABLE 18 (Cont'd.)

AVERAGED VALUES OF TOTAL NORMAL EMITTANCE AND RATIOS OF CALCULATED AND OBSERVED TEMPERATURES DERIVED FROM HOT GAS/COLD WALL ARC PLASMA TESTS

B-9 B-10 C-11 C-11 F-14 F-15 G-19 G-19 G-22 H-22 H-23
B-9 B-10 B-11 CC-11 CC-12 D-13 F-14 F-15 F-15 I-23 I-23

ij

		STONE STONE		0.840	0.724	82.1 1.18	1.063 0.807	.667	- 43 - 42 - 42 - 42	Ŧ	114	307	.383	.003	. 229	.962	196	673	781	94.5	1.024	260	043	14.	112	929	32	6 05	1833	771	732 762	345	22	. Z	134	6 5 1	131	2.5		1.422 1.036 0.861	
ક	S(CALC) Celculated	Receion	(mile/sec)		200		0,939																			0.123							717					267 0.6		522	
(FLAT FACED CYL NDERS)	Eff scrive Ra lius	-	. :	30	200	00	900	. 02.60	02 50	02 50	24.50	93.55	10.	***	5 N	9 9	6 0 6 2 0 2 0 2 0 3	05.0	05:0	0520	0250	5365	0.10							_			_				-	5.04E: 2	2.0434	0.0464 0.0458 0.0450	
LAT FACE	Diameter	Inicial/Final	(er ne)	200/200	50c/300	256/250	256/250	250/250	500/200	500/500	495/163	495/246	495/204	495 321	695,363	1000,1000	250, 250	250, 250	500, 500	500/500	500/300 0.	1000/200	0001/2001.					-				_	_		_		→	501/321	501/337	201/301 201/34: 202/36:	
		ž.		_			_			-•	0.28	0.29	6.33	18.5	22.	8.		-			-	. 6	-			_											-	0.30 36.00	. 32 32 32 32 32 32 32 32 32 32 32 32 32 3	0.45 3.31 3.31 3.31 3.31	
D ATJ GRA				0.007 115	0.008 115	0.051	0.051 115	0.051	900	900		000	070	.023	213 81	2007					0.008	=				0.010						.010 .010 .010						* # # # # # # # # # # # # # # # # # # #		****	
CO(B10) AN	4	r- ,#	2290	2880	2300 2540		5000 4720	4780 5580	98 98 98 98 98 98			0829	4250	94680 5180	5500 6355 0	3080	3040	2560	200	2772		_				_						00	3033 6.0	3159 0.0	3042	3186 0.0	3342 0.0	4820 1.060 6040 1.080	6260	4800 1.064 4980 0.02	
)178(B9), PC	S(OBS) Observed le Recession		P1 0.077	22.0	o 0	 	00	0 4	44	. 4 "			ų e	90	2 2 2 4	1.070	10 1.080	11 0.130	13 0.160	15 0.220 6.230																		3,130			
VA(BS), PT	Sample Co.	•																		B9-CE15						MED311												B10-2M		B10-6M	
TES FOR R	Calculated Recession SIOBSI Rate SICALCI	mils/sec)	2.553 1.02			0.00						271 [.148 459 [.685																										1.253			
ED RECESSION RATES FOR RVA(BS), PT0178(BS), POCO(B10) AND ATJ GRAPHITES FOR AND ATJ GRAPHITES	Radius Calc	Œ	0.0375 2.9			395 0.5	- 0	0-			6 6		ni 14	520 0.13	520 0.15	9.0	200							6.29	0.25	22.0	0.25	0.23	0.20	0.225	0.228	0.197	0.233	6.23	0.20	0.216		0.391		1 to 62 mile.	
SERVED RE	Diameter Ru Initial/Final		500/221 0.		-			-0													50 0.0260		63 0.006 															39 0.0467 84 0.0461	•	edunes edun to	
90 TWY 07	Mach Diar No. Initia		0.33				3.20						; Z;		2000	200	2009	7008	250/2	2,052	250/250	2/062	63/63 														•	500/399	306/3	A	
	Dennity	_	22:	2	123	22:	22:	12	711	211 211	211	112	2:	111	923	111	112	115						25	101 101	266	101	101	101	202	201	201	201	6 6	662	200		114 3.20	le culinder c		
	. "		4500 1.050				470	1890 0.299	630	995 1.070 010 0.012	500 0.024 855 0.218	300 1.000	000 1.000	410 0.007	750 0.007	560	90000 90000 90000	30 0.008	20 0.028	20 0.028 20 0.028	4320 0.051 4650 0.051					7. 2. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0. 0.												0.034	30° half and		
S(OBS) Observed		(mils/sec)									1.460															6.321 2871 0.334 2844							0.241 271 0.262 265					2.120 581	ch diameter-		
	Sample No.	B5.24	B5-3M	35-6M	B5-1R B5-2R	85-48	85-58 85-68	85-72 85-1124	B5-12H	B5-14R	B5-15K	2 2 2 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3		85-GE1	BS-CE3	BS-GES	B5-0E7	BS-0E9	B5-GE14	35-CE16						MEDS17												B10-9R 2	3		

COMPARISON OF CALCULATED AND OBSERVED RECESSION RATES FOR RYA(BS), PTO176(B9), POCO(B10) AND ATS GRAPHITES (TLAT FACED CYLNDERS)

	30 30 30		333	1.349			È		. 512 . 512	3	2	9 X	3:	i.	3.5	33	3	22.		C. 894	27.7	1.152	2		1 i	1 064 0 671	1 076	Ĭ	32 00	22	25	37		35	8	202	36.7	13	58	23
Calculated	Recession Rate		0.872	0.0	0.0	100	0	3		7	6.963 962	9		2	. 0	 	85	96.0	, 10 10 10 10	7.03 1.03 1.03 1.03 1.03 1.03 1.03 1.03 1		1	103	0.0	9 6 0 0 0	0, 10¢	500	5.5	 28		0.0	3.7. 3.7.	16.	2£3	32	21	23	2		100
Lifective			•.0 <u>w</u>												-	0.104	_	_	_									_		_	_,	. 0. 0.							 .	. •
	Diameter Initial/Firel	(mg)s)	052/05 2		_							_		_	_	1000/1600															-	250/250	_							
	Ke h		§—									_			_	9	_		_			_			_						_	. —								-
	Denotity	1ba/85 ³)	555	55	255	252	55	5	25!	55	201	22	25	6	5	<u> </u>	5	56	55	66 61	601	5	151	5	25	101	22	22	66	15	555	<u> </u>	55		52	5 5	66	55	25	22
	۱٠,	_	0.036	0.036	250		200		100	30.0	0.0	980.0	***	9	0.056	0.036 0.00T	0.00	200	9.00	9.00	0.00	90.0	500	8	90.0	0.001		90	8 8	88	888	38	3	888	88	88	9.00	88	88	88
	н		3870 3915 3915	4784			726	Ę	Ę	2	1933	200	316	9	ž	90 8 4	378	212	2325	2413	224	2142	Š	12	502	232	222	22		2		3	3		33	252	2529	5 % 5 %	Š	32
MOBS)	Recession	(mils/sec)	1.20	12.	1.270	152	1.332	273	. 450	4	750		25	8	2	320 0, 946	0.069	0.0	9.0	0.110	0.063	76.		60.0	0.0	0.114 0.070	2 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	0.098		18	200	, ,	77.0		7.77	0.240 0.240	0.344 0.345	27	15.	0.200
	Mo. No.		MED496 MED501 MED495	MEDIA	MEDA!	NED S	MEDSO!	KEDA	KEDGE	NED S	KEDIN	MED 69	MED501	V.ED	MEDAN	MEDIT	MED639	MEDICA	KERGI	MED642	MED633	MED639	7	MED S	KED64	KEDES	KED!X	KEDES		T T		3				MED459	KEDI		Į.	KED97
	SCALC:		355			1,310	 	1, 527	577	25.5	22.	2	. 55 52.	1,361	1.316	2		310	1.337	2	3	12	25. 25. 25.	1,015 1,263	53	33	12:		17.1	1.276	1.25	0,786	0. 816 0. 736	0. 552 0. 716	 	1.179	30.0	- -		
S(CALC)	Lagrange .	mile/sec)	200	202.0	£\$	0.432		0,367 0,478	0.367	222	0.250	200	0.2% 0.2%	0.22:	97.50	2	0,269	0.255	6.201	0.76	100	0.215	0.1 1 0.214		0.201	200	2	125	50.0	0.10	6.113 6.112	0.182	0.097	5.3	. 0.0 0.0	85.5		6.19		
Lifective		4							• 00°				_				_	-	9.00	_				_										→ 50 · 0		_	_	-•		
	ᆲ		0001/0001						- <u>.</u>		_			_	_		_	•	3-						_		5	2						 8	_			-		
		E	<u> </u>						→ §		_	_		_			-	•	Š.	_							- 3/401	}						→ /0001	—			-		
	1		ë											_	_			-	8		_										_			-8				→		
	Destro	(Jbe/ft	255	959	225	55	55	22	66	25	5	5	101	9 6	66	2	26	<u>5</u> 5	5!	10	55	101	55	5 6	50	5	255	222	5	55	55	5 5	25	6 6	55	60	25	25		
	٠,	(mtm)		900	ij	4 4	13	 11	0.0 4.0 4.0	88	9.0	8	. o	000	900	0.0		88	0.00	90.0	000	88	\$ 8 8 8	9 6	000	8	388	28	2 5	000	88	8 8	6.6	0.00	88	0.00	99	88		
•			222	2 2 2	33	22	53	22	25.5	77	27.0	2	3	2673	31	2		22	260	2		5 ¥	4 22 22 22 23	77	2601	252		2		22.5	355	225	777	72	222	229				
Open v	Recession		7.5.5	944	11	3 3		573	0.619	200	1	Z.	0.30	22	0.317	2	ia 	0. NS 0. NS	0.270		0.25	0.20 204 204	0.212	0.216	0.217	0.27	2		0.123	•	6,139	0.0	0.0	0.052	1000	0.08	6.0	9.15		
	No.		K K K K K K K K K K K K K K K K K K K	KEOSS	MED330	MEDIA	MEDSAL	MEDS3:	MED530	MEDS20	100		KERS!	KED913	MED921	KEDSI	X LOS	KED919	MEDSIS	MEDS16	MEDS:	MEDS 15 MEDS 16	MEDS!3	MEDS14	MEDS21	KED82			KEDESS	KEDES	KEDAG	MEDAN	NED SE	MED433		KERE		X X		

TABLE 19 (CONT)

COMPARISON OF CALCULATED AND OBSERVED RECESSION RATES FOR RVA(B-5), PT0178(B-9), POCO(B-10) AND ATJ GRAPHITES (FLAT FACE CYLINDERS)

Sample No.	S(OBS) Observe Recession Rate	edì	P _e (atm)	Density (lbs/ft ³)	Mach No.	Diameter Initial/Final (mils)	Radius R B	S(CALC) Calculated Recession Rate	S(OBS) S(CALC)
B5-23M B5-24M B5-25M B5-26M B5-31M B5-31M B5-32M B5-16R B5-28R B5-29R B5-29R	0.878 0.992 0.522 0.269 0.430 0.538 0.422 1.463 0.018 0.086 0.244	3725 4105 3420 3035 2995 3285 3475 5855 2165 2780 3465	1.000 1.000 1.000 1.000 1.000 1.000 0.218 0.005 0.008 0.011	112 112 112 112 112 112 112 112 112 112	0.10 0.13 0.15 0.15 0.10 0.10 3.20 3.20 3.20 3.20	502/350 502/330 503/400 503/380 502/340 501/350 502/405 739/620 503/500 504/490 502/470	ft. 0.0443 0.0443 0.0449 0.0460 0.0438 0.0442 0.0707 0.0522 0.0517 0.0505	1.525 1.420 1.192 1.175 1.174	0.689 0.650 0.368 0.226 0.366 0.458 0.352 1.156 0.243 0.457 0.819

TABLE 20

CALCULATION OF FLUX-ENTHALPY CONDITIONS CORRESPONDING
TO THE TRAJECTORY FOR THE FDL-7MC HYPERSONIC RE-ENTRY

VEHICLE UNDER MAXIMUM CROSS RANGE CONDITIONS

 $(R_B = 3^{\circ})$

Time	Altitude	Velocity	i _e	a	Pe
(sec)	(kft)	kft/sec	BTU/lb	BTU/ft2sec	(atm)
200 400 600 800 1000 1200 1400 1600 1800 2000 2400 2600 2800	260 200 J 90 180 180 180 170 160 150 140 130 120 110	26 25 23 22 20 18 16 14 13 12 10 8 7	13500 12500 10600 9700 8000 6500 5100 3900 3400 2900 2000 1300 1000 500	210 480 450 480 360 260 210 170 170 160 110 70 60	0.03 0.21 0.25 0.34 0.28 0.23 0.24 0.28 0.36 0.43 0.45 0.42
3000 3200	80 60	3 2	200 100	10	0.38 0.30 0.31

 $W/S = 53 lbs/ft^2$ $W/C_L S = 312 lbs/ft^2$

TABLE 21
SUMMARY OF DATA USED IN THEORETICAL CALCULATIONS OF
RECESSION RATES AS A FUNCTION OF FLUX-ENTHALPY CONDITIONS

Material Code	T °R*	<u> </u>	ΔΗ _f *** (BTU/1b)	Density (lb/ft³)	(gms) (ΔS_{f}^{\pm}
HfB _{2.1} (A-2)	6570	0.40	490	625	66.7	5.0
ZrB ₂ (A-3)	6335	0.37	840	350	37.6	5.0
HfB ₂ +20v/oSiC						
(A-4)	5700	0.48	495	585	57.5	5.0
Boride Z(A-5)	5300	0.55	705	355	37.5	5.0
$HfB_{2,1}(A-6)$	6570	0.50	490	665	66.7	5.0
HfB ₂ +20v/oSiC						
(A-7)	5700	0.48	480	565	59.0	5.0
ZrB ₂ +20v/oSiC						
(A-8)	5300	0.55	740	340	35.8	5.0
HfB ₂ +35v/oSiC						
(A-9)	5500	0.59	560	485	49.3	5.0
ZrB ₂ +SiC+C						
(A-10)	5500	0.62	1150	280	28.7	6.0
HfC+C(C-11)	6210	0.55	730	565	59.5	7.0
ZrC+C(C-12)	5725	0.45	1050	340	38.0	7.0
JTA(D-13)	5000	0.57	2000	190	17.5	7.0
JT0992(F-15)	5800	0.60	1400	290	28.8	7.0
JT0981(F-16)	5170	0.51	1880	195	19.2	7.0
$WSi_2/W(G-18)$	6570	0.28	70	1200	184.0	2.0
$Sn-A\ell/Ta-W(G-19)$	5890	0.46	65	1055	181.3	2.0
Hf-Ta-Mo(I-23)	4320	0.54	50	840	179.0	2.0
Ir/C(I-24)	4950	0.30	50	1400	192.0	2.0

^{*} Estimated values.

^{**} Calculated value, $\Delta H_f = T \Delta S_f$.

TABLE 22

CALCULATION OF TEMPERATURE GRADIENTS THROUGH

ARC PLASMA TEST SAMPLE ZrB₂(A-3)-2MC

		input Values	_	Obs	served Values
q	=	365 BTU/ft ² sec	$\mathbf{T}_{\boldsymbol{\epsilon}}$	=	4930°R
i_e	=	3230 BTU/1b	T		3400° R at x = 101 mils
P_e	=	1,060 atm			
€ S	=	€ = 0.47			,
R	=	0.020 ft. (246 mils)			
L	×	0.035 ft. (429 mils)			
I	2	0.0011 ft. (14 mils)			
$\mathbf{k}_{\mathbf{F}}$	=	0.0001 BTU/ft sec OR for oxide 0.0120 BTU/ft sec OR for boride			
k _s	=	0.0120 BTU/ft sec R for boride			

x mils	Computed Temperature	x mils	Computed Temperature OR
	20	*******	R
0.0	4633	231	3325
21.0	3424	273	3314
42.0	3411	315	3306
63.0	3398	357	3300
105.0	3376	399	3297
126.0	3366	420	3297
168.0	3347		

TABLE 23

COMPARISON OF OBSERVED AND CALCULATED INTERNAL TEMPERATURE RESULTS

Test	$egin{aligned} egin{aligned} egin{aligned\\ egin{aligned} egi$	es 10 ³ ks 10 ³ k _F R/L/I	$T_{ m f}$ (CALC) $T_{ m f}$ (OBS)	T_{f} (CALC) T(d) (CALC) Tf (OBS) T_{f} (CALC)	T(d (OBS))
	(^o R) (^o R)	(BTU/ft sec OR) (mils)			
ZrB2 +	$ZrB_2 + SiC(A-8)$				
-25M	999	0.59 10.0 0.5 213/696/8	/8 1.12	0.89	0.93
-26M	<u>ہ</u> ر	$B10/M^{2}$ sec, $I_{c} = 5280 B10/M^{2}$, $F_{c} = 1.91 Am$ 33660(395) 0.59 10.0 5 218/696/6	/6 1.12	0.81	0.87
-27R		0.59 10.0 0.5 213/696/7	77 1.27	0.88	0.92
-28R	8TU/Ht-se (680(400)	BTU/III sec, $1_6 = (9/0)$ BTU/III, $P_6 = 0.11/3$ and $2680(400) = 0.59 = 10.0 = 0.5 = 213/696/8$ THI $(4.0.5) = -2.50$ BTII/II, $P_6 = 0.11/3$ and $P_6 = 0.11/3$	/8 1.04	0.78	0.64
-29M	3260(96)	c, 1e = (350 D10/10), Fe = 0.111 cum 0.59 10.0 0.5 213/686/8	/8 1.07	0.89	0.83
-30MS	2570(395)	1e = 3330 DIO/10, $1e = 1.00 d0.59 10.0 0.5 214/688$	/6 1.30	0.82	0.85
-31RS	2820(95)	1.e = 4040 DIU/ID, Fe = 1.01 B 0.59	/5 1.35	06.00	.6.0
-32RS	q = 440 B I U/II-8ec, 3260 2720(399) q = 437 B I U/ft ² sec,	$B1U/II^{-8eC}$, $I_{e} = 1390 B1U/ID$, $F_{e} = 0.220 aun$ 2720(399) 0.59 10.0 0.5 214/688/5 $BTU/ft^{2}sec$, $I_{e} = 7270 BTU/Ib$, $P_{e} = 0.228 atm$	aum /5 1.29 atm	0.80	0.83
ZrB2(A-3)					
-2MC	7	3400(101) 0.47 12.0 0.1 246/422/14	/14 0.94	0.73	0.69
-3MC	q = 303 BIO/M-3eC, 5170 3760(102) q = 460 BTU/ft ² sec,	BTU/ft²sec, ie = 3380 BTU/lb, Pe = 1.06 atm	/31 0.99	0.60	0.73

TABLE 24

$\tilde{T}(d)$ (OBS)			0.91	0.34	06.0	0.95	0.85	0.85	7.1.0	95.0	0.55	0.51	0.83
T(d) (CALC) T_f (CALC)			0.84	0.75	08.0	0.94	08.0	0.94	0.79	0.95	0.86	0.95	0.86
$\mathbf{T_f}$ (CALC) $\mathbf{T_f}$ (OBS)			1.00	1.02	1:34	1,31					1.61	1.44	1.48
⁶ S 10 ³ k _S 10 ³ k _F R/L/I	(BTU/ft sec ^o R) (mils)		0.55 8.0 0.5 219/699/13	3 BTU/ft ² sec, ie = 3500 BTU/lb, Pe = 1,02 atm 3570($\frac{405}{2}$) 0.55 8.0 0.5 219/690/10	sec, $i_e = 3640 \text{ BTU/lb}$, $P_e = 1.02 \text{ atm}$ 0.55 8.0 0.5 219/682/0	7 BTU/ft ² sec, $i_e = 6540$ BTU/lb, $P_e = 0.162$ atm 3060(101) 0.55 8.0 0.5 219/684/0	sec, $i_e = 4360 \text{ BTU/lb}$, $P_e = 1.04 \text{ atm}$ 0.55 8.0 0.5 219/690/0	sec, $i = 4580 \text{ BTU/lb}$, $P_e = 1.04 \text{ atm}$ 0.55 8.0 0.5 $e 219/702/0$	sec, ie = 5750 BTU/lb, Pe = 0.169 atm 0.55 8.0 0.5 219/738/0	sec, $i_e = 6290 \text{ BTU/lb}$, $P_e = 0.169 \text{ atm}$ 0.55 8.0 0.5 488/690/0	ec, $i_e = 7030 \text{ BTU/lb}$, $p_e = 0.145 \text{ atm}$ 0.55 8.0 0.5 $e_488/690/0$	sec, $i_e = 7250 \text{ BTU/Ib}$, $P_e = 0.150 \text{ atm}$ 0.55 8.0 0.5 488/690/0	$q = 512 \text{ BTU}/ft^2\text{sec}$, $i_e = 6800 \text{ BTU}/lb$, $P_e = 0.167 \text{ atm}$ 3180 2640(399) 0.55 8.0 0.5 488/675/0 $q = 497 \text{ BTU}/ft^2\text{sec}$, $i_e = 6510 \text{ BTU}/lb$, $P_e = 0.167 \text{ atm}$
$\mathbf{T_f} \mathbf{T}$ (dmils) observed	(^o R) (^o R)	(A-7)	4370 3980(109)	$q = 513 BTU/ft^{6}s$ 4230 3570(405)	$q = 495 BTU/ft^2s$ 3190 2860(401)	$q = 487 BTU/ft^2 sec, i$ 3220 3060(101) 0	$q = 493 BTU/ft^2s$ 3230 2740(399)	$q = 522 BTU/ft^2sec$, 3730 3160(97)	$q = 503 BTU/ft^2sec$, $3660 2830(400)$	$q = 489 BTU/ft^2 sec, i$ 2970 2840(100) 0	$q = 492 BTU/ft^2s$ 2940 2780(401)	$q = 492 BTU/ft^2 sec, i$ 3310 3010(101) 0	q = 512 BTU/ft ² s 3180 2640(399) q = 497 BTU/ft ² s
Test		HfB2+SiC(A-7)	-36MH	-37MH	-39RH	-44MS		-46RS			-50RH	-49RHS	-51RHS

TABLE 25

T(d) (OBS) T (OBS)	0.84 0.76 0.92 0.83	0.92 0.72 0.67 0.57	0.94
$egin{array}{cccccccccccccccccccccccccccccccccccc$	0.83 0.66 0.93 0.79	0.90 0.76 0.84 0.73	0.94
T (CALC)	0.96 0.95 1.39 1.33	1.02 0.86 0.87	0.90
(BTU/ft sec 0 R) (mils)	$\frac{4670}{4670} 3940(100) 0.55 8.0 0.5 219/689/14$ $q = 495 BTU/ft^2sec. i_e = 4390 BTU/lb, P_e = 1.04 atm$ $5010 3810(397) 0.55 8.0 0.5 219/687/62$ $q = 502 BTU/ft^2sec, i_e = 4400 BTU/lb, P_e = 1.04 atm$ $3110 2860(102) 0.55 8.0 0.5 219/690/0$ $q = 498 BTU/ft^2sec, i_e = 7140 BTU/lb, P_e = 0.138 atm$ $3260 2700(400) 0.55 8.0 0.5 219/690/0$ $q = 503 BTU/ft^2sec, i_e = 7520 BTU/lb, P_e = 0.134 atm$	$+C(A-10)$ 3840 3550(96) 0.62 10.0 0.5 219/690/6 q = 400 BTU/ft^2sec, ie = 3870 BTU/lb, Pe = 1.02 atm 4745 3440(389) 0.62 10.0 0.5 213/692/16 q = 400 BTU/ft^2sec, ie = 3990 BTU/lb, Pe = 1.02 atm 5020 3360(101) 0.62 10.0 0.5 219/690/12 q = 495 BTU/ft^2sec, ie = 6320 BTU/lb, Pe = 0.147 atm 5170 2930(400) 0.62 10.0 0.5 213/690/17 q = 495 BTU/ft^2sec, ie = 6460 BTU/lb, Pe = 0.147 atm	3310 3100(202) 0.52 6.5 251/670/0 q = 135 BTU/ft ² sec, i _e = 2530 BTU/lb, P _e = 1.00 atm 3480 285¢(463) 0.52 6.5 251/671/0 q = 135 BTU/ft ² sec, i _e = 2930 BTU/lb, P _e = 1.00 atm
T T (dmils) observed (OR) (OR)	HfB2 + SiC (A-7) -40M		3310 3100(20) q = 135 BTU/ft 3480 285c(46) q = 135 bTU/ft
Test	HfB2 + -40M -41M -42R -43R	-38M -39M -40R -41R RVA(B-5)	-31M -32M

TABLE 26
COMPARISON OF OBSERVED AND CALCULATED
INTERNAL TEMPERATURE RESULTS

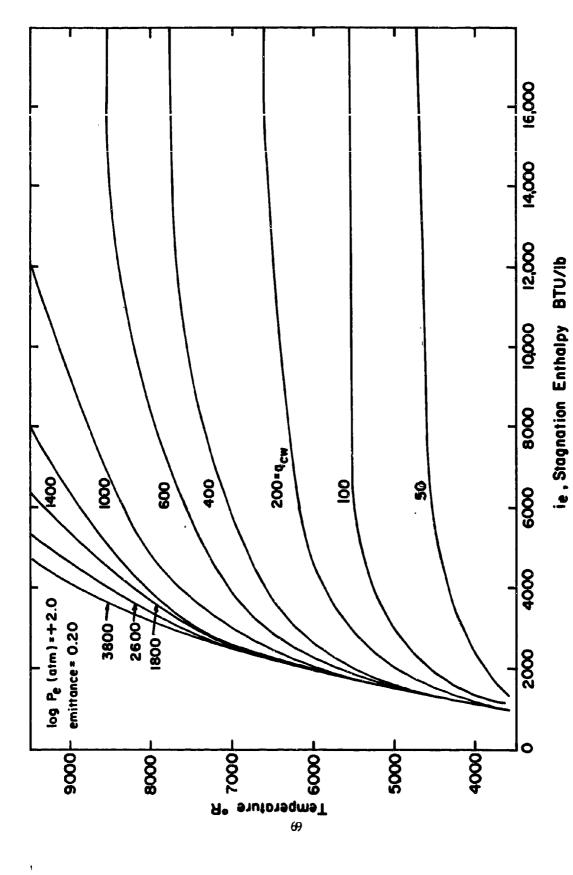
T(4) (OBS) T (OBS)		0.91	0.84	0.87	08.0	0.99	0.85	0.68	0.55	0.97	3.92	99.0	0.68
T(d) (CALC) T _f (CALC)		0.82	0.82	0.89	0.79	0.95	0.85	0.85	0.73	96.0	0.88	0.88	0.85
$\mathbf{T_f}$ (CALC) $\mathbf{T_f}$ (OBS)		1.04	0.98	1.13	1.14	1.29	1.24	0.93	0.87	1.48	1.51	0.98	1.11
T _f T (dmils) $^{\epsilon}_{S}$ $^{10^{3}}k_{S}$ $^{10^{3}}k_{F}$ R/L/I observed $^{(0}R)$ (BTU/ft sec $^{(0}R)$ (mils)	6	1630(102)	$q = 410 \text{ BI U/H}^2\text{sec}$, $t_e = 5950 \text{ BI U/lb}$, $r_e = 1.01 \text{ atm}$ 3960 3320(391) 0.62 10.0 0.5 219/686/5	3280(109) 5 P.T.1/62	2980(393) 0.6 = 7230 D10/1D, Fe 2980(393) 0.65 = 7210 D417/1. D	2880(102)	v	3220(94)	SBTU/II sec, ie = 7400 BTU/Ib, Pe 2870(399) 0.62 10.0 0.5 e BTII/142666 i = 7470 BTII/II. E	3020(108)	2770(404) 0.9 PTH/#2.60	3210(103)	q = 507 BTU/ff*sec, 1 _e = 6010 BTU/lb, P _e = 0.167 atm 4190 2830(392) 0.62 10.0 0.5 488/671/5 q = 522 BTU/ff²sec, i _e = 6010 BTU/lb, P _e = 0.167 atm
Test	ZrB2+Si	~-34MH	-35MH	-36кн	-37RH	-42MS	-43MS	-44RS	-45RS	-46RH	-48RH	-47RHS	-49RHS

TABLE 27

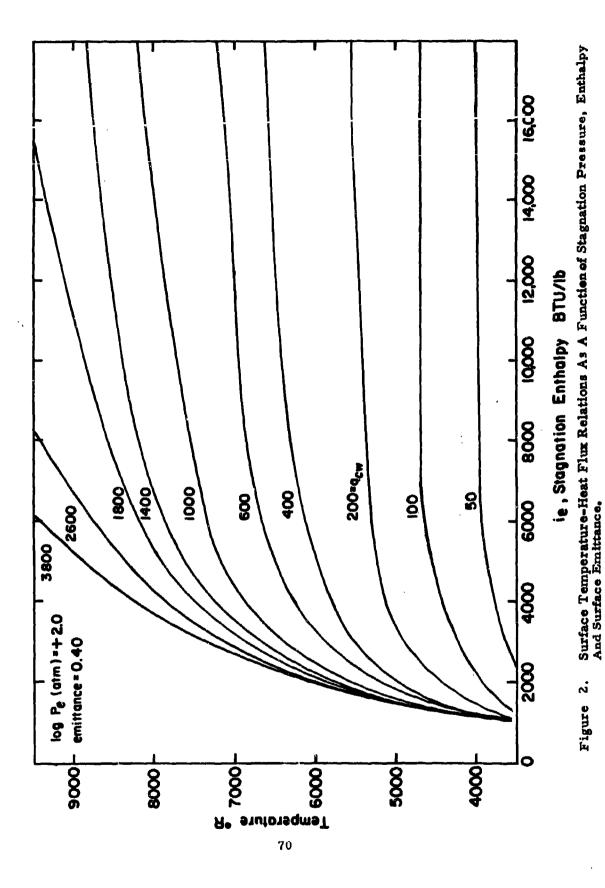
	T. T (dmils)	ć		$\mathbf{T_{f}}$ (CALC)	T(d) (CAL,C) T(d) (OBS)	T(d) (OBS)
Test	observed S	103ks 103kg R/L/I		T (OBS)	T (CALI)	T_{f} (OBS)
1	(^o R) (^o R)	(BIU/ft sec R) (mils)	R) (mils)			
ra-M	Hf-Ta-Mo(1-23)	·				
45MS	3700 3170(99) 0.	.60 7.5 0.	5 196/793/11	1.11	0.85	0.86
37177	$q = 445 BTU/ft^{6}sec, i$	15 BTU/ft4sec, ie = 3700 BTU/lb, $P_e = 1.05 \text{ atm}$	$P_e = 1.05 atm$ $5 = 109/704/13$	7	0.72	0.66
3	a = 470 BTU/42 sec. i.	= 3760 BTU/lb.	P. = 1.05 atm		:))
-47RS	4840 3160(102) 0.	60 7.5 0.	5 = 196/700/37	1.00	0.70	0.65
	$q = 498 BTU/ft^2 sec, i,$	e = 7340 BTU/1b,	Pe = 0, 222 atm			
-48RS	4840 2720(409) 0.	.60 7.5 0.	5 199/700/26	0.98	99.0	0.58
	$q = 498 \text{ BTU/ft}^2 \text{sec}, i_e = 7090 \text{ BTU/lb}, P_e = 0.229 \text{ atm}$	e = 7090 BTU/lb,	Pe = 0.229 atm			
2/W(WSi2/W(G-18)					
.17M	3600 3310(102) 0.	3310(102) 0.58 16.7 2.0 252/455/4	0 252/455/4	1.04	26.0	0.92
	$q = 320 BTU/ft^2 sec, i$	_ = 3150 BTU/1b,	Pe = 1,02 atm			
-18M	3490 3110(200) 0.	58 16.7 2.	0 252/444/4	1.08	96.0	0.89
	q = 316 BTU/ft ² sec, i,	. = 3280 BTU/lb,	$P_e = 1.02 atm$,
19MS	2860 2770(96) 0.	.58 16.7 2.	0 252/444/4	1.31	26.0	0.97
	q = 310 BTU/ftcsec, i,	= 3380 BTU/lb,	$P_e = 1.02 atm$		•	1
-20MS	2760 2620(200) 0.	2620(200) 0.58 16.7 2.0 253/447/4	0 253/447/4	1,35	95.0	0.95
	q = 306 BTU/ft'sec, i	$6 \text{ BTU}/\text{ft}^2 \text{sec}$, $i_e = 3160 \text{ BTU/lb}$, $P_e = 1.02 \text{ atm}$	$P_e = 1.02$ atm			

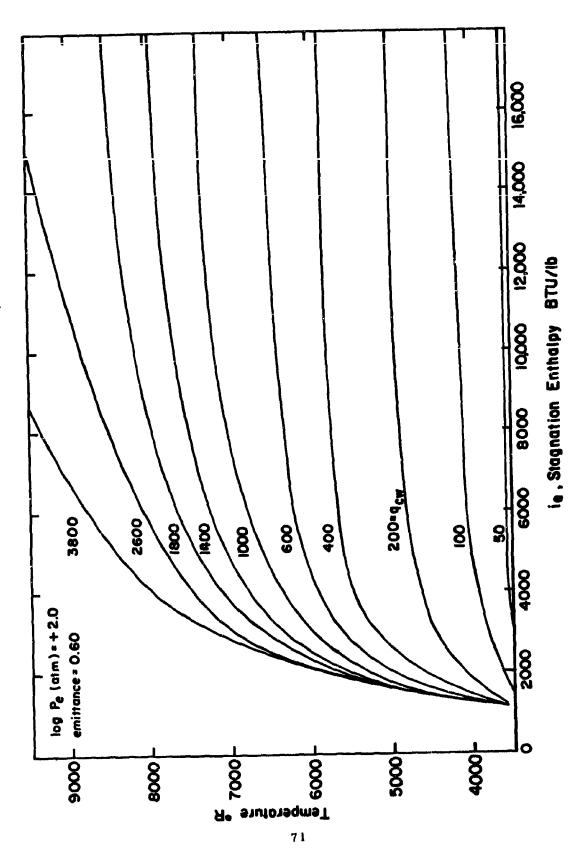
TABLE 28

Test	$egin{aligned} \mathbf{T}_f & \mathbf{T} \text{ (dmils)} \\ & & \text{observed} \end{aligned}$	[€] S 10 ³ k _S 10 ³ k _F R/L/I	$\mathbf{T_f}$ (CALC) $\mathbf{T_f}$ (OBS)	$T(d)$ (CALC) T_f (CALC)	T(d) (OBS)
	$(^{\circ}R)$ $(^{\circ}R)$	(BTU/ft sec ^Q R) (mils)			*
Hf-Ta-Mo (I-2	do (I-23)				
-54M	4650 3560(106)	0.60 7.5 0.5 252/764/42	42 0 98	0 72	ì
7133	q = 455 BTU/ft2sec,	ie = 3800 BTU/lb, Pe		2	0.75
-25M	4890 5260(408)		36 0.94	0.67	0.68
-43R	4530 3560(120)		m 26 0 98	72 0	1
,	q = 403 BTU/ft ² sec,	ie = 7690 BTU/lb, P.		2	67.0
-1MC	5220 3990(97)	0.60 7.5 0.5	46 0.86	0.76	0.76
-2MC	$q = 4.25 \text{ B.I.U/III}^{\circ}$ 8ec, 5310 3640(93)	$_{1e} = 3220 \text{ BTU/Ib, Pe}_{0}$) ,
	$q = 505 BTU/ft^2 sec.$		0.60	0.68	0.65
-4MC	5395 3840(99)	0.60 7.5 0.5	00 00 66	0 64	
521ATU	q = 480 BTU/ft ² sec,	ie = 3560 BTU/lb, P.		# 0.0	0.7
UMCC-	4460 3690(100)	0.60 7.5 0.5	57 1.03	0.69	0.83
-38MH	4 - ±22 DIO/IE-sec, 4680 3330/398/	$L_{\rm e} = 3560 \; \rm{BTU/lb}, P_{\rm e}$		•	
	35	1.50 (.5 0.5)	48 0.95	0.65	0.71
-39RH	(001)089	0.60 7.5 0.5 $252/670/22$	E 1 02	.00	(
40011	BTU/ft2sec,	$i_e = 6740 \text{ BTU/lb}, P_e = 0.137 \text{ a}$		10.0	0.88
HVOF-	300(410) FII/62	0.60 7.5 0.5 253/710/	22 1.06	0.72	08 0
-49RH	4270 346012061	$t_{e} = 6950 \text{ BTU/lb, Pe}$))
	3TU/ft2sec,	i. = 7480 BTII/lb D = 0 111 2+2-	1.10	0.80	0.81
-51RH	3240(400)	0.60 7.5 0.5	THE 1 0.6	7	: : :
500 tre	q = 398 BTU/ft2sec,	ie = 6900 BTU/lb, Pe		0. <i>(</i> 4	0.75
CUNOC.	5570(104) 5. Tii (6.2)	0.60 7.5 0.5 500/905/25	1.04	0.83	72 0
-52RHS	4 = ±02 D10/IIEsec, 4500 2980/3981	$^{1}e = 5750 \text{ BTU/lb}, Pe = 0.115 \text{ atm}$			- •
	q = 398 BTU/ft2'sec,	BTU/ft²sec, i. = 5890 BTU/lb p - 0 118 24m	26 1.01	0.74	99.0
		3 OTT *	9		

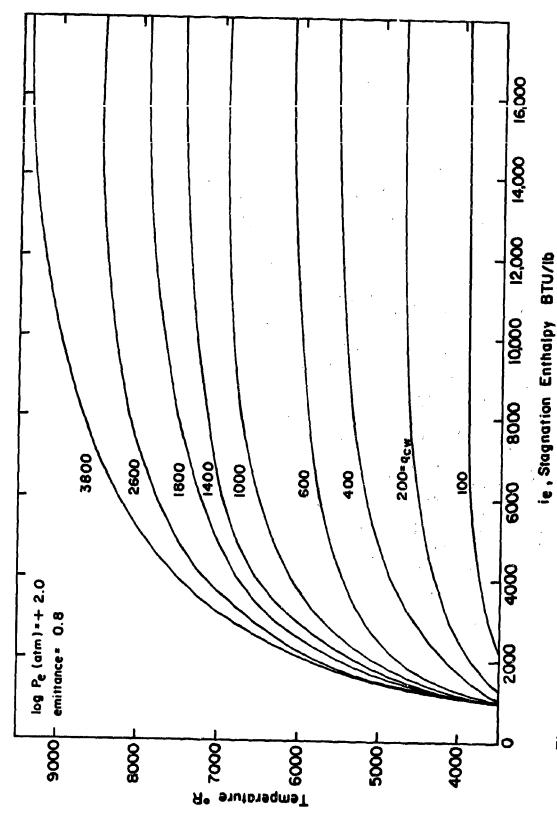


Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 1.

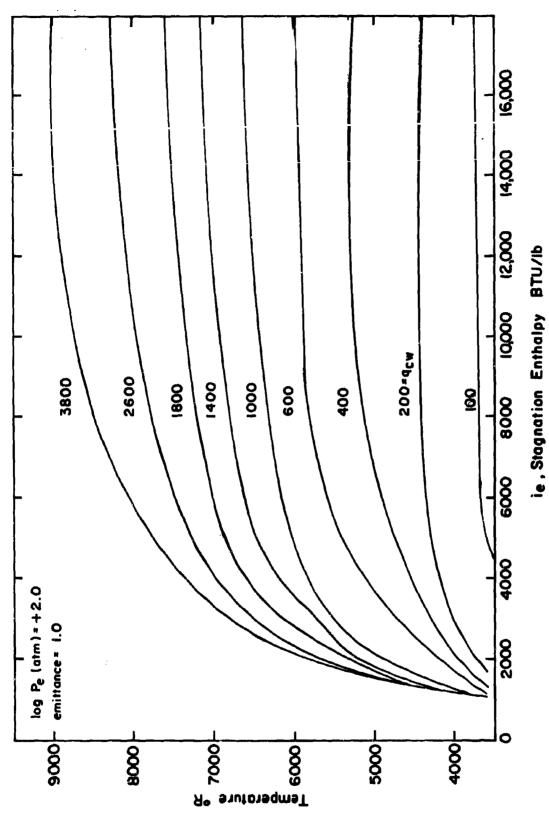




Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 3.



Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 4.



Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 5.

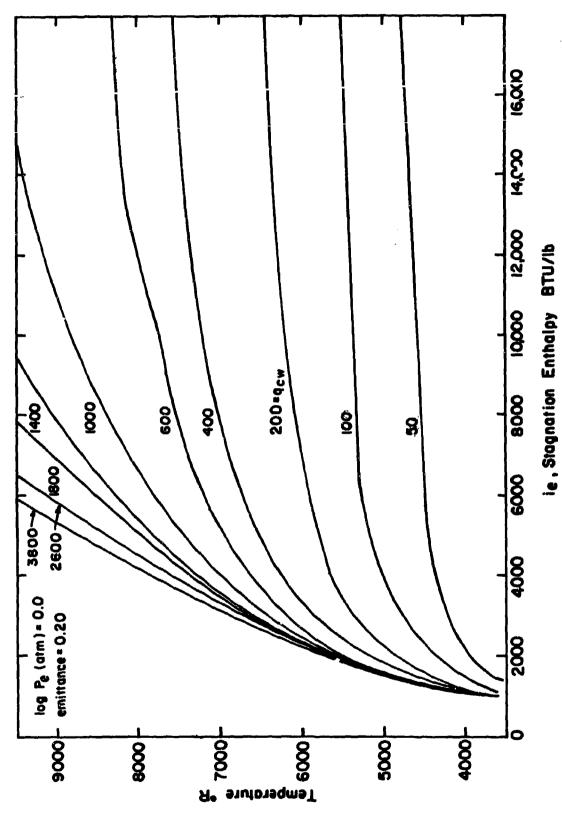
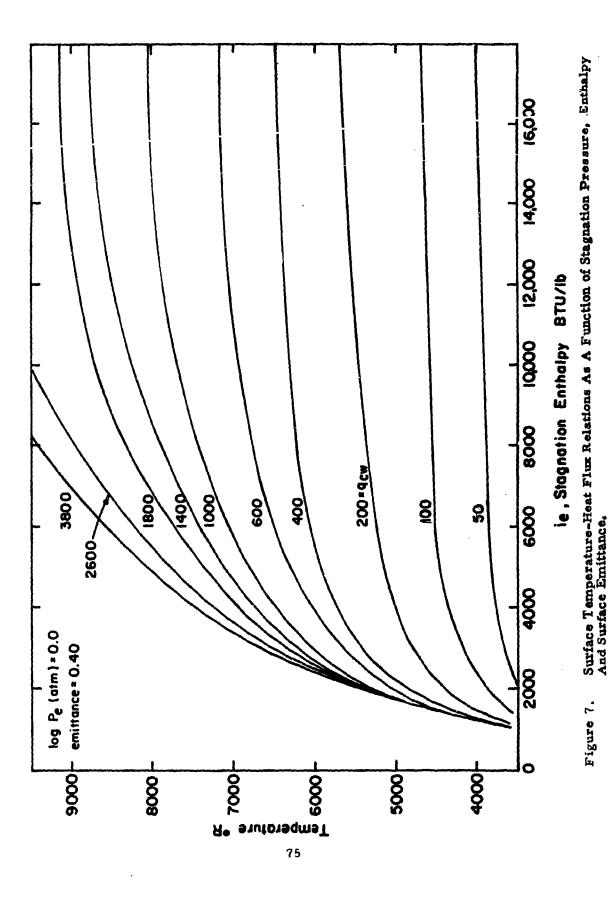
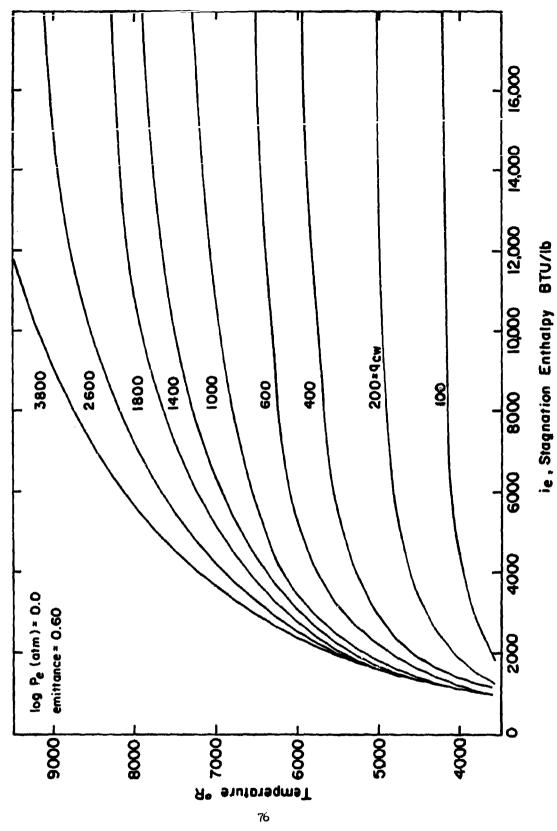
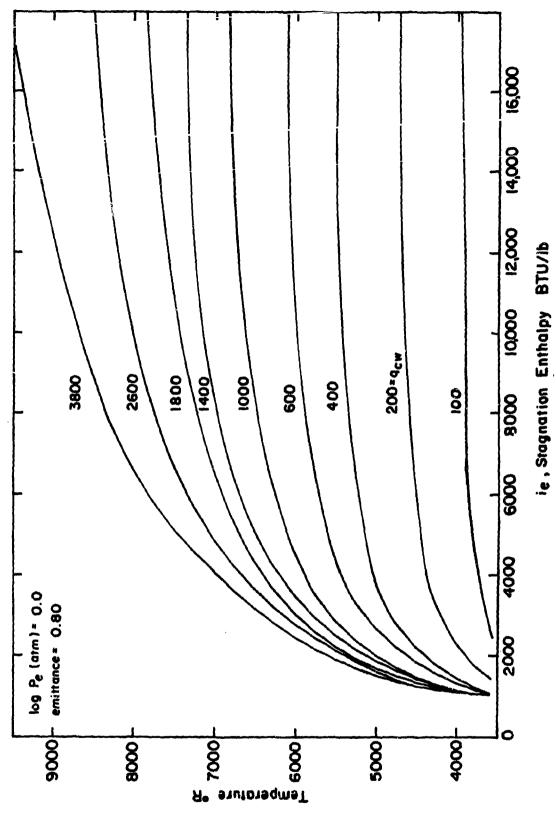


Figure 6. . Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy and Surface Emittance.





Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 8.



Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 9.

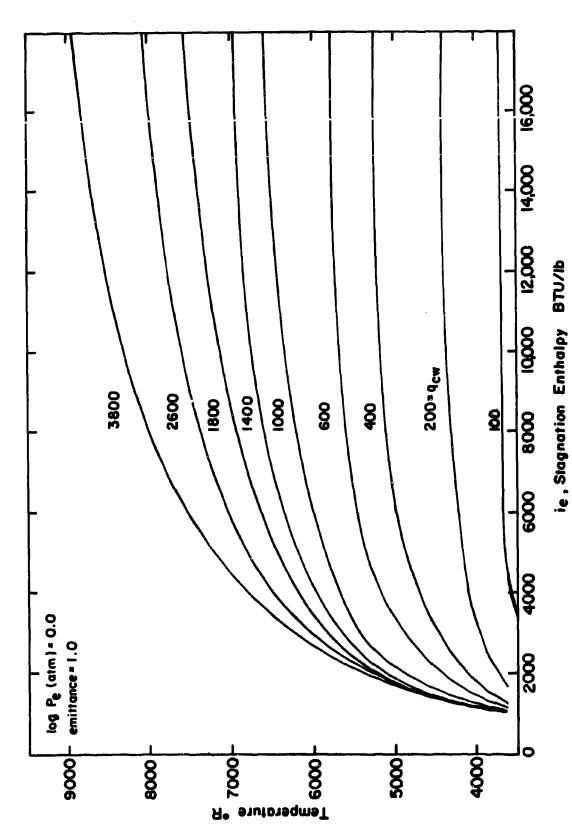
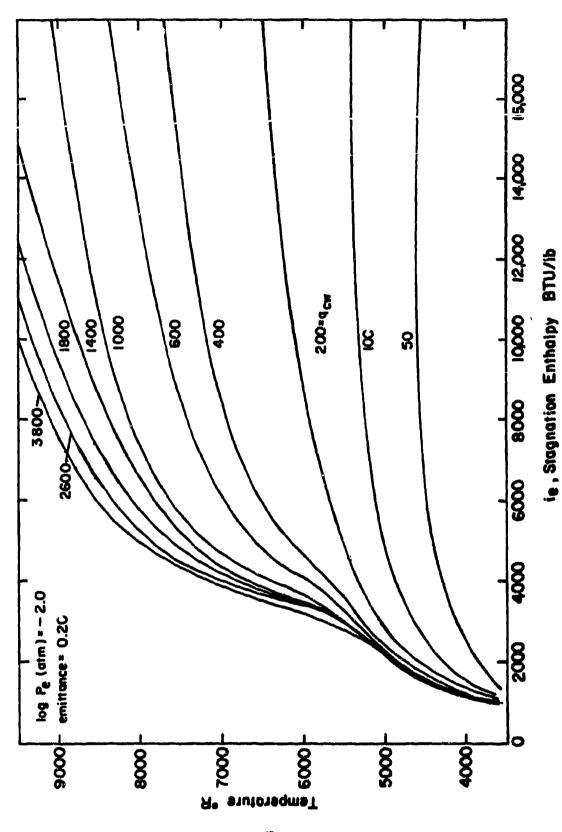


Figure 10. Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Entitalpy And Surface Emittance.



Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance.

Figure 11.

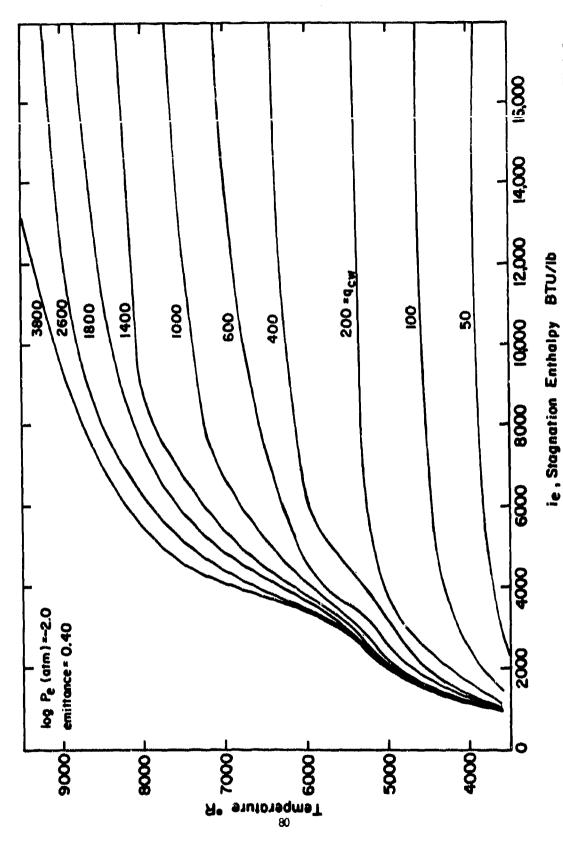


Figure 12.. Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance.

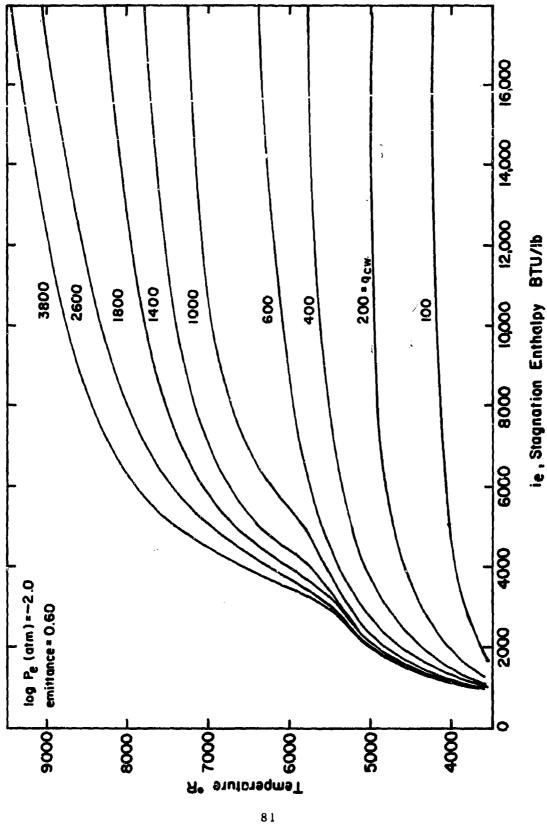
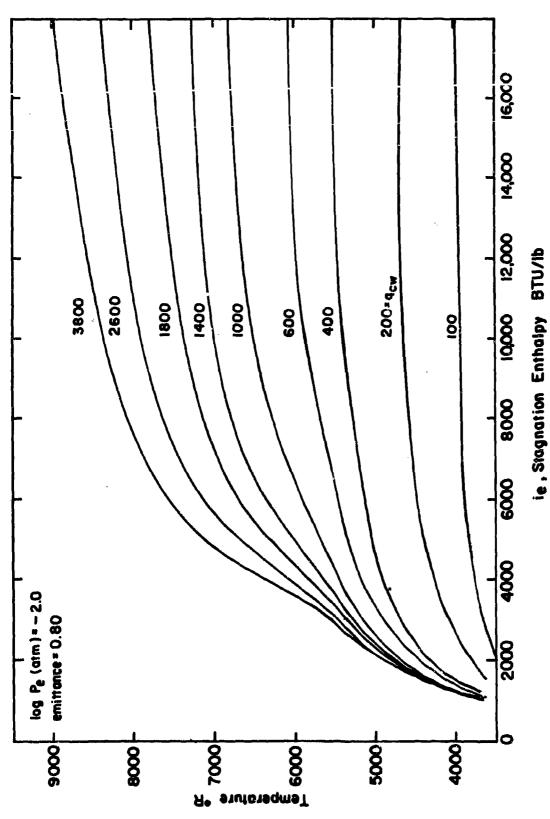
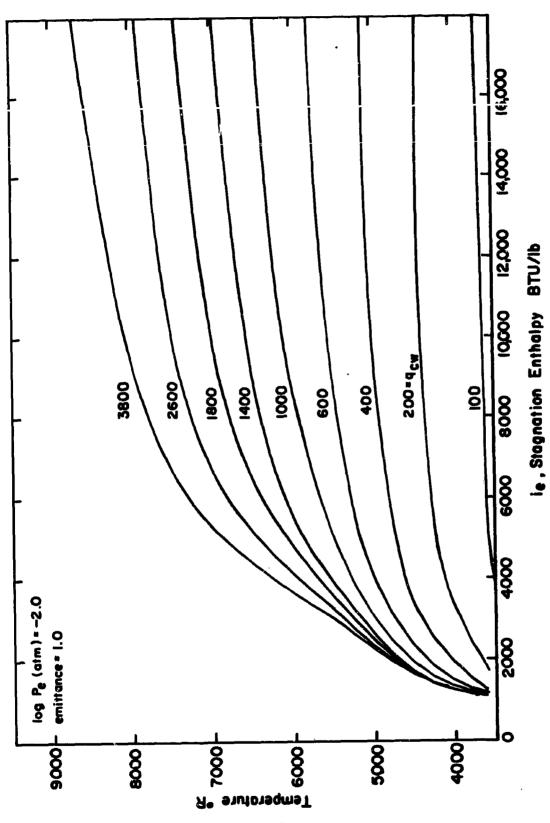


Figure 13. Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance.



Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 14.



1

Surface Temperature-Heat Flux Relations As A Function of Stagnation Pressure, Enthalpy And Surface Emittance. Figure 15.

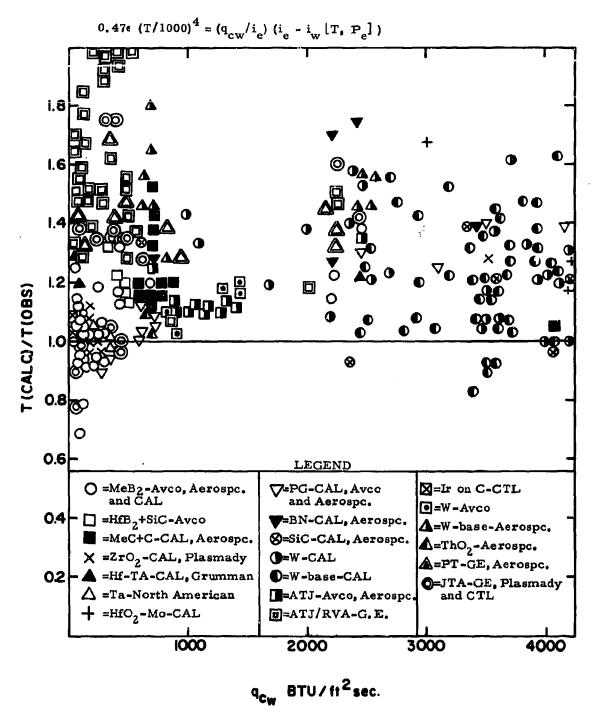
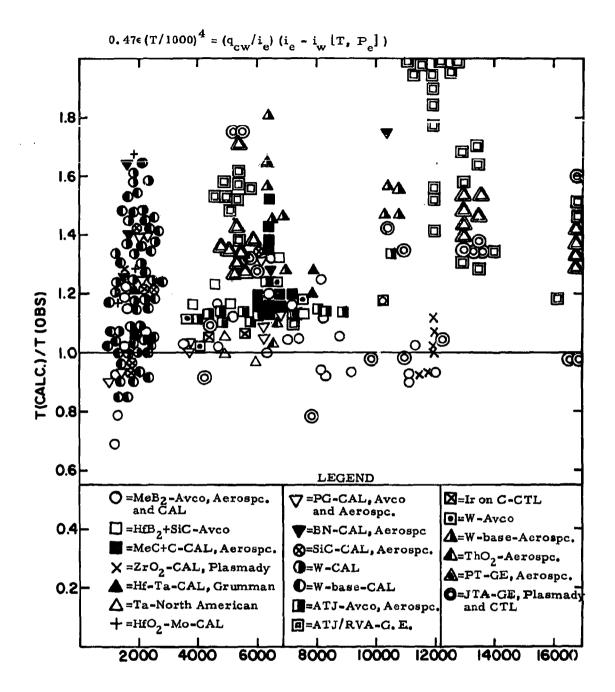


Figure 16. Ratio of Calculated to Observed Surface Temperatures Based on the Cold Wall Heat Transfer Coefficient and Radiation Equilibrium Vs. Cold Wall Heat Flux.



ie , BTU/Ib

Figure 17. Ratio of Calculated to Observed Surface Temperatures Based on the Cold Wall Heat Transfer Coefficient and Radiation Equilibrium Vs. Stagnation Enthalpy.

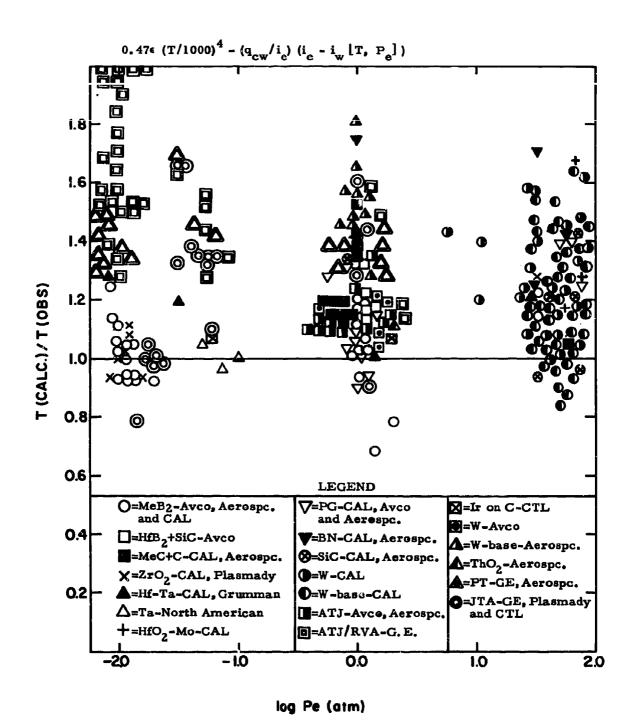


Figure 18. Ratio of Calculated to Observed Surface Temperatures Based on the Cold Wall Heat Transfer Ccefficient and Radiation Equilibrium Vs. Stagnation Pressure.

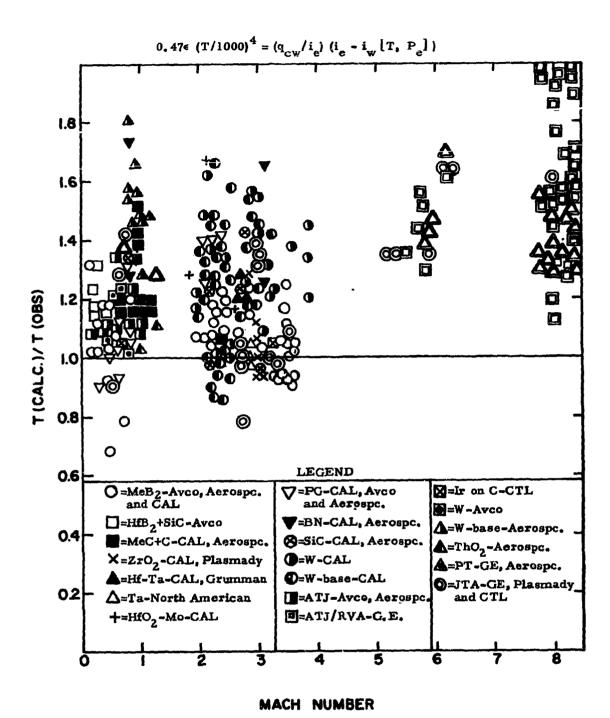


Figure 19. Ratio of Calculated to Observed Surface Temperatures Based on the Cold Wall Heat Transfer Ceefficient and Radiation Equilibrium Vs. Mach Number.

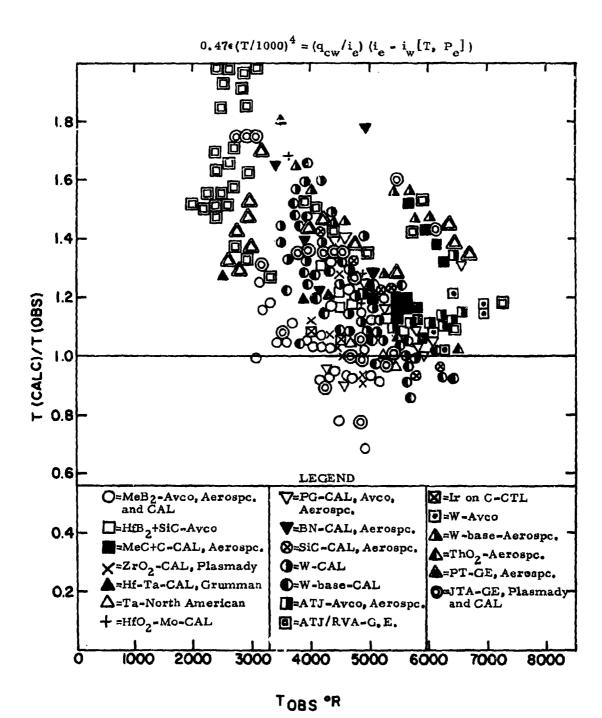


Figure 20. Ratio of Calculated to Observed Surface Temperatures Based on the Cold Wall Heat Transfer Coefficient and Radiation Equilibrium Vs. Observed Temperature.

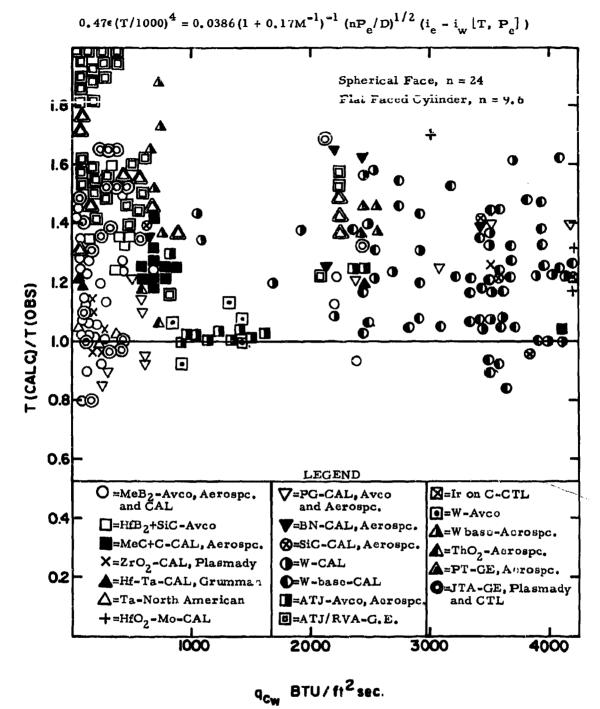


Figure 21. Ratio of Calculated to Observed Surface Temperatures Based on the Fay-Riddell Heat Transfer Coefficient and Radiation Equilibrium Vs. Cold Wall Heat Flux.

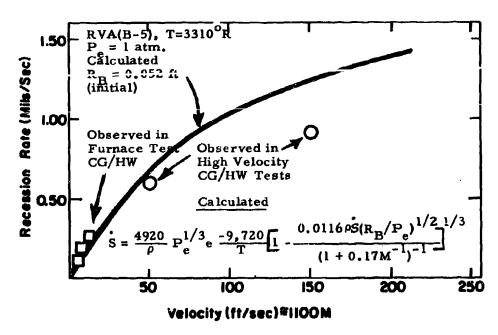


Figure 22. Comparison of Observed and Calculated Recession Rates for RVA(B-5) Graphite Cylinders at 3310 R

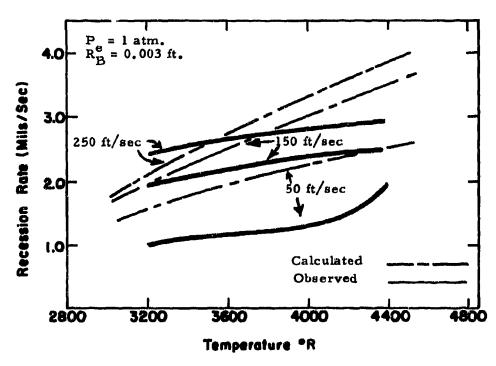
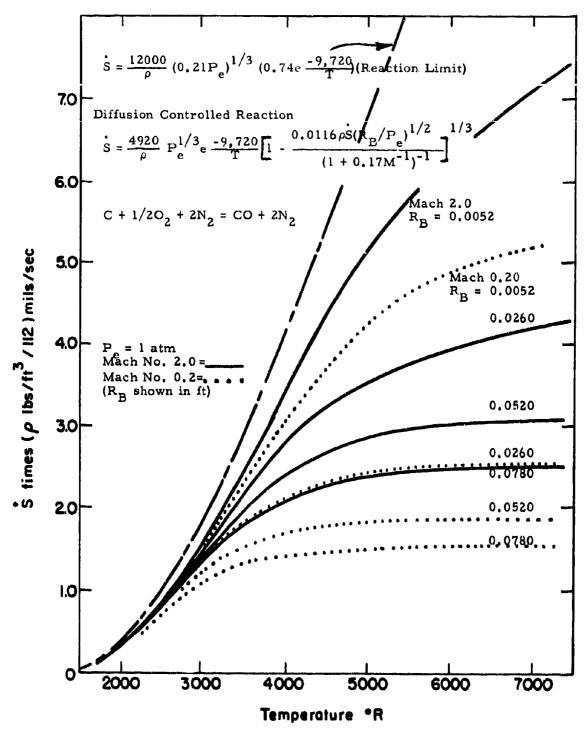


Figure 23. Comparison of Observed and Calculated Recession Rates for Graphite Cones, as a Function of Temperature and Air Velocity



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Figure 24. Calculated Effects of Mach Number and Body Radius on the Oxidation of Graphite at P_e = 1 atm.

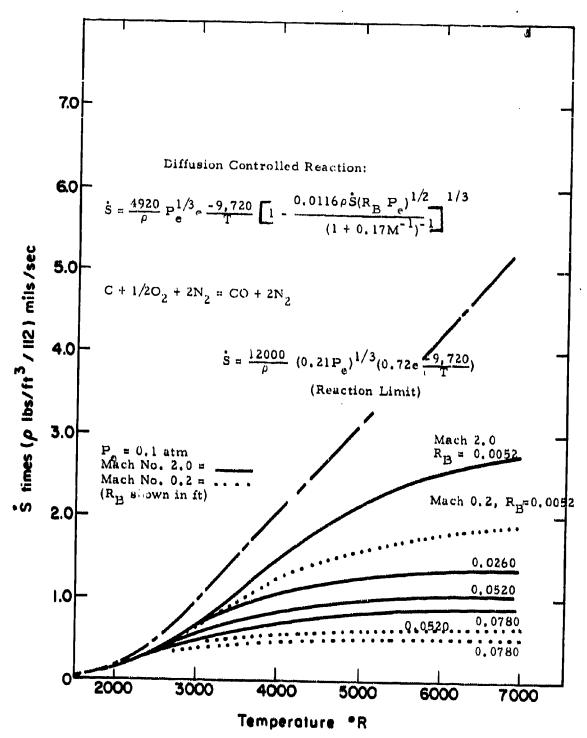


Figure 25. Calculated Effects of Mach Number and Body Radius on the Oxidation of Graphite at P_{e} = 0.1 atm.

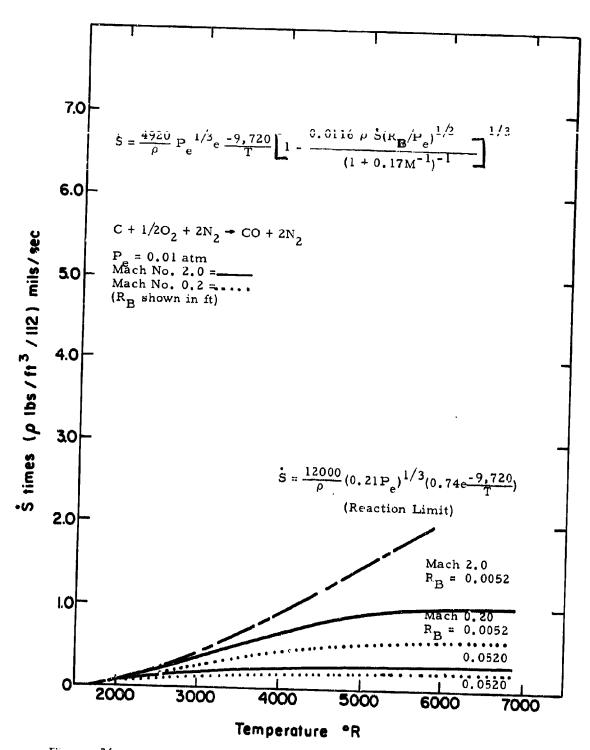


Figure 26. Calculated Effects of Mach Number and Body Radius on the Oxidation of Graphite at $P_e = 0.001$ atm.

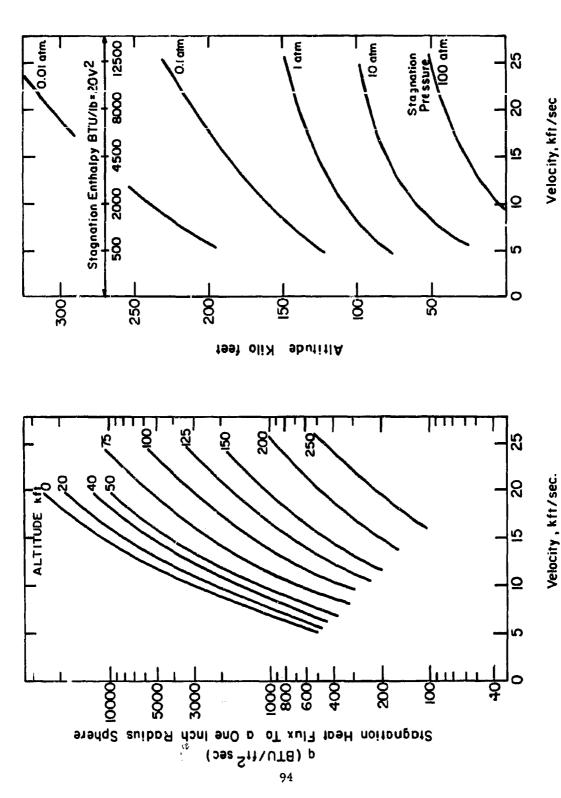


Figure 27. (Altitude/Velocity) Vs. Stagnation (Enthalpy/Flux) Relations for a One Inch Sphere (39-41).

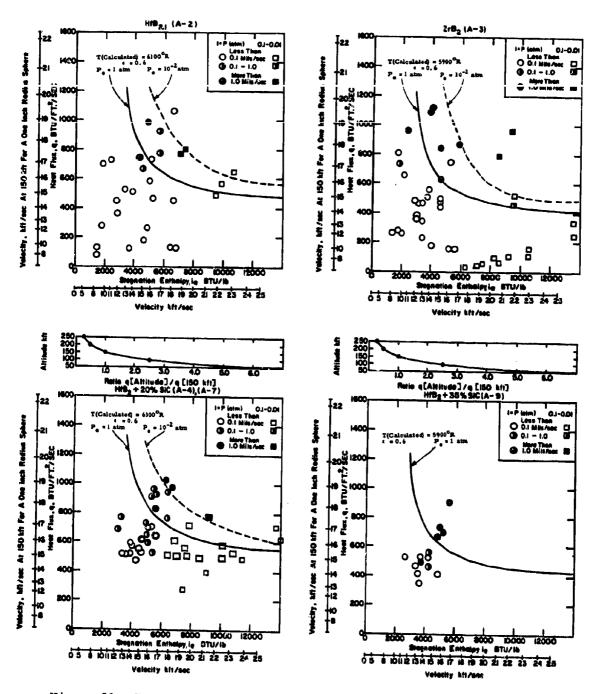


Figure 28. Recession Rates of HfB₂, 1 (A-2), ZrB₂(A-3), HfB₂+ 20%SiC(A-4)(A-7) and HfB₂ + 35%SiC(A-9)², 1 as a Function of Heat Flux and Stagnation Enthalpy.

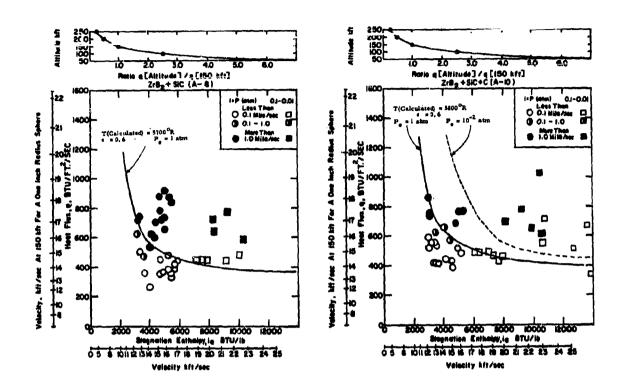


Figure 29. Recession Rates of ZrB₂+20%5iC(A-8) and ZrB₂+14%SiC+30%C (A-10) as a Function of Heat Flux and Stagnation Enthalpy.

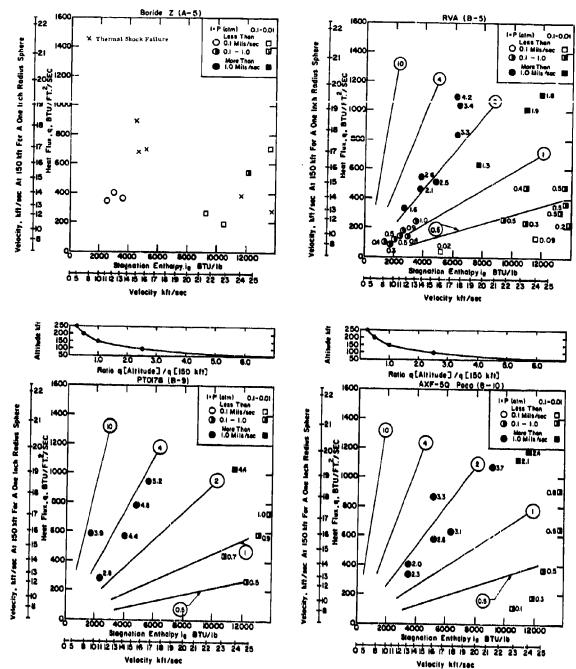


Figure 30. Recession Rates of Boride Z(A-5), RVA(B-5), PT0178(B-9) and POCO(B-10) as a Function of Heat Flux and Stagnation Enthalpy. Lines Indicate Theoretical Recession Rates Calculated from the Equation S = q/1.17V². Numbers on Points Show Measured Recession Rates.

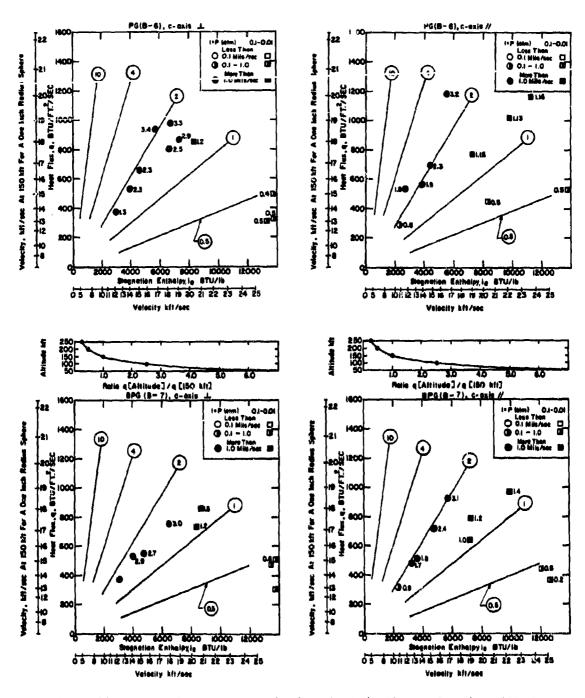


Figure 31. Recession Rates of PG(B-6) and BPG(B-7) as a Function of Heat Flux and Stagnation Enthalpy. Lines Indicate Theoretical Recession Rates Calculated From the Equation $S = q/1.17V^2$. Numbers on Points Show Measured Recession Rates.

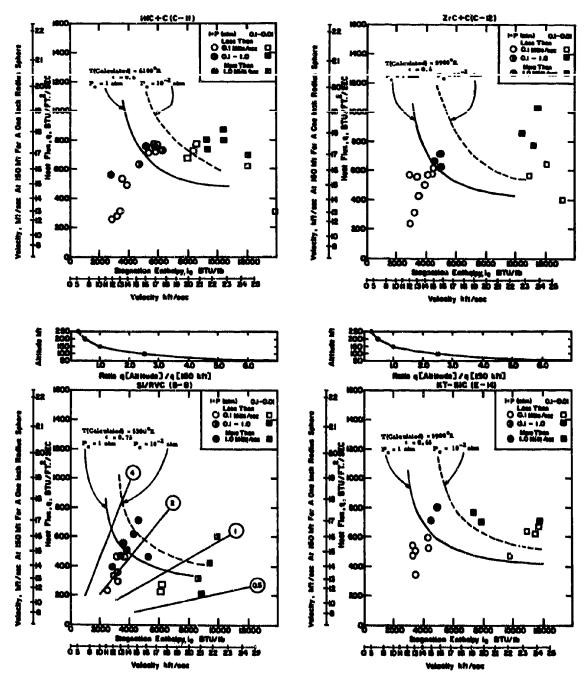


Figure 32. Recession Rates of HfC+C(C-11), ZrC+C(C-12), Si/RVC(B-8) and KT-SiC(E-14) as a Function of Heat Flux and Stagnation Enthalpy, Lines on (B-8) Plot Indicate Theoretical Graphite Recession Rates Calculated From the Equation S = q/1.17V².

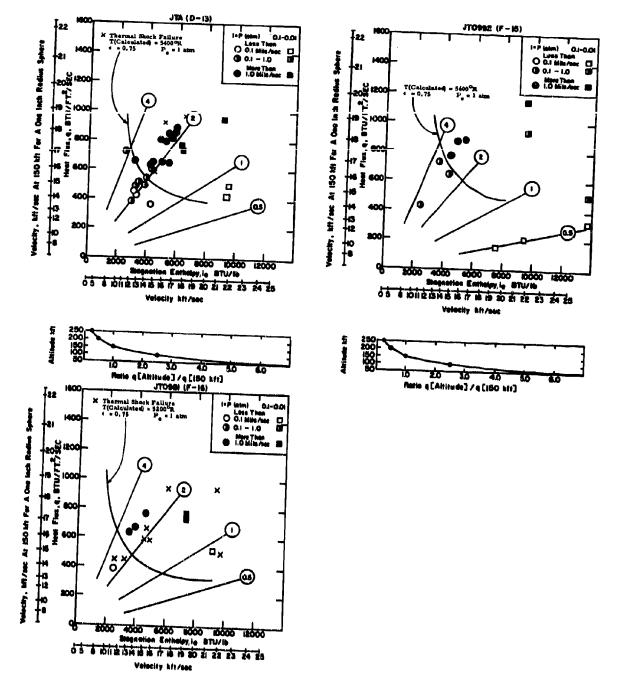


Figure 33.. Recession Rates of JTA(D-13), JT0992(F-15), and JT0981(F-16) as a Function of Heat Flux and Stagnation Enthalpy. Lines Indicate Theoretical Graphite Recession Rates Calculated From the Equation $k = q/1.17V^2$.

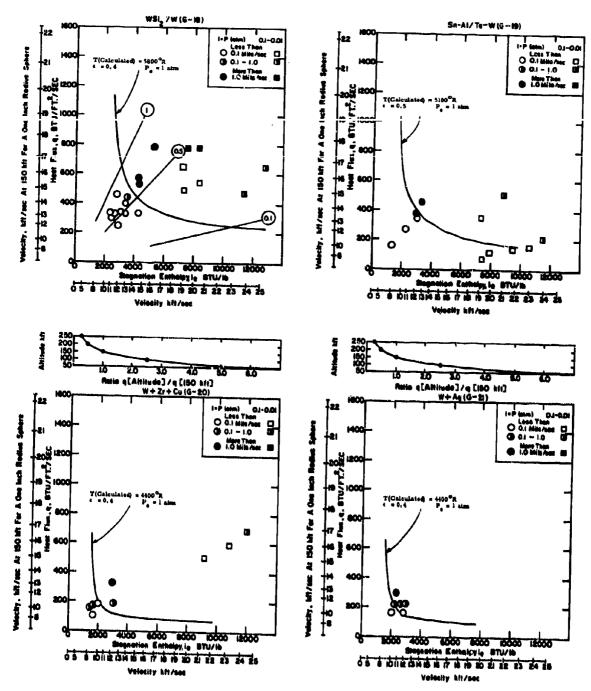


Figure 34. Recession Rates of WSi₂ 'W(G-18), Sn-Al /Ta-W(G-19), W+Zr+Gu (G-20) and W + Ag(G-21) as a Function of Heat Flux and Stagnation Enthalpy. Lines on (G-18) Plot Indicate Theoretical Tungsten Recession Rates.

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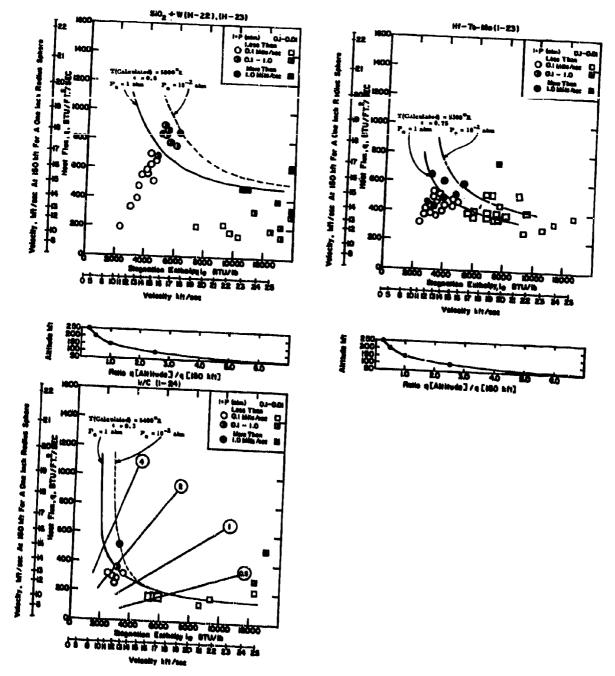


Figure 35. Recession Rates of SiO₂ + W(H-22)(H-23), Hf-Ta-Mo(I-23) and Ir/C(I-24) as a Function of Heat Flux and Stagnation Enthalpy. Lines on (I-24) Plot Indicate Theoretical Graphite Recession Rates.

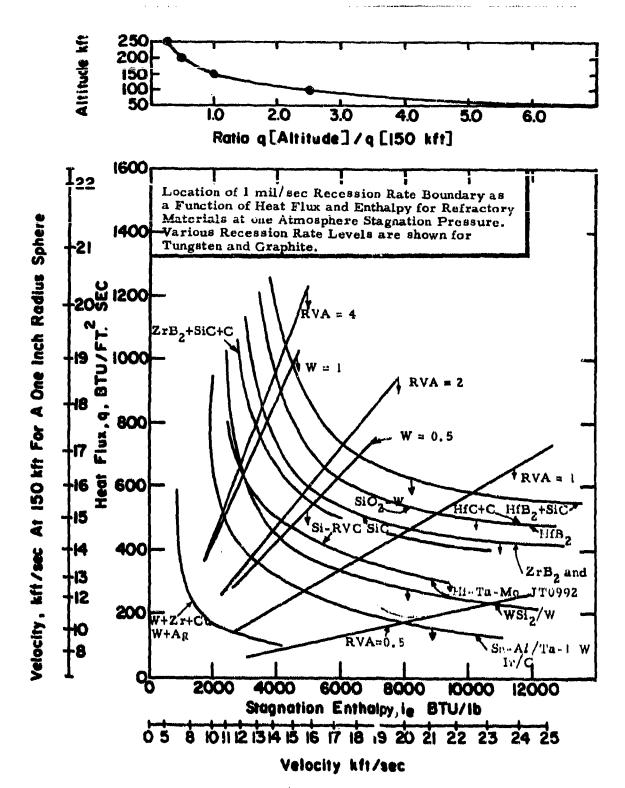


Figure 36. Location of 1 mil/sec Recession Rate Boundary as a Function of Heat Flux and Enthalpy for Refractory Materials at P = 1 atm. Various Recession Levels are Shown for RVA Graphite and Tungston.

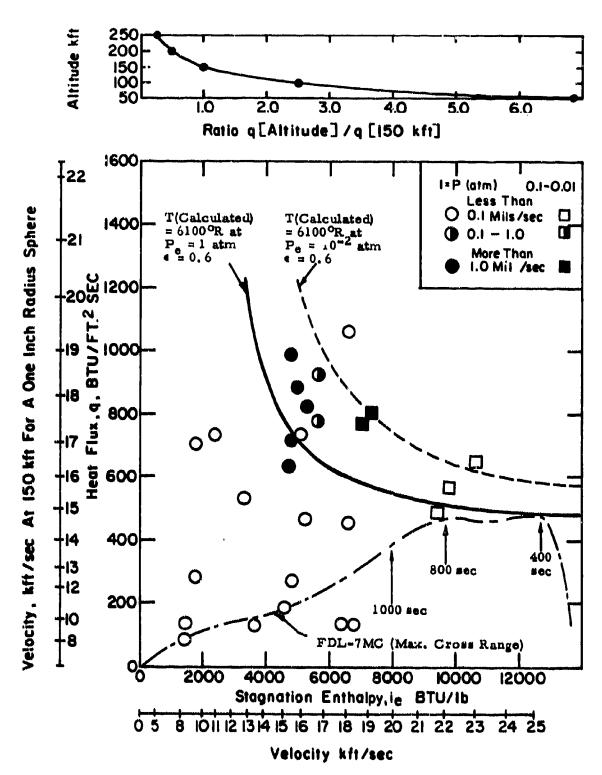


Figure 37. Recession Rate of Hafnium Diboride as a Function of Heat Flux and Stagnation Enthalpy. Compared with FDL-7MC-MCR Trajectory.

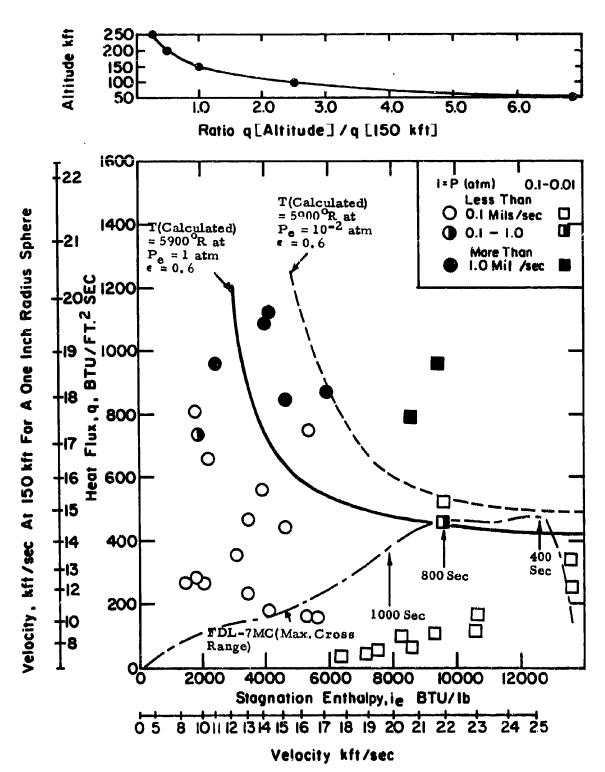


Figure 38. Recession Rate of Zirconium Diboride as a Function of Heat Flux and Stagnation Enthalpy. Compared with FDL-7MC-MCR Trajectories.

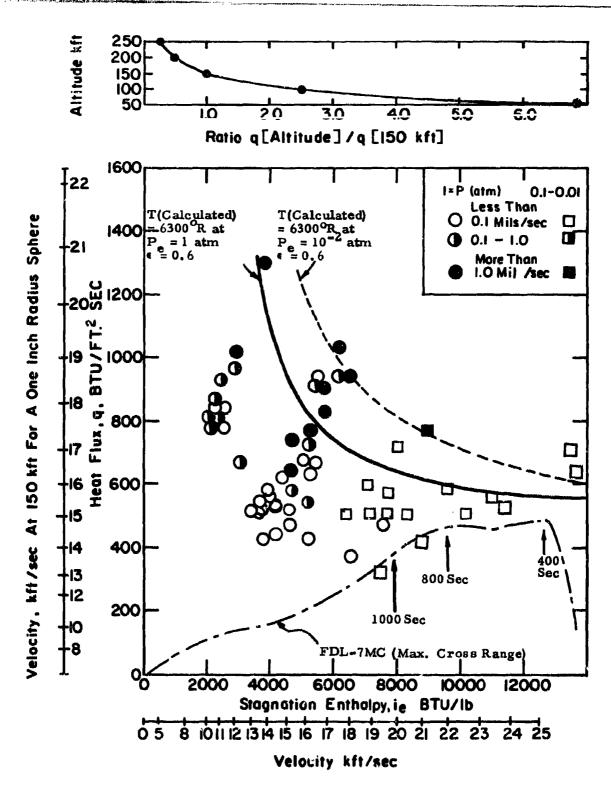


Figure 39. Recession Rate of HfB₂ + SiC Composites as a Function of Heat Flux and Stagnation Enthalpy. Compared with FDL-7MC-MCR Trajectory.

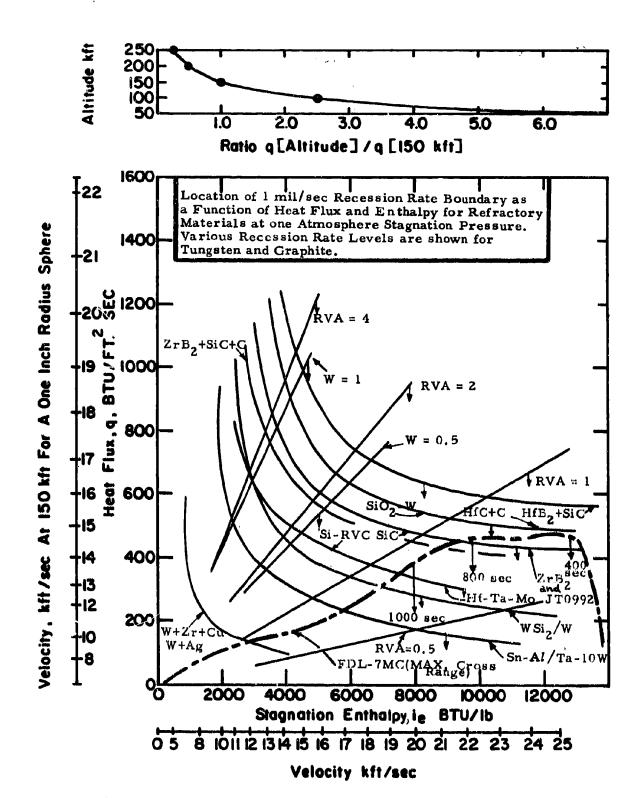


Figure 40. Superposition of the FDL-7MC Trajectory on the Flux-Enthalpy Map showing Recession Rates for Refractory Materials.

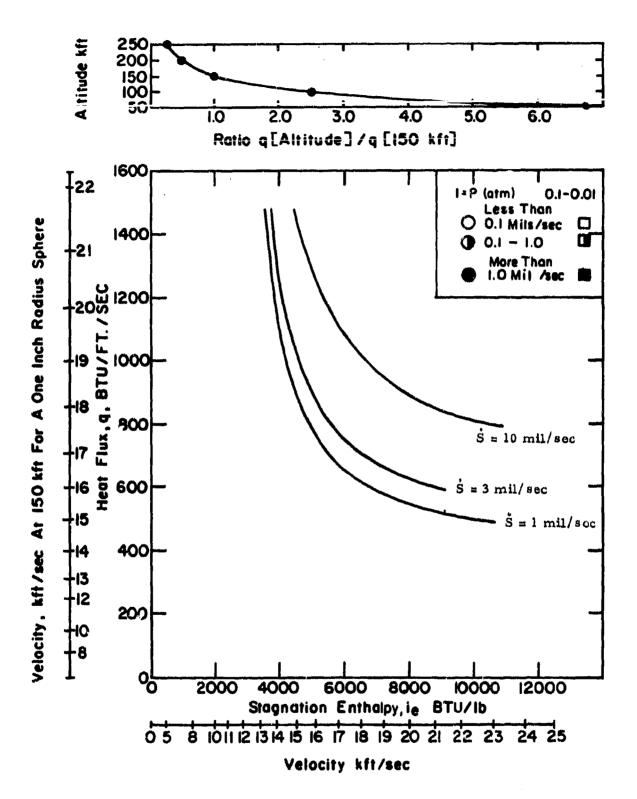


Figure 41. Computed Boundaries for Melting Recession of HfB2.1(A-2),

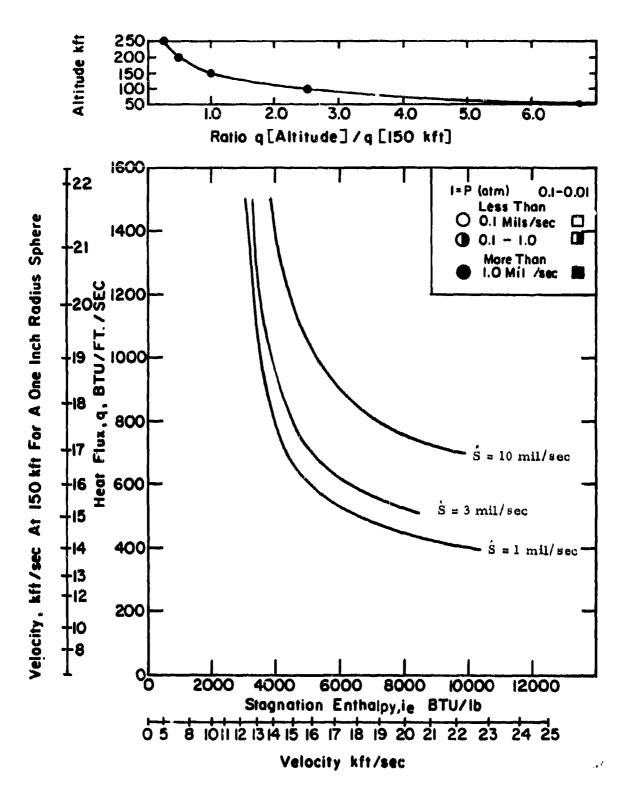


Figure 42. Computed Boundaries for Melting Recession of ZrB2(A-3).

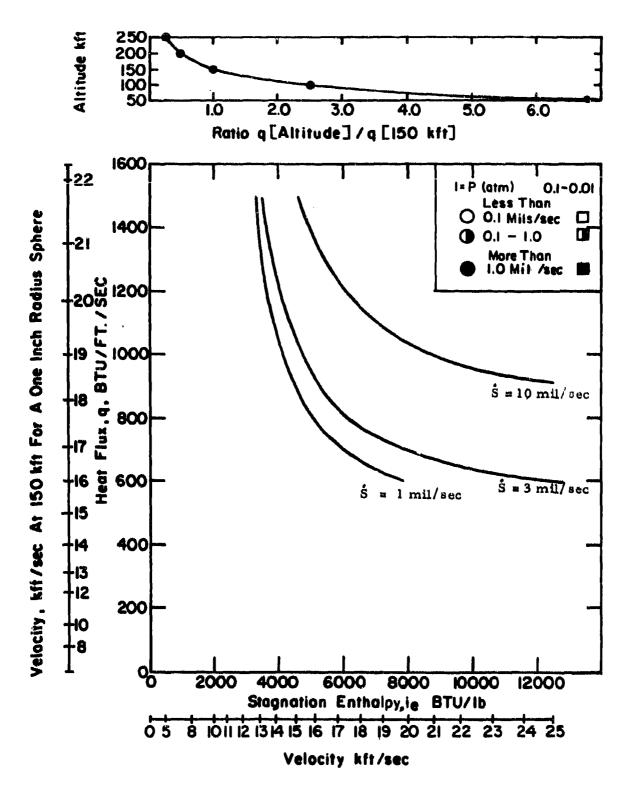


Figure 43; Computed Boundaries for Melting Recession of HfC+C (C-11).

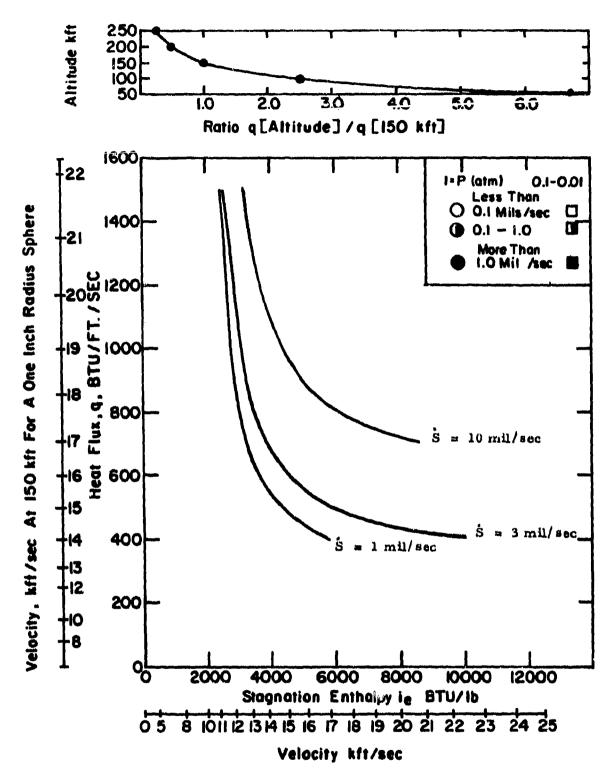


Figure 44. Computed Boundaries for Melting Recession of ZrC + C (C-12).

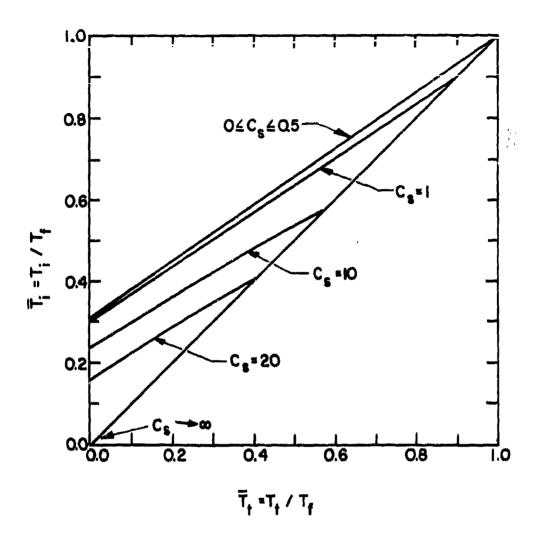


Figure 45. Variation of T_i with T_t for a Number of C_S Values According to Equation 64 when I/L = 0 and k_F/k_S = 0.10.

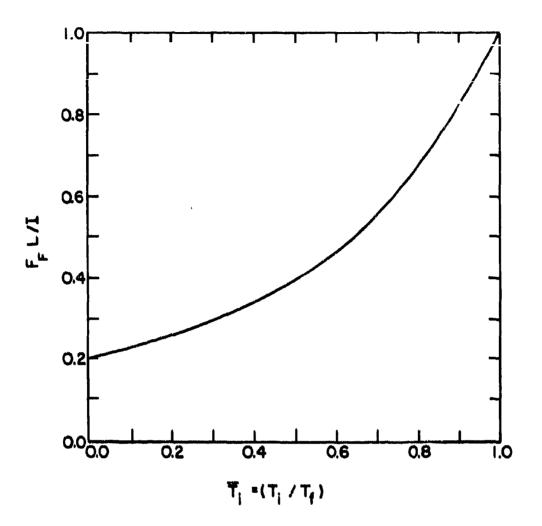


Figure 46. The Effect of Interface Temperature, T_i, on the Level of the Radiation Contribution, F_F, According to Equation 66.

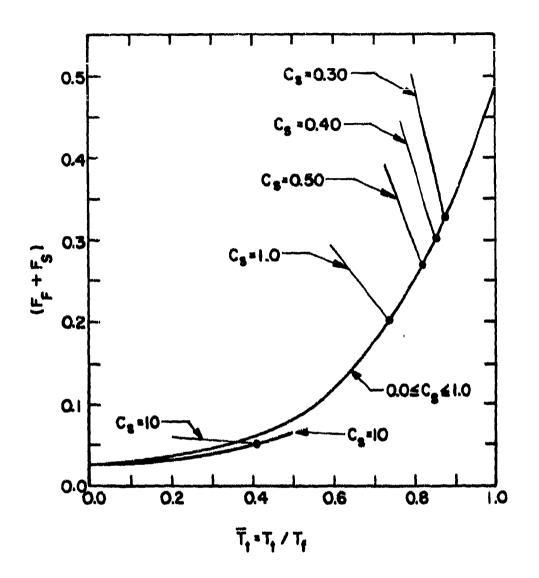


Figure 47. The Dependence of the Total Radiation Parameter $(F_F + F_S)$ on T_t for various values of C_S when I/L = 0.10 and $k_F/k_S = 0.10$ derived from Equations 66 and 69. Intersections Define Solutions on the Basis of Equation 72.

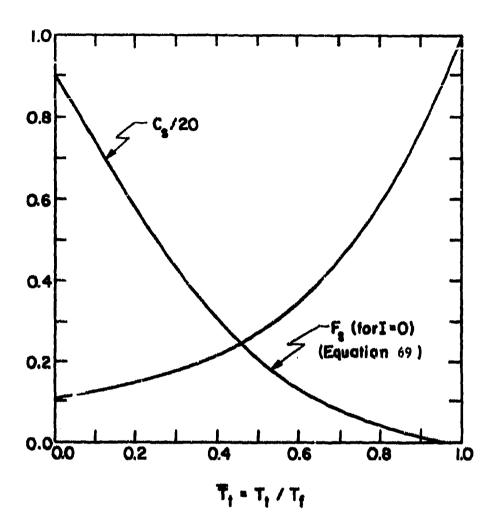


Figure 48. The Effect of C_S on T_t and the Corresponding Radiation Parameter F_S for the case where I=0 (i.e. $T_i=T_f$) Based on Equation 74.

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The oxidation of refractory borides, graphites and JT composites, hyper- eutectic carbide-graphite composites, refractory metals, coated refractory metals, metal-oxide composites, and iridium coated graphites in air have been studied under high velocity atmospheric flight conditions. Elucidation of the relationship between hot gas/cold wall (HG/CW) and cold gas/hot wall (CG/HW) surface effects in terms of heat and mass transfer rates at high temperatures was a principal goal. A method of presentation which compares recession rate as a function of heat flux and enthalpy for the candidate materials was developed. This description provide a means for comparing performance for various trajectories by applying a flux/en- thalpy-altitude/velocity translation in considering candidate materials. Comparison of the trajectory of the FDL-7MC lifting reentry vehicle (Lift/Drag ratio between 2.5 and 3.0 and a 3" nose radius) eliminates all of the candidate materials except the boride composites. These composites have survived multicycle exposures totaling 20,000 seconds under conditions simulating the most severe portions of the FDL-7MC			
trajectory. Measurement of temperature gradients through oxide films formed during arc plasma exposures indicate substantial gradients (1000°R through 100 mils) can exist. This abstract is subject to special export controls and each transmittal			
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